

Modelling large deformation and failure using LS-DYNA

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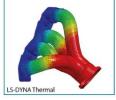
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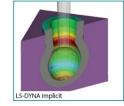


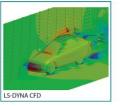
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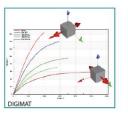














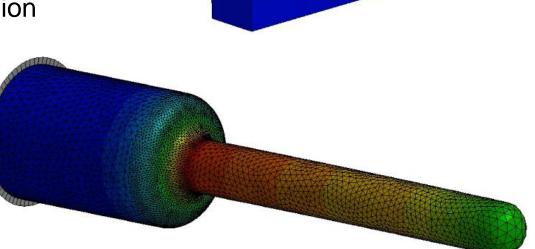


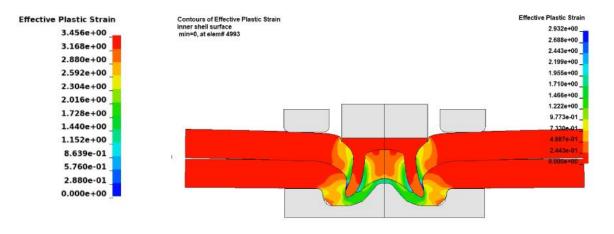
Applications

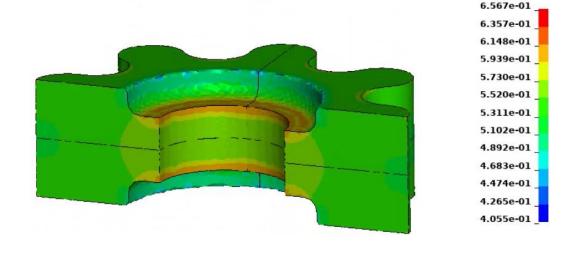


Example applications for simulation of large deformations and failure

- Self Piercing Rivets
- Flow Drill Screws
- Cutting/Milling
- Forging
- Powder Compaction
- Extrusion







History Variable#1

Simulation methodology – Large deformations



Overview on available methods in LS-DYNA

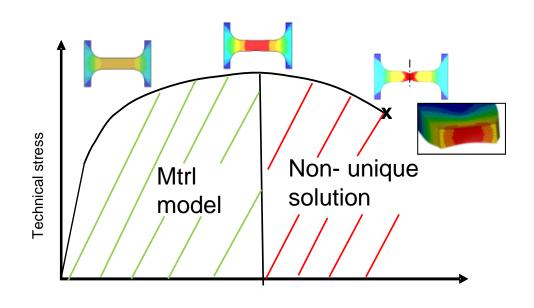
- The applications can show one or several of the following behaviors
 - Large deformations with material movement
 - High contact pressures
 - Material failure
 - Thermo- mechanical process
- There are several methods in LS-DYNA to solve these types of problems
- Time discretization can be either implicit or explicit
- Spatial discretization
 - FEM
 - Remeshing Adaptivity Both 2D and 3D available
 - ALE/S-ALE (Structured) Arbitrary Lagrangian Eulerian method.
 - Meshless methods
 - SPH Smooth Particle Hydrodynamics
 - EFG Element Free Galerkin
 - SPG Smooth Particle Galerkin

Failure modeling

Background



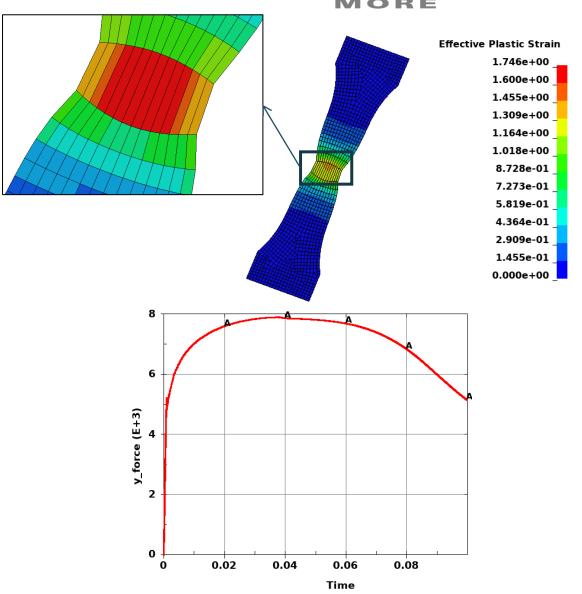
- Failure can be modelled using several approaches in LS-DYNA
- As long as the solution is stable (no material instabilities), the solution is unique and the behavior is determined by the material model
- Beyond the point of instability (e.g. diffuse necking) the solution becomes non- unique.
 - Element type and size
 - Material model
 - Hardening
 - ...



Failure modeling - Constitutive model

DYNA

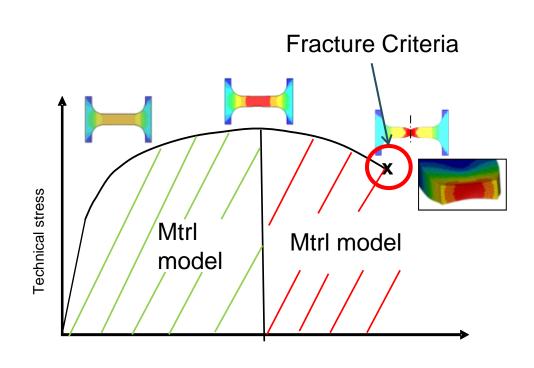
- The constitutive model alone will provide a softening response beyond the instability point.
- For very large deformations e.g. cutting or milling this might be enough since the material data is quite uncertain for high strains, strain rates and temperatures.
- The material "splitting" is then handled by the simulation method (3D adaptivity, ALE, etc)



Failure modeling - Criteria



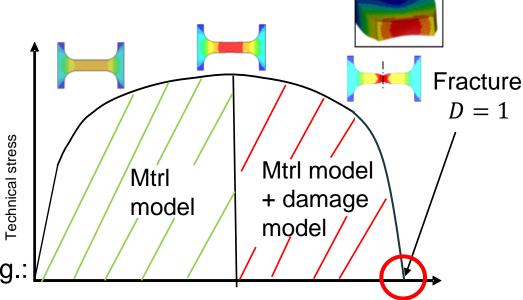
- Fracture Criteria is predicting when a material fails
- It can be based on e.g. effective plastic strain, pressure, maximum principal strain or a failure curve.
- Many material models inside LS-DYNA contains erosion criteria e.g.
 - *MAT_24 : Piecewise Linear Plasticity
 - *MAT_15 : Johnson Cook
 - *MAT_224: Tabulated Johnson Cook
- The user can add erosion criteria to any material model in LS-DYNA using *MAT_ADD_EROSION
- When the failure criteria is reached, the element is deleted from the simulation.



Failure modeling - Damage



- Damage modeling means predicting <u>how</u> a material fails
- The most common way of doing this is to add damage model to the constitutive model
- The damage model is a scalar (D) that evolves with e.g. plastic deformation and couples to the stress
- As the damage is coupled the material softens
- When D=1, the material carries no load and can be deleted
- Many material models in LS-DYNA accounts for damage, e.g.:
 - MAT_104: Damage
 - MAT_187 : Samp-1
- Damage can be added to "any" material model in LS-DYNA using e.g.:
 - *MAT_ADD_DAMAGE_DIEM
 - *MAT_ADD_DAMAGE_GISSMO
 - *MAT_ADD_GENERALIZED_DAMAGE





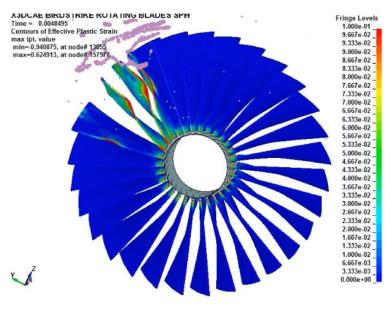
Lagrangian FEM

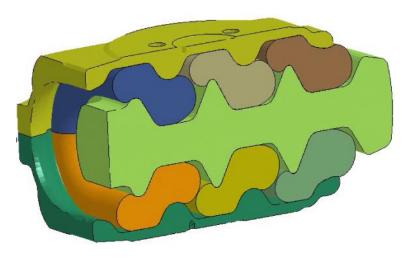
Modeling Large deformations using FEM

DYNA

- Handling large deformations basically means dealing with
 - Large transformations and rotations
 - Large material deformations
- LS-DYNA handles large translations and rotations by using objective stress update
 - The stress measure is not dependent on the frame of reference and large rigid body motions does not introduce strains
- In lagrangian FEM the element follows the material deformation which leads to element distorsion.
 - Inverted elements (negative volume)
 - Timestep drops
 - Loss of accuracy
- Lagrangian FEM can handle a lot of abuse, but at some point it is necessary to switch to other formulations



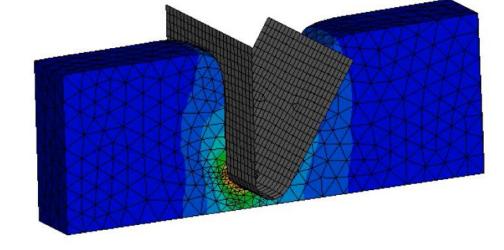




Adaptive remeshing



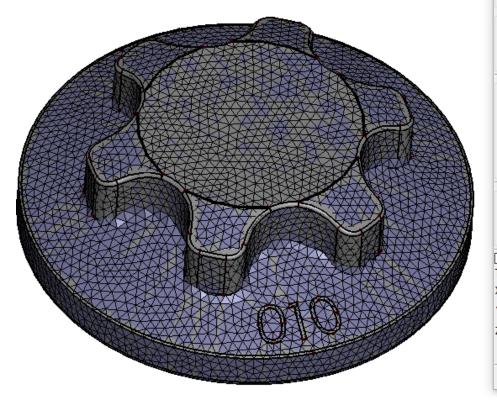
- By using remeshing, it is possible to use lagrangian Finite Elements for simulating arbitrary deformations.
- Supports Thermo- mechanical simulations
- Can be combined with EFG formulation.
- By using "Look Ahead Adaptivity", thus refinement based on contact curvature, it is possible to limit the number of elements.
- No erosion criteria possible. The mesh will conform to the cutting surface
- Drawbacks are:
 - Adaptive part can not separate in 3D
 - Remeshing is time- based Risk for inverted elements
 - Restart and mapping of results leads to loss of data.
 - Constraints are lost at adaptive refinement

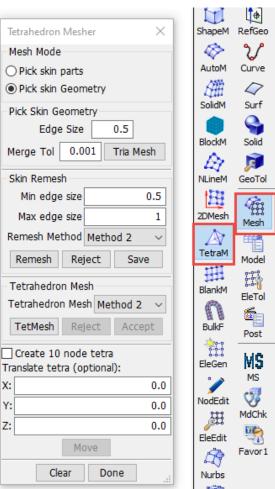




Preprocessing

Terahedron mesh in LS-PREPOST







- Tetrahedron mesh needs a triangular skin.
- The skin can be from a meshed part or geometry
- Remesh: Remeshes the skin based on the triangular skin. Thus, the geometry information is lost.
- The re-meshing is based on minimum and maximum edge size
- When the surface mesh is OK, do the Tetrahedron mesh
- Element quality can be checked in model checking



LS-DYNA remeshing keywords



*CONTROL_ADAPTIVE									
\$# adpfreq	adptol	adptyp	maxlvl	tbirth	tdeath	lcadp	ioflag		
0.001	1.000E20	7	3	0.01.	.00000E20	0	0		
\$# adpsize	adpass	ireflg	adpene	adpth	memory	orient	maxel		
0.0	0	0	1.0	0.0	0	0	0		

- adpfreq: Adaptive remesh frequency. Can be controlled by loadcurve lcapd
- adptyp: Set to 7 for 2D/3D remeshing
- adpene: Remesh based on contact search (look ahead adaptivity) and tool curvature.
- adptol: For 2D adaptivity this is the target element size

*CONT	ROL_REMES	HING						
\$#	rmin	rmax	vf_loss	mfrac	dt_min	icurv	iadp10	sefang
	1.00	1.50	1.0	0.0	0.0	4	0	0.0

- rmin: 3D adaptivity minimum edge length
- rmax: 3D adaptivity maximum edge length
- icurve: Number of elements along look ahead radius

*PART \$# blank								title
		1.5	1.3					
\$#	pid	secid	mid	eosid	hgid	grav	adpopt	tmid
	1	1	1	0	0	0	2	0

 adpopt: Set to 2 for remeshing adaptivity (both 2D and 3D)

*CONSTRAINED_LOCAL

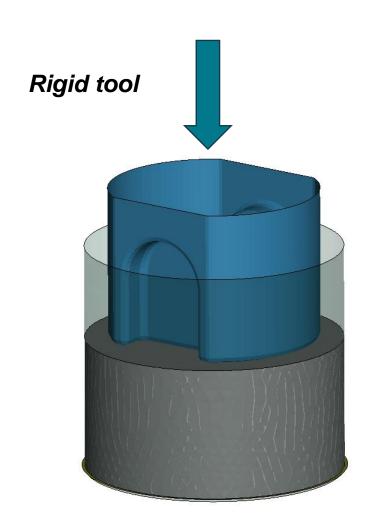
\$# tc rc dir x y z cid tol

1 1 1 0.0 30.0 0.0 1 0.0

- *BOUNDARY SPC constraint will not work with adaptivity.
- Use *CONSTRAINED LOCAL instead
- Nodes within a tolerance of a local coordinate plane is constrained.

Forging example





200 adaptive remeshes with look ahead adaptivity

*CONTROL_AD	*CONTROL_ADAPTIVE									
\$# adpfreq	adptol	adptyp	maxlvl	tbirth	tdeath	lcadp	ioflag			
0.1	1.000E20	7	3	0.01	.00000E20	0	0			
\$# adpsize	adpass	ireflg	adpene	adpth	memory	orient	maxel			
0.0	0	0	4.0	0.0	0	0	0			

1 point tetrahedron element

*SEC	*SECTION_SOLID								
\$#	secid	elform	aet						
	1	13	0						

Blank mesh size is less than tool mesh. Rmax/Rmin \approx 3

*CON	*CONTROL_REMESHING									
\$#	rmin	rmax	vf_loss	mfrac	dt_min	icurv	iadp10	sefang		
	1.00	3.00	1.0	0.0	0.0	4	0	0.0		

Forming contact allows for "Look ahead" adaptivity

```
*CONTACT_FORMING_ONE_WAY_SURFACE_TO_SURFACE_ID

$# cid
3

$# ssid msid sstyp mstyp sboxid mboxid spr mpr
5 3 3 3 0 0 0 0 0 0
```

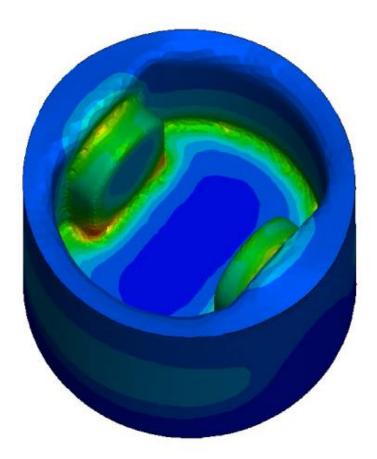
Forging Example

Post processing – LS-PREPOST

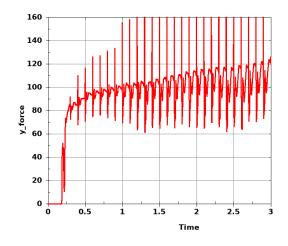


"Look ahead" adaptivity

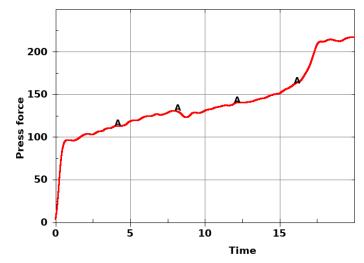




DYNAMORE



Contact force is noisy from adaptivity



Contact force with filtering



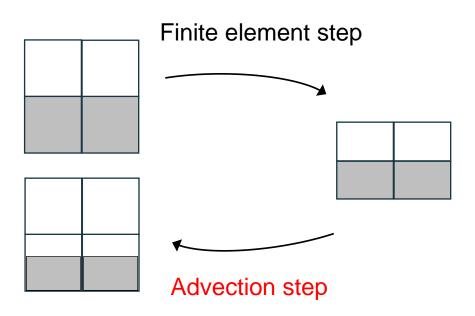
ALE/S-ALE

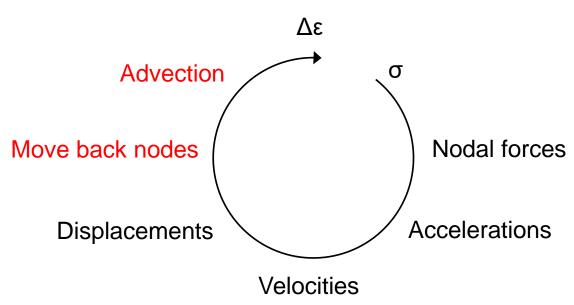
(Structured) Arbitrary Lagrangian Eulerian

Background ALE - Multi-Material Eulerian formulation



- The steps performed each time step are:
 - Finite element step: Using the explicit central difference method, bulk viscosity, and mixture law. This deforms the mesh. This is a Lagrangian time step.
 - Advection step: The solution on the deformed mesh is mapped back to the original un-deformed mesh including interface reconstruction.





Background ALE



- FEM vs ALE in LS-DYNA
 - Efficiency: ALE is less efficient than FEM in the sense to obtain the same accuracy, for models without excessive strains and deformation, more computational effort and smaller elements are generally needed. One must also mesh the space outside the solids/fluids which can further increase the disadvantage of ALE.
 - Extreme deformations: ALE can handle extreme deformations that FEM cannot.
 - Surface & boundary conditions: In a FEM simulation the boundary of materials/parts are accurately defined making it possible to easily prescribe boundary conditions and contact conditions. This is not the case in ALE. Example if two parts touch in the ALE simulation they will generally stick or weld together.

LS-DYNA ALE keywords



nadv: Cycles between advection steps

*CONTROL_ALE									
\$#	dct	nadv	meth	afac					
	1	1	2	-1.0					

```
*CONSTRAINED LAGRANGE IN SOLID
$# lstrsid
               alesid lstrstyp
                                   alestyp
                                                nquad
                                                           ctype
                                                                     direc
                                                                                mcoup
                    1
     start
                  end
                           pfac
                                      fric
                                               frcmin
                                                                                 damp
                                                            norm
                                                                   normtyp
       0.0
               1.E10
                            0.1
                                                  0.5
                                                                                  0.0
                                       0.0
                                                               0
```



afac: -1. Turns smoothing off (Euler

meth: Advection method

method)

ctype: Coupling method. Penalty or constraint based.

nquad: Number of coupling points

- direc: Tension/compression
- fric: Coefficient of friction.
- **pfac:** Penalty factor.

*INITIAL VOID PART pid

pid: Part can contain material but is initially void. Part material is set to "material".

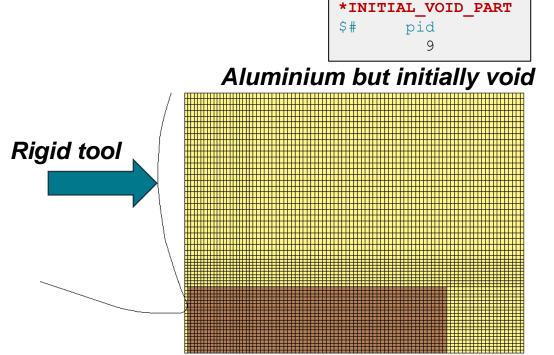
*SECTION SOLID secid elform aet 12 0

- elform:
 - 11 for multimaterial
 - 12 for material and void

Cutting example

Model





Euler Lagrange coupling output



1 point ALE element with material and void



Full Euler - no smoothing

*CON	*CONTROL_ALE									
\$#	dct	nadv	meth	afac						
	1	1	2	-1.0						

Aluminium

Constraint based Euler Lagrange coupling



Lagrangian

ALE

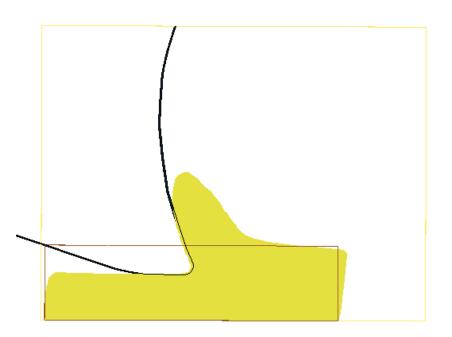
Number coupling points

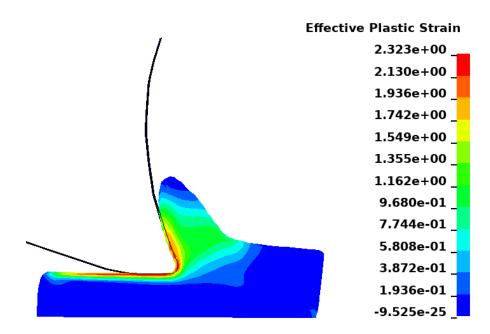
Cutting example

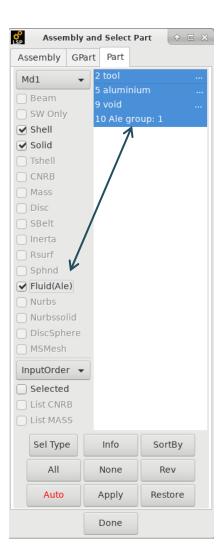
Post processing – LS-PREPOST

DYNA

Turn on ALE part in Assembly and Select Part









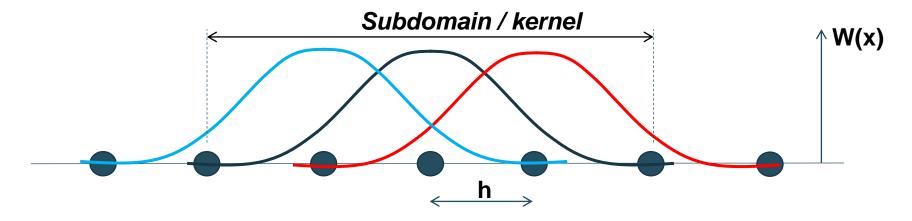
Meshless Methods

SPG/EFG and SPH

Introduction to meshfree methods



- A meshfree method is not a discrete particle method It is a continuum based method
 - In meshfree methods the particles serve as computational points for field variables inside a continuum
- The influence between the particles are based on kernel functions
 - The smoothing/interpolating kernel makes the kernel functions continuous and differentiable
- Handles arbitrary large deformations
- Timestep is insensitive to deformation
- Drawbacks of the method
 - Treatment of essential boundary conditions is difficult cannot be described exactly
 - Lack of consistency (shape functions do not add to unity) renormalization of kernel function necessary
 - Tensile instability Unphysical/premature failure do to lack of influence between particles

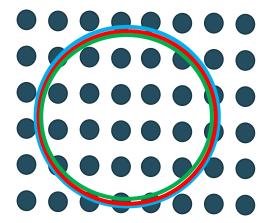


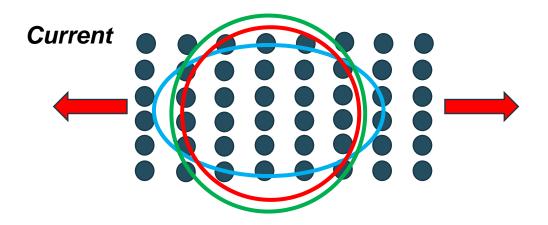
Introduction to meshfree methods - Kernel



- The type of kernel influences how the deformation of the material changes the structure of the subdomain
- Eulerian: Fixed kernel, the particles are moving in and out of the kernel domain
- Lagrangian: Updated kernel, the particles within the kernel domain stays the same
- Semi Lagrangian: The kernel is defined in the current configuration but the number of particles is the same
- Lagrangian kernel reduces tensile instabilities
- The subdomain size can be scaled
 - Large size is more numerically stable
 - Large size requires more CPU time
 - Ductile material requires a larger size while a semi- brittle can be handled with a smaller size

Initial

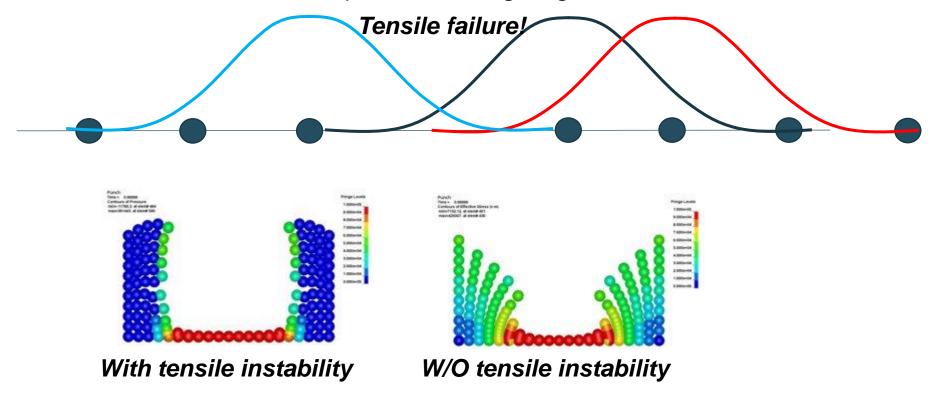




Introduction to meshfree methods – Tensile instabilities



- Tensile instabilitity is a premature failure which occurs when the influence from two nodes becomes too low and the "bond" fails
- The choice of kernel influences the sensitivity for tensile instabilities where in general an Eulerian kernel is more sensitive compared to a Lagrangian kernel



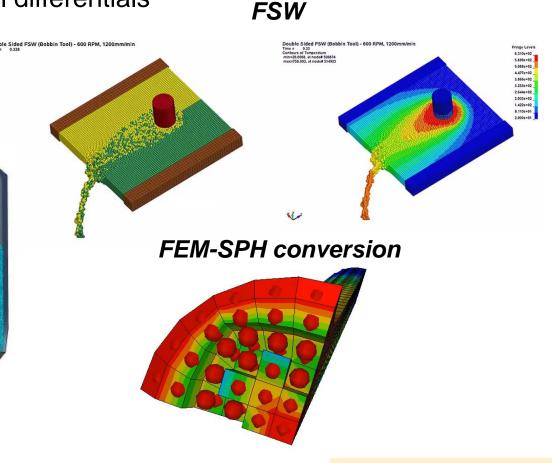
SPH – Smooth Particle Hydrodynamics

Sloshing



- Commonly used for fluids, materials mixing and large deformation simulations
- Uses Collocation method (strong form) to solve partial differentials

Special feature: FEA to sph conversion

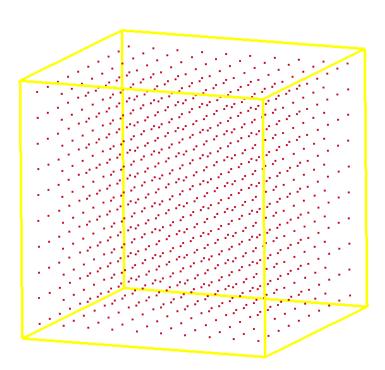


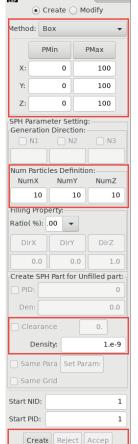
Preprocessing



SPH works best if the spacing between the particles is similar

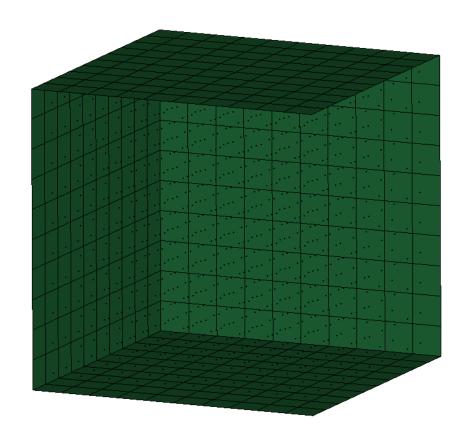
By Geometry (Box, cylinder, cone)

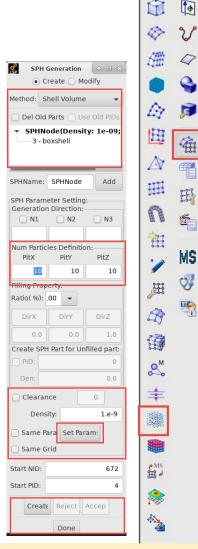




Done

By Shell Volume





LS-DYNA SPH keywords



*ELEMENT_SPH \$# nid pid mass 1005811 1 8.000000e-09

- mass is calculated by the pre- processor
- The radius of the particle is calculated from the mass and used in the contacts as particle thickness

*SEC	*SECTION_SPH									
\$#	secid	cslh	hmin	hmax	sphini	death	start	sphkern		
	1	1.20	0.2	2.0	0.01.0	00000E20	0.0	0		

- Possible scaling of the initial smoothing length
- Higher cslh increases simulation time
- Smoothing length is variable and hmin and hmax sets the maximum smoothing lengths
- Default are good values

*CON	TROL_SPH							
\$#	ncbs	boxid	dt	idim	nmneigh	form	start	maxv
	1	0	1.00E20	3	150	0	0.01.0	00000E15
\$#	cont	deriv	ini	ishow	ierod	icont	iavis	isymp
	0	0	0	0	0	0	0	100
\$#	ithk	istab	ql	_	sphsort			
	0	0	0.01		0			

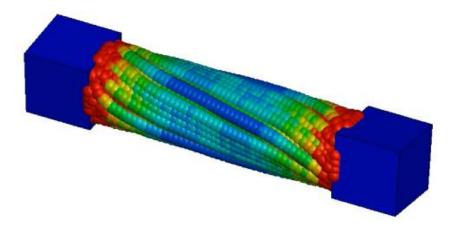
- idim: 2D/3D problem
- nmneigh: Might have to be increased depending on model
- form: Sets the kernel function type
- Artificial viscosity is always used to dissipate numerical high frequency noise

Connect SPH and FEM/Boundary conditions

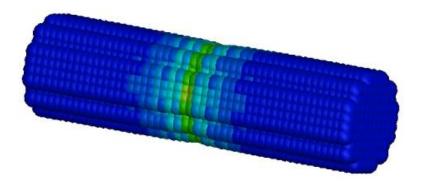


Building the FE model

■ SPH particles can be connected to FEM by *CONTACT TIED NODES TO SURFACE ...

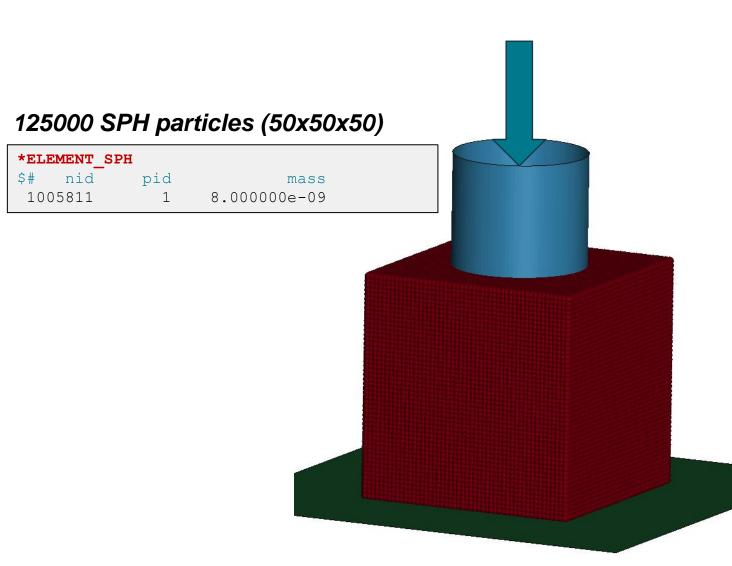


Boundary conditions (constraints and prescribed motions) using *BOUNDARY ...



Punch example





Automatic nodes to surface contacts with set particle radius

*CONTACT_AUTOMATIC_NODES_TO_SURFACE										
\$#	surfa	surfb	surfatyp	surfbtyp						
	6	2	4	3						
\$#	fs	fd	dc	VC						
	0.5	0.0	0.0	0.0						
\$#	sfs	sfm	sst	mst						
	1.0	1.0	1.0	0.0						

Default smoothing lengths

*SEC	*SECTION_SPH									
\$#	secid	cslh	hmin	hmax						
	3	1.20	0.2	2.0						

Default Eulerian kernel

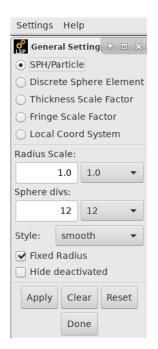
*CONTROL_SPH						
\$#	ncbs	boxid	dt	idim	nmneigh	
	1	01.00	000E20	3	150	

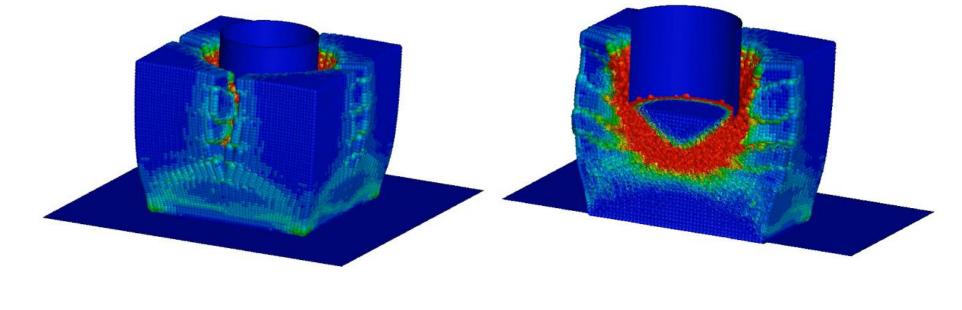
Postprocessing

DYNA

General Settings

- The postprocessing settings for SPH is found under General Settings
 - Radius scale factor. 1. is actual particle size
 - Sphere divs sets how smooth the visualization is
 - Style sets the particle style
- Fringe plot and cut model as usual





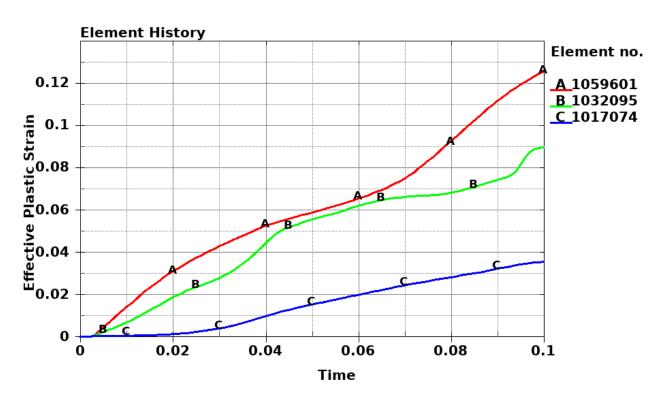
Postprocessing

Histories



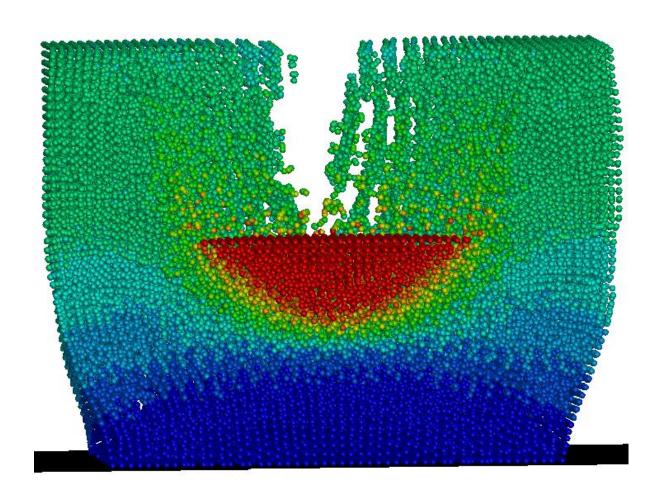
- Particle data can be History plotted as usual
 - Stresses
 - Displacements/Velocities

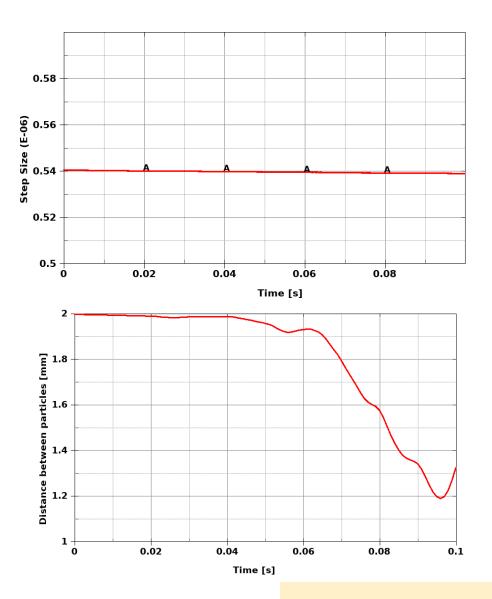




Punch example – Timestep



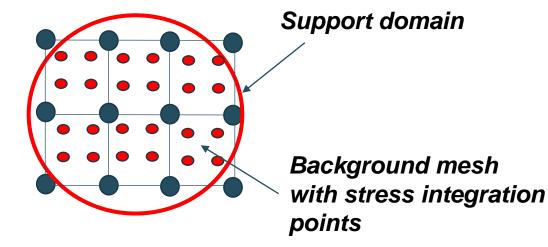




EFG – Element Free Galerkin



- EFG is a mix between FEM and SPH
 - Both EFG and other particle methods has a support domain where the approximated field variables are computed
 - EFG has a background mesh used to integrate the weak forms
- Approximating polynomials are constructed by minimizing the error with respect to the field values through Moving Least Squares (MLS)
- Background mesh makes an easier interpretation of the physical domain
 - Easier for boundary conditions and contacts
- Mesh distorsion issues!
 - Limits are further than for FEM
- Can be combined with 3D adaptivity to avoid mesh distorsion
- Background mesh makes it possible to use for coupled analyses



Preprocessing



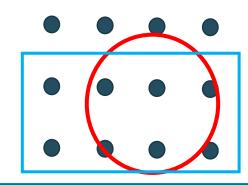
- The EFG particles are created automatically from the nodes of a solid element mesh
- Create the EFG background mesh as a usual solid element mesh (tetrahedrons, pentahedrons or hex)
- Change from *SECTION SOLID to *SECTION SOLID EFG
- The same 3D adaptive keywords apply to the EFG remeshing setup
- Since EFG method is formulated with a background mesh, contacts, loads etc is applied as for a regular lagrangian FEM model

LS-DYNA EFG keywords



*SEC	*SECTION_SOLID_EFG							
\$#	secid	elform	aet	unused	unused	unused	cohoff	unused
	1	42	0				0	
\$#	dx	dy	dz	ispline	idila	iebt	idim	toldef
	1.1	1.1	1.1	0	0	1	2	0.01
\$#	ips							
	1							

	regular mesh	irregular mesh
Foam	1.0 ~ 1.2	1.0 ~ 1.2
Metal	1.2 ~ 1.4	1.0 ~ 1.2
Fluid or EOS	1.4 ~ 1.6	1.2 ~ 1.4



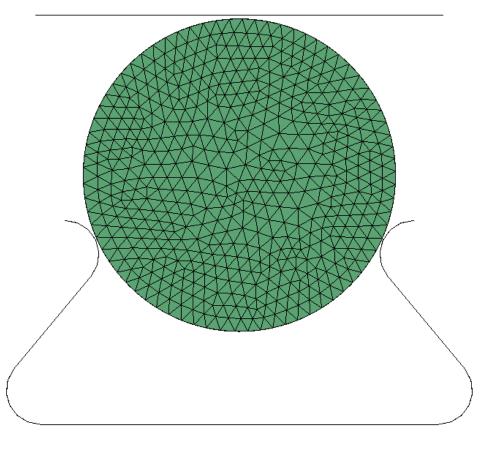
- elform: 41 is hex/penta and 42 is tetrahedron
- dx, dy, dz: Scales the dilatation parameters
- ispline: Choice of kernel function Quadratic/cubi, rectangular/circle support
- idim: Weak form integration. Local boundary or Gauss integration.
- toldef: Deformation tolerance for the activation of adaptive EFG Semi-Lagrangian and Eulerian kernel
- **ips**: Pressure recovery (ELFORM=42)

*CONTROL REMESHING EFG rmin vf loss mfrac cid rmax dt min icurv segang 0.01 0.1 ivt iat iaat ier mm \$# iat2 iat3 iat1

- rmin, rmax: Minimum and maximum edge length
- iat: Flag for interactive adaptivity
- iat1, iat2, iat3: Tolerance for interactive criteria: shear strain, element edge ratio and volume change

Rubber compression example





Automatic surface to surface contact

*CONTACT_AUTOMATIC_SURFACE_TO_SURFACE									
\$#	surfa	surfb	surfatyp	surfbtyp					
	6	2	4	3					
\$#	fs	fd	dc	VC					
	0.05	0.0	0.0	0.0					

Default kernel

\$# secid elform aet 1	*SE	*SECTION_SOLID_EFG											
	\$#	secid	elform	aet									
		1	42	0				0					
1.1 1.1 1.1 0 0 1 2	\$#	dx	dy	dz	ispline	idila	iebt	idim					
		1.1	1.1	1.1	0	0	1	2					

Constant adaptive frequency

```
*CONTROL_ADAPTIVE

$# adpfreq adptol adptyp

0.011.00000E20 7
```

Element length based remeshing

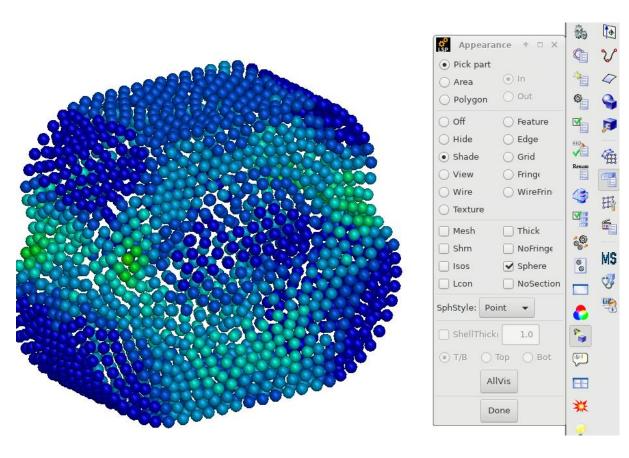
```
*CONTROL_REMESHING

$# rmin rmax vf_loss

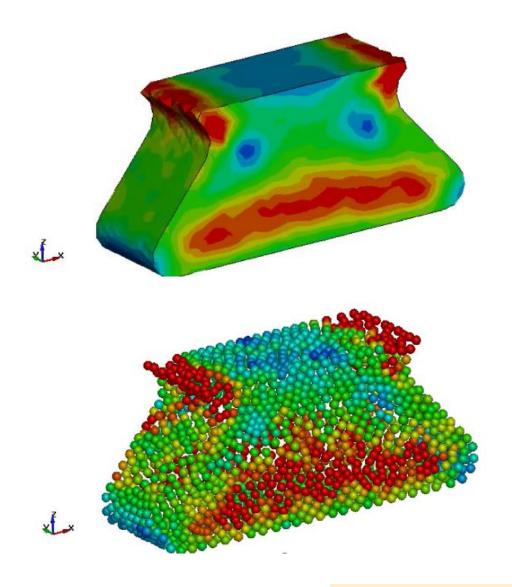
1.5 2.5 1.0
```

Postprocessing





Spherical nodal representation



SPG – Smooth Particle Galerkin



- SPG is a truly mesh free method similar to SPH
- SPG has a Galerkin weak formulation
- A displacement smoothing technique is applied to stabilize low energy modes occurring from the direct nodal integration (DNI)
 - The displacement smoothing function adds a stabilizing term to the equilibrium equation
- Bond based failure is possible by removing the failed bond in the neighbour sorting algorithm
- Bond based failure criteria in *SECTION card
- Damage model by *MAT ADD DAMAGE GISSMO
- Timestep is not influenced by deformation

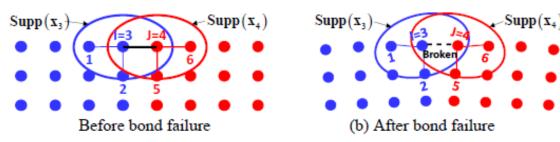


Figure 1. Illustration of SPG bond failure mechanism.

Preprocessing



- The SPG particles are created automatically from the nodes of a solid element mesh
- Change from *SECTION_SOLID to *SECTION_SOLID_SPG
- LS-DYNA automatically changes from FEM to SPG, no SPG particle definition needed
- SPG particles can be connected to FEM by node merge or tied contacts.
- Use node to surface contacts for SPG to FEM contact
- Multiple stage analysis is possible. *INTERFACE_SPG_1 outputs a result file and
 *INTERFACE SPG 2 indicates that the spg parts should be initialized from the result file.

LS-DYNA SPG keywords



*SE	CTION_SOL	ID_SPG							
\$#	secid	elform	aet	unused	unused	unused	cohoff	unused	
	3	47	0				0		
\$#	dx	dy	dz	ispline	kernel		smstep	msc	
	1.5	1.5	1.5	0	1	0.0	0	0.0	
\$#	idam	fs	stretch	itb	unused	unused	unused	pdamp	
	1	0.4	0.0	3				-0.001	

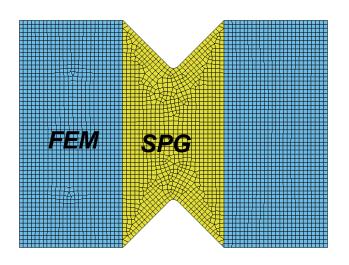
- elform: 47 is SPG element
- dx, dy, dz: Scales the dilatation parameters. 1.4 to 1.8 are recommended
- ispline: Choice of kernel function Quadratic/cubic
- kernel: Choice of kernel
- idam: Bond failure mechanism (e.g. effective plastic strain, shear strain..)
- **fs**: Critical bond fail value
- itb: Stabilization option
- pdamp: Particle to particle damping (itb=3)

*C	ONTACT_SPG	;						
\$#	pid1	pid2	pid3	pid4	pid5	pid6	pid7	pid8
	1	2						
\$#	iself1	iself2	iself3	iself4	iself5	iself6	iself7	iself8
	1							
\$#	pfac1	pfac2	pfac3	pfac4	pfac5	pfac6	pfac7	pfac8
	0.1	0.1						
\$#	fs	fd	dc	nfreq				
	0.12							

- pid#: SPG part in contact
- iself#: Self contact flag
- pfac#: Penalty factor
- **fs**: Friction coefficient
- nfreq: Contact update frequency

SPG and GISSMO damage model example

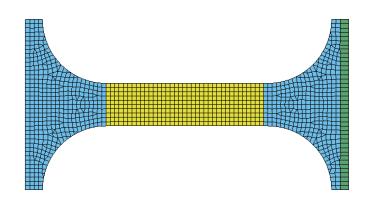




Eulerian kernel with MCSPG stabilization

*SEC	*SECTION_SOLID_SPG											
\$#	secid	elform	aet	unused	unused	unused	cohoff	unused				
	2	47	0				0	0				
\$#	dx	dy	dz	ispline	kernel	lscale	smstep	swtime				
	1.8	1.8	1.8	0	1	0.0	0	0.0				
\$#	idam	fs	stretch	itb	unused	unused	unused	unused				
01.00000E28		0.0	3				-0.005					

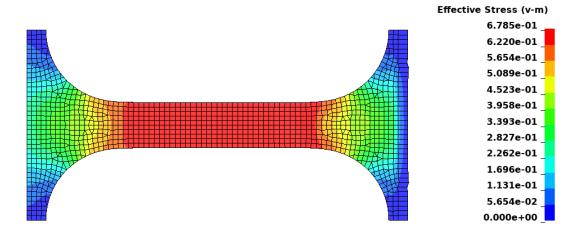
Add on GISSMO failure model

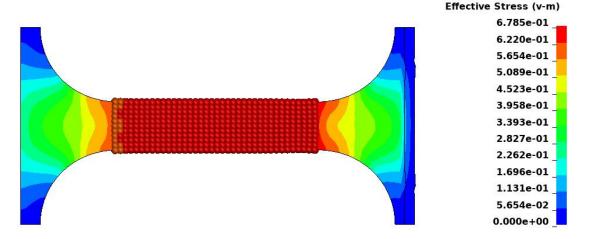


*MA	r_ADD_DAMA	GE_GISSMO						
\$#	mid	_	dtyp	refsz	numfip			
	5		1.0	0.0	1.0			
\$#	lcsdg	ecrit	dmgexp	dcrit	fadexp	lcregd		
	5	0.1	1.0	0.0	1.0	4		
\$#	lcsrs	shrf	biaxf	lcdlim	midfail	hisvn	soft	lp2bi
	0	0.0	0.0	0	0.0	0.0	0.0	0.0

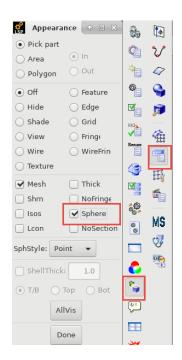
Postprocessing SPG





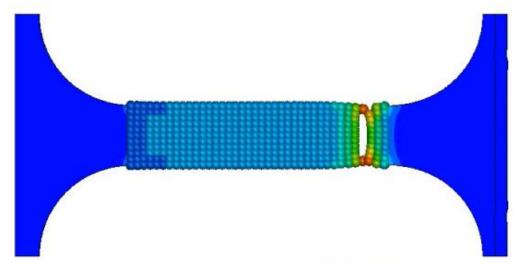


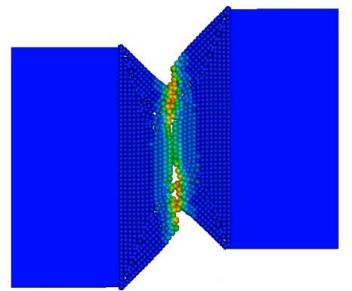
- Results can be postprocessed on element or particle
- Switch to Spere representation under Model and Part/Appear

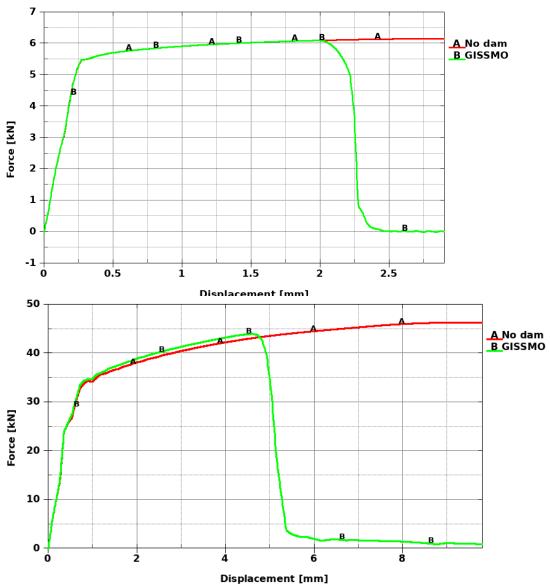


Postprocessing SPG











Summary

Summary



- LS-DYNA has several options if you want to do large deformation analysis
- The methods ranges from Lagrangian FEM with remeshing to Eulerian FEM, through EFG to element free methods like SPG and SPH
- The type of spatial discretization you choose depends on your application and the aim of your analysis.

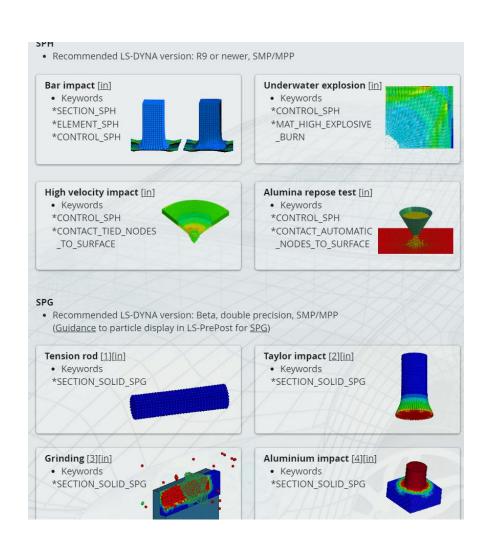
	FEM	Remeshing	ALE/S-ALE	EFG	SPH	SPG
Large Deformations	No	Yes	Yes	Yes (with adaptivity)	Yes	Yes
Failure	Damage	Material model (no separation)	Material model	Material model (no separation)	Material model	Damage
Thermal	Coupled	Coupled	Adiabatic	Coupled	Adiabatic	Adiabatic
Primary application	Structural	Forging/ Extrusion	Fluid/High speed impact	Forging/ Extrusion	Fluids/Cutting	Failure

Further information



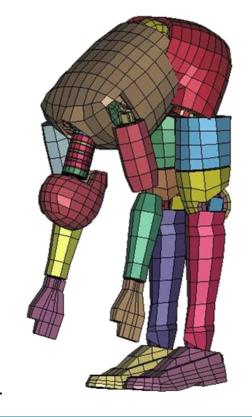
- LST Computational and Multi-Scale Mechanics group
 - https://www.lstc-cmmg.org/multiphysics
- YouTube
 - Dynamore Express
 - Beyond FEA The Element-Free Galerkin (EFG) Method
 - Beyond FEA Smoothed Particle Hydrodynamics (SPH)
 - Beyond FEA Arbitary Lagrangean-Eulerian (ALE) Method

- Dynalook
 - Papers from European and International conferences





Thank You



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