#### DYNAmore Nordic Breakfast Seminar 2019

# State of the art modelling of thermo-mechanical coupling in LS-DYNA

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#### **Motivation – Assembly Simulation**

- State of the art digital process chain contains
  - (Hot) forming and press hardening simulations
  - Clamping simulations
  - Mechanical assembly steps, i.e. clinching, roller hemming, ...
  - Thermal assembly steps, i.e. resistance spot welds, laser welds, line weld (MIG, MAG), ...
  - Springback analysis
- Closed virtual process chain within LS-DYNA by data transfer from one stage to the next
  - Assembly of whole side-panel of a car
  - Hundreds of spot-weld, dozens of parts and multiple level of assemblies
- Tailored simulation strategies for each of the individual steps
  - As efficient as possible for each process, but without neglecting the critical effects
  - Keep track of material properties that might change significantly during process (e.g. phase evolution)





#### Content

- 2-way coupled approaches
  - Data transfer and simulation principles
  - Application example: laser cutting
- Interlude: Hot forming and press-hardening simulations
- 1-way coupled approaches
  - Motivation and data transfer
  - New keyword \*LOAD\_THERMAL\_BINOUT
  - Application example
- Prescribed thermal loading conditions for resistance spot welding
  - Motivation
  - New keyword \*LOAD\_THERMAL\_RSW





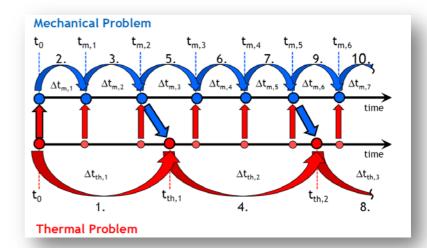
#### **Data Transfer and Simulation Principles**

- Default strategy in LS-DYNA is a 2-way coupling
  - Staggered weak approach
  - Two solvers run in parallel and share data
  - Thermal time step is independent of the mechanical time step

Data transfer

#### Thermal Calculations

- Based on current geometry, calculate:
  - Heat from plastic work and phase changes
  - Contact conductance from gap thickness and contact pressure
  - Heat from interface friction
- Update temperature field



#### **Mechanical Calculations**

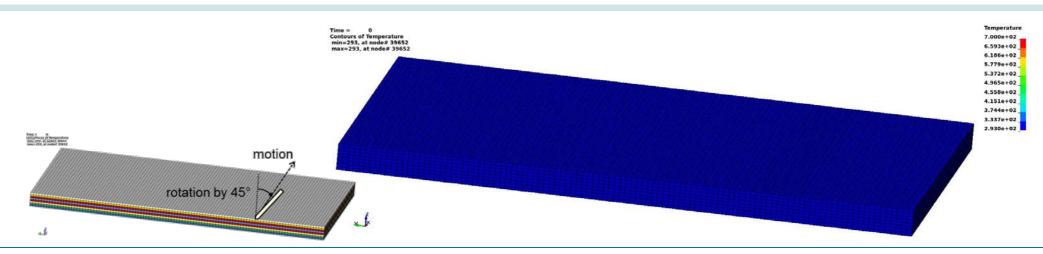
- Based on current temperature, calculate:
  - Plastic work and phase changes
  - Part contact gap thickness
  - Temperature dependent material
  - Thermal expansion
- Update geometry





## 2-way Coupled Application: Laser Cutting

- Strong coupling between structure and temperature field
  - local heating of a surface by a laser with a certain position and orientation
  - material evaporates and topology of cut part changes
- LS-DYNA implementation with \*BOUNDARY\_FLUX\_TRAJECTORY
  - surface flux boundary conditions that follows a prescribed path (node set)
  - resulting surface heat distribution depends on base distribution and current orientation of laser and surface
  - element erosion based on maximum temperature
  - newly exposed segments are accounted for

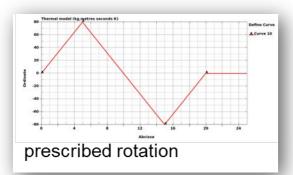


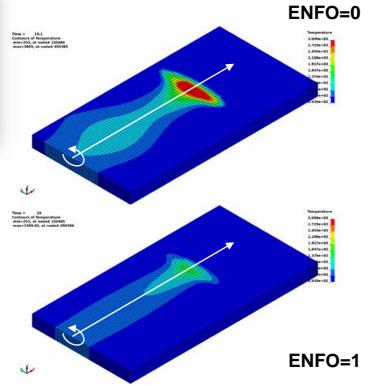




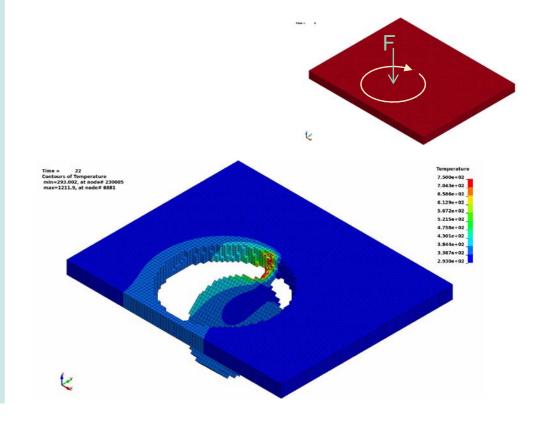
# 2-way Coupled Application: Laser Cutting

- \*BOUNDARY\_FLUX\_TRAJECTORY
  - tilting changes projection on the surface
  - change of intensity can be balanced





 nodal path not necessarily defined on the cut part







# Interlude: Hot forming and press-hardening simulations

- Motivation
- Selected simulation topics
  - Thermal thick shell
  - Heat transfer in contact
  - Direct vs iterative solvers and tolerances
- Material modeling
  - MAT\_244
  - MAT\_254

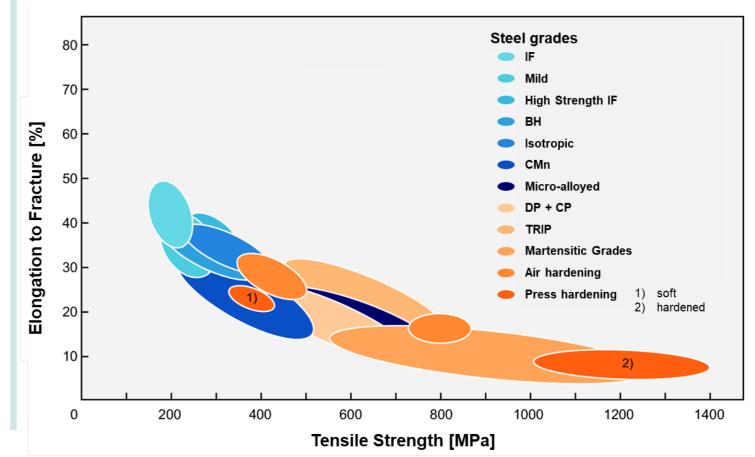




#### **Motivation I: Steel Grades**

- goal: benefit from the advanced characteristics of UHS steels
- properties of the product are process dependent (mainly of cooling rate)
- relatively well-known and wellcontrolled environment

#### [source: Hochholdinger 2012]







# **Motivation: "Real" Process**



**Courtesy of Daimler AG** 





# **Coupled Warmforming Simulations**



- Transfer
  - heat loss by
    - convection
    - radiation



- Positioning
  - heat loss by
    - convection
    - radiation
    - contact to lower die
  - temperature gradient through blank



- Hot forming & quenching
  - heat change by
    - contact conductance
    - contact friction
    - solid solid phase transition
    - die cooling





# Interlude: Hot forming and press-hardening simulations

- Motivation
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- Material modeling
  - MAT\_244
  - MAT\_254

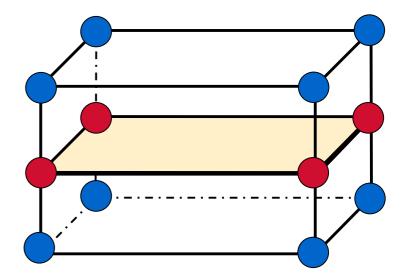




#### **Thermal Thick Shell Element**

- standard shell element has a constant temperature over thickness
- LS-DYNA features a twelve-node thermal thick shell element formulation.
  - bi-linear shape functions in-plane, quadratic shape function in thickness direction
     (G. Bergman & M. Oldenburg, Luleå University of Technology)
  - user only specifies standard four node shell element
  - LS-DYNA automatically generates top and bottom virtual nodes
  - new temperature degrees of freedom assigned to virtual nodes

nodes provided by user 
virtual nodes

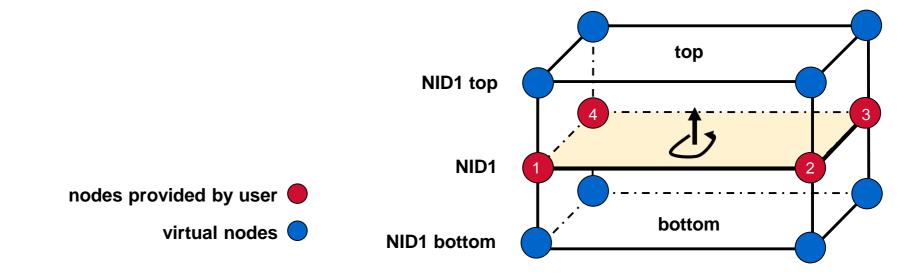






#### **Thermal Thick Shell Element**

- Implications on initial and boundary conditions
  - shell has top, middle and bottom layer
    - → surface conditions have to be applied to the correct layer
  - each node ID occurs 3-times (top, middle, bottom)
    - → need input possibility for all three nodes to define nodal conditions







#### **Thermal Thick Shell Element**

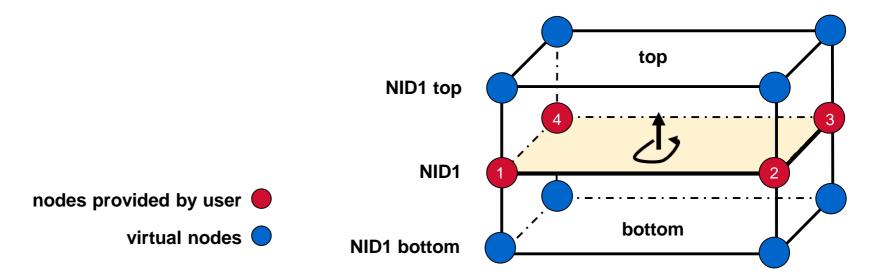
General keyword form of a thermal boundary/initial condition

LCID	MULT	LOC	
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LCID: load curve ID (function of time or temperature)

MULT: load curve multiplier

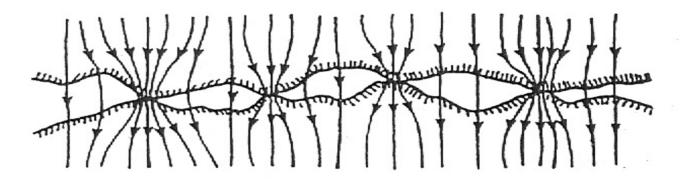
LOC: shell / node layer (-1: bottom, 0: middle, 1: top)







#### **Contact conductance**



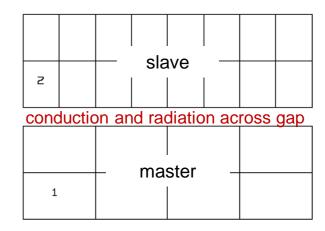
- Heat transfer between contacting parts is influences by
  - contact pressure (elastic/plastic deformation)
  - contact temperature
  - material in gap (vacuum, gas, lubricant)
  - surface flatness
  - surface roughness
  - surface finish (e.g., oxide layer)



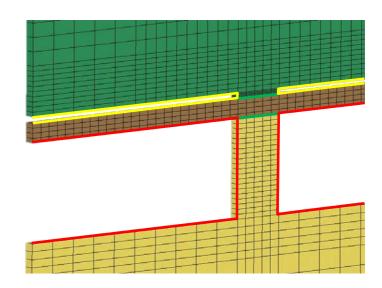


#### **Contact conductance**

- Three modes of heat transfer:
  - Conduction across fluid filling gap with variable gap thickness
  - Radiation across gap
  - (Metal-to-metal) contact with voids, i.e. closed contact



- Heat exchange by radiation
  - Contact radiation, where a segment can see one surface on the other body
  - Enclosure radiation, where a segment can see many other segments on the other body
  - Borderline between the above cases







#### \*CONTACT\_FORMING\_ONE\_WAY\_SURFACE\_TO\_SURFACE\_THERMAL\_FRICTION

#### FORMING

- do not account for master surface shell thickness
- special mesh adaptivity algorithms available

#### ONE\_WAY

- only affects contact treatment of mechanical solver
- consider only slave penetration into master surfaces

# slave master

#### THERMAL

- additional card required to input parameters for heat transfer across contact area
- switch for thermal one-/two-way

#### FRICTION

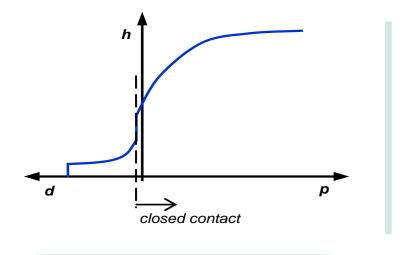
- additional card is expected
- define coefficient of static and dynamic friction as function of temperature and pressure
- define heat transfer coefficient as function of temperature and pressure





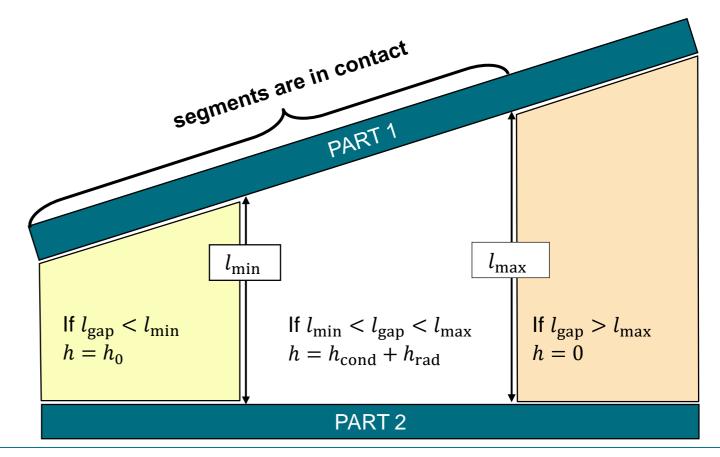
# Thermal contact keyword

	1	2	3	4	5	6	7	8
Therm.	K	FRAD	н0	LMIN	LMAX	FTOSLV	BCFLAG	ALGO



$$h_{\text{cond}} = \frac{k}{l_{\text{gap}}}$$

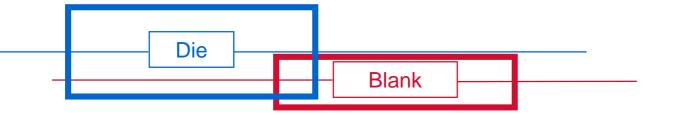
$$h_{\text{rad}} = f_{\text{rad}}(T_{\text{m}} + T_{\text{s}})(T_{\text{m}}^2 + T_{\text{s}}^2)$$



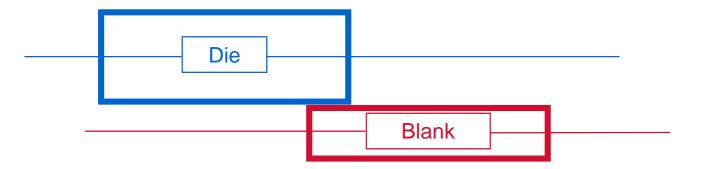


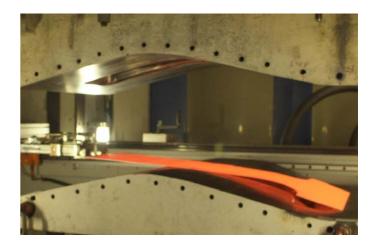
# **Defining surfaces in contact**

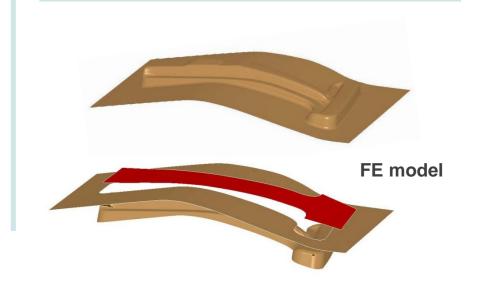
- There are two choices for structural contact between shells
  - die mid-plance (CAD defined surface) contacts blank surface



die surface contacts blank surface











#### **Defining surfaces in contact**

die mid-plane contacts blank surface



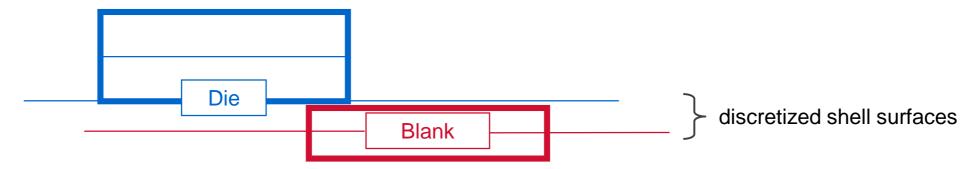
- CAD software precisely defines the coordinates of the die mid-plane surface
- activated with
  - \*CONTACT\_FORMING\_SURFACE\_TO\_SURFACE
- modelling technique
  - die is a rigid material and does not deform.
  - thermal analog is that the die does not change temperature (ALGO=1).
  - die is given a rigid body motion and a temperature boundary condition.
  - mid-plane of the die is the reference surface for mechanical and thermal contact.





#### **Defining surfaces in contact**

offset die surface contacts blank surface



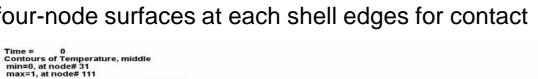
- activated with
  - \*CONTACT\_FORMING\_SURFACE\_TO\_SURFACE
  - \*CONTACT\_SURFACE\_TO\_SURFACE
  - \*CONTROL\_CONTACT attribute THOFF=1
- Modeling technique
  - die is a rigid material or an elastic material.
  - die is given an initial temperature.
  - due to contact, a temperature gradient is calculated through the die thickness.



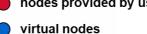


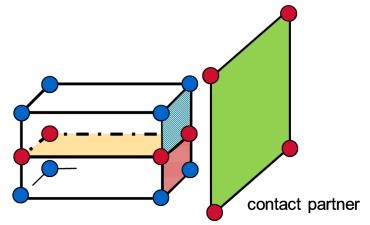
# **Heat Transfer over Shell Edges in Contact**

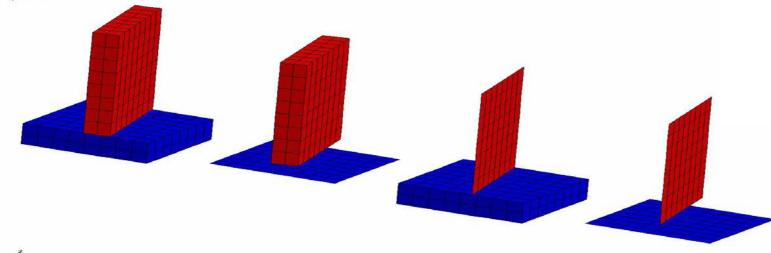
- Situation so far:
  - heat transfer only available for surface to surface type contact formulations
  - for shell contacts only heat flux normal to shell surface implemented
- Thermal thick shells allow for reconstruction of two
  - four-node surfaces at each shell edges for contact









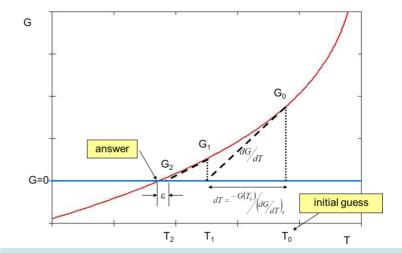


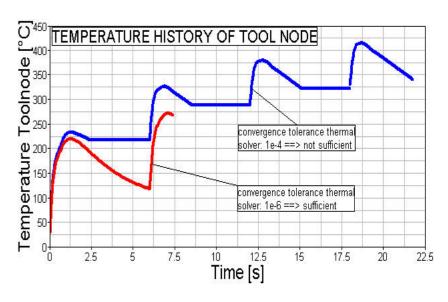




# **Linear Solver Settings for thermal solution in LS-DYNA**

- For non-linear problems, a Newton's method is applied
  - iterative solution scheme to find the root of a function
  - sequence of linear problems to be solved
  - solution always with respect to a certain tolerance
- Linear solution controlled by \*CONTROL\_THERMAL\_SOLVER
  - direct and iterative solvers are available
  - direct solvers
    - find "exact" root up to numerical precision
    - relatively high CPU and memory consumption
  - iterative solvers
    - efficient in terms of CPU and memory
    - iteration converged if error is below a given tolerance
  - Tolerances of iterative solvers problem and unit system specific









# Interlude: Hot forming and press-hardening simulations

- Motivation
- Selected simulation topics
  - Thermal thick shell
  - Heat transfer in contact
  - Direct vs iterative solvers and tolerances
- Material modeling
  - MAT\_244
  - MAT\_254





# \*MAT\_UHS\_STEEL / \*MAT\_244 - Overview

- based on the work by P. Åkerström
- tailored for hot stamping / press hardening processes
- accounts for austenite decomposition into ferrite, pearlite, bainite, and martensite as well as reaustenitization
- mechanical features:
  - elasto-plastic material with a von-Mises plasticity model
  - temperature and strain-rate effects
  - transformation induced strains and plasticity
  - thermal expansion
- any mechanical quantity  $\alpha$  is determined by a rule of mixtures based on the current phase fractions  $x_i$  and the quantity  $\alpha_i$  of phase i:

$$\alpha = \sum_{i=1}^{5} x_i \alpha_i$$





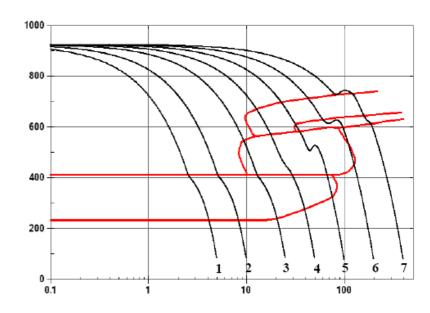
#### \*MAT\_UHS\_STEEL / \*MAT\_244 - Overview

#### User input:

- alloying elements in mass percentB, C, Co, Mo, Cr, Ni, V, W, Cu, P, Al, As, Ti
- latent heats for phase change reaction
- activation energy for phase transformation
- initial grain size
- yield curves for each phase
- coefficients of thermal expansion

#### material output

- current phase fraction of ferrite, pearlite, bainite and martensite
- computed Vickers hardness
- resulting yield strength



recalculated CCT diagramm





#### \*MAT\_UHS\_STEEL / \*MAT\_244 - Limitations

#### Process limitations

- tailored for hot stamping / press hardening processes
- basic welding functionality with a ghosting apporach has been added
- only limited capability for phase transformations during heating phases
- no methodology for tempering available

#### Material limitations

- restriction to five phases
- only feasible for 22MnB5:
  - heuristic formulas for start and end temperatures of phase transformation
  - empirical equations for the phase transformation parameters





#### \*MAT\_254 - Overview

- up to 24 individual phases (= 552 possible phase change scenarios)
- phase changes in heating, cooling or in a temperature window
- user can chose from a list of phase change models for each scenario
- basic mechanical features:
  - elasto-plastic material with a von-Mises plasticity model
  - temperature and strain-rate effects
  - transformation induced strains and plasticity
  - thermal expansion
- any mechanical quantity  $\alpha$  is determined by a rule of mixtures based on the current phase fractions  $x_i$  and the quantity  $\alpha_i$  of phase i:

$$\alpha = \sum_{i=1}^{24} x_i \alpha_i$$





# \*MAT\_254 / \*MAT\_GENERALIZED\_PHASE\_CHANGE

	1	2	3	4	5	6	7	8
Card 1	MID	RHO	N	E	PR	MIX	MIXR	
Card 2	TASTART	TAEND	TABCTE				DTEMP	
Card 3	PTLAW	PTSTR	PTEND	PTX1	PTX2	PTX3	PTX4	PTX5
Card 4	PTTAB1	PTTAB2	PTTAB3	PTTAB4	PTTAB5	PTTAB6		
Card 5	PTEPS	TRIP			NUSHIS	GRAI	T1PHAS	T2PHAS
Card 6	FUNUSH1	FUNUSH2	FUNUSH3	FUNUSH4	FUNUSH5	FUNUSH6	FUNUSH7	FUNUSH8
Card 7	LCY1	LCY2	LCY3	LCY4	LCY5	LCY6	LCY7	LCY8
Card 8	LCY9	LCY10	LCY11	LCY12	LCY13	LCY14	LCY15	LCY16
Card 9	LCY17	LCY18	LCY19	LCY20	LCY21	LCY22	LCY23	LCY24

- very general material implementation to capture micro-structure evolution
- implementation available for solids and shells and for explicit and implicit





#### \*MAT\_254 – Overview

- elaborate features:
  - latent heat algorithm
  - calculation and output of additional pre-defined post-processing histories
  - calculation and output of additional user-defined history values
    - refers to \*DEFINE\_FUNCTION keyword
    - Possible input: time, user-defined histories, phase concentrations, temperature, peak temperature, temperature rate, stress state, plastic strain data
  - enhanced annealing option by evolution equation for plastic strain depending on time and temperature





#### \*MAT\_254 - Phase transformation

	1	2	3	4	5	6	7	8
Card 3	PTLAW	PTSTR	PTEND	PTX1	PTX2	PTX3	PTX4	PTX5
Card 4	PTTAB1	PTTAB2	PTTAB3	PTTAB4	PTTAB5			

#### microstructural phase evolution

- up to 24 individual phases
- parametrization of the phase transformation to be given in a matrix-like structures (\*DEFINE\_TABLE\_2D/3D)
- matrix input for
  - phase transformation law (2D)
  - start and end temperatures (2D)
  - transformation constants (2D)
  - temperature dependent parameters (3D)

		to							
		ph. 1	ph. 2	ph. 3		ph. n			
	ph. 1	-	data	data		data			
	ph. 2	data		data		data			
from	ph. 3	data	data			data			
			:	÷					
	ph. n	data	data	data					





#### \*MAT\_254 - Phase transformation

	1	2	3	4	5	6	7	8
Card 3	PTLAW	PTSTR	PTEND	PTX1	PTX2	PTX3	PTX4	PTX5
Card 4	PTTAB1	PTTAB2	PTTAB3	PTTAB4	PTTAB5	PTTAB6		

- Johnson-Mehl-Avrami-Kolmogorov (JMAK):
  - Evolution equation:

$$\frac{dx_b}{dt} = n(T)(k_{ab}x_a - k'_{ab}x_b) \left( \ln \left( \frac{k_{ab}(x_a + x_b)}{k_{ab}x_a - k'_{ab}x_b} \right) \right)^{\frac{n(T) - 1.0}{n(T)}}$$

$$k_{ab} = \frac{x_{eq}(T)}{\tau(T,\varepsilon^p)} f(\dot{T}), k'_{ab} = \frac{1.0 - x_{eq}(T)}{\tau(T,\varepsilon^p)} f'(\dot{T}),$$
  
$$\tau(T,\varepsilon^p) = \tau^0(T) \cdot \alpha(\varepsilon^p)$$

#### Parameter:

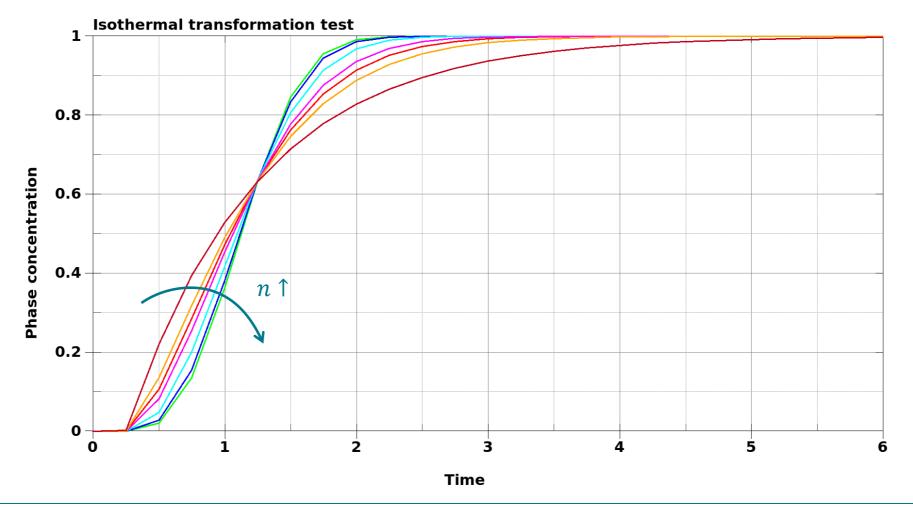
- PTTAB1: n(T)
- PTTAB2:  $x_{eq}(T)$
- PTTAB3:  $\tau^0(T)$
- PTTAB4:  $f(\dot{T})$
- PTTAB5:  $f'(\dot{T})$
- PTTAB6:  $\alpha(\varepsilon^p)$





# \*MAT\_254 - Phase transformation validation

influence of parameter n(T) on isothermal transformation

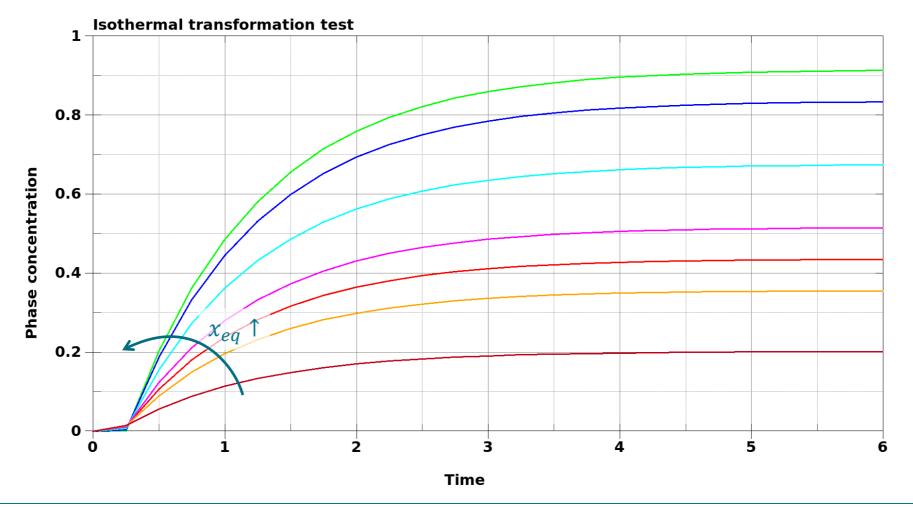






# \*MAT\_254 – Phase transformation validation

influence of parameter  $x_{eq}(T)$  on isothermal transformation

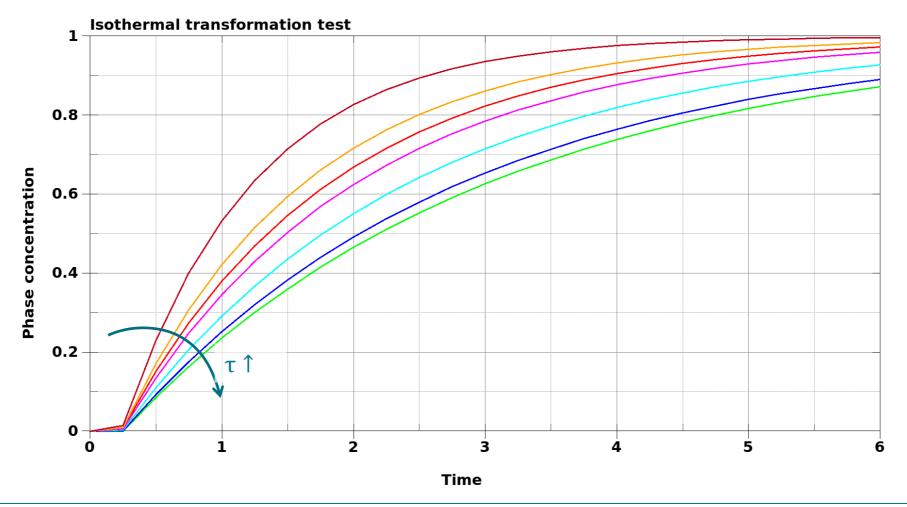






# \*MAT\_254 – Phase transformation validation

influence of parameter  $\tau(T)$  on isothermal transformation







# **Applications**

- Different steel grades in welding simulations
- New UHS steels in press-hardening and hot forming
- Bake hardening of aluminum 6xxx









### Content

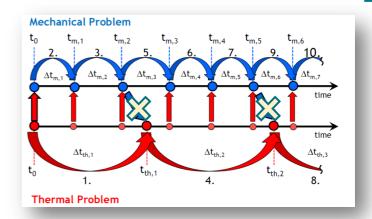
- 2-way coupled approaches
  - Data transfer and simulation principles
  - Application example: laser cutting
- Interlude: Hot forming and press-hardening simulations
- 1-way coupled approaches
  - Motivation and data transfer
  - New keyword \*LOAD\_THERMAL\_BINOUT
  - Application example
- Prescribed thermal loading conditions for resistance spot welding
  - Motivation
  - New keyword \*LOAD\_THERMAL\_RSW





## **Motivation for 1-way Coupling**

- For some assembly stages the effect of structural deformation onto the thermal simulation is negligible
  - Distortion and/or material phase evolution due the thermal distribution are of interest to the user



- Results of a thermal run serves as loading for structure simulation with \*LOAD\_THERMAL\_D3PLOT
  - Evolution in time of temperature distribution linearly interpolated between the output time steps
  - Thermal thick shell feature is supported also for the structure-only simulation
  - Temperature results are read from the d3plot file of the thermal run

### Challenges with this approach:

- Complex input file format (d3plot) to be generated by a mapping tool
- Meshes (models!) for both simulations have to coincide
- Time scaling has to match as well
- Implemented more flexible \*LOAD\_THERMAL\_BINOUT to read data from one or more LSDA database(s)





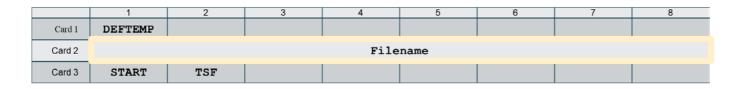
	1	2	3	4	5	6	7	8			
Card 1	DEFTEMP										
Card 2		Filename									
Card 3	START	TSF									

### Aims and scope of the new keyword

- Use flexible and open LSDA data format to define thermal loading of a structure
- Required structure of LSDA files matches the TPRINT section in LS-DYNA binout file, so results from thermal and from coupled LS-DYNA runs can be used without further modification
- Only partial overlap between meshes should be required
- Allow for a sequential thermal loading and for an easy modification of the sequence

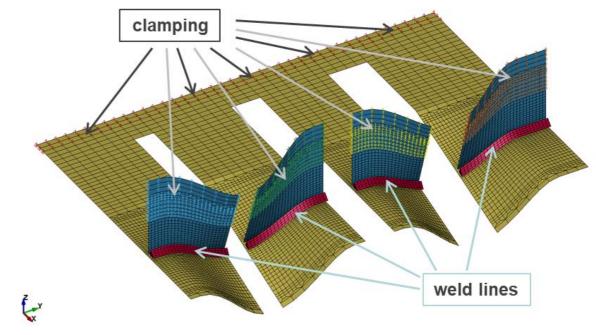






- File name of thermal run given in keyword
- Thermal thick shells are accounted for
- Time step sizes do not have to match

### **Welding Example:**

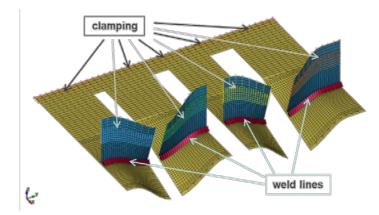




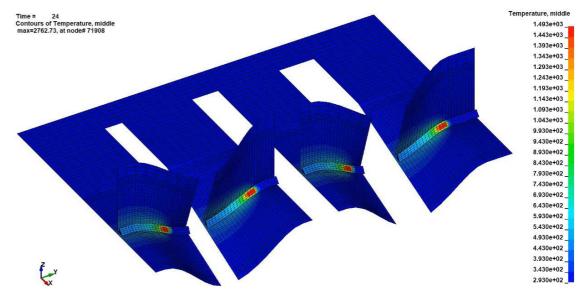


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Card 1	DEFTEMP											
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Card 3	START	TSF										

- File name of thermal run given in keyword
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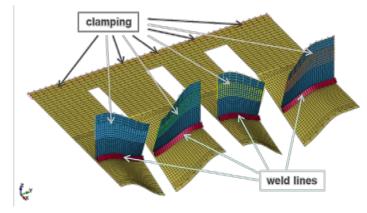




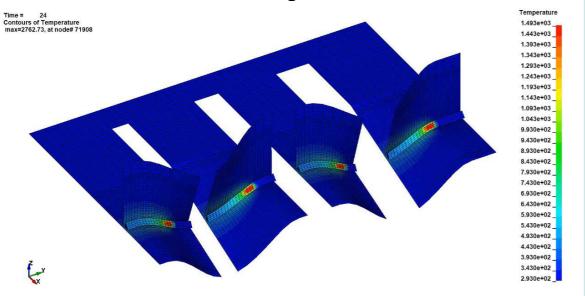


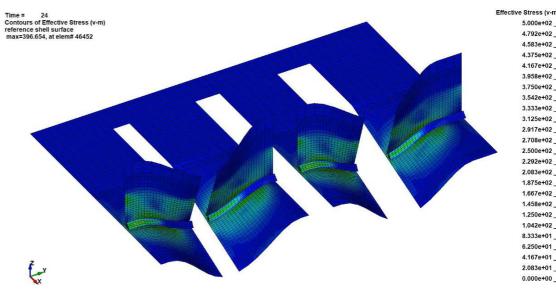


	1	2	3	4	5	6	7	8				
Card 1	DEFTEMP											
Card 2		Filename										
Card 3	START	TSF										



#### Structure run with thermal loading:





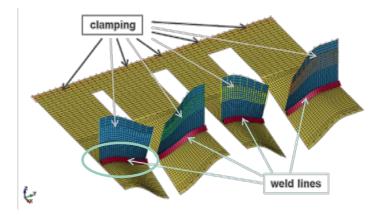
von Mises stress



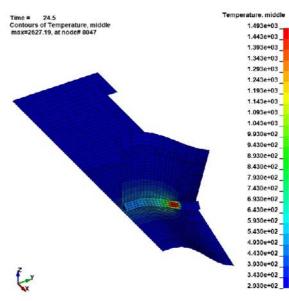
**Temperature** 

	1	2	3	4	5	6	7	8			
Card 1	DEFTEMP										
Card 2	Filename										
Card 3	START	TSF									

- File name of the input is to be given in the keyword
- Thermal thick shells are accounted for
- Time step sizes do not have to match
- Only partial overlap of the meshes is required
  - Data transfer based on user given ID of the nodes
  - Default temperature is used for those nodes of the structure simulations that are not included in the thermal run



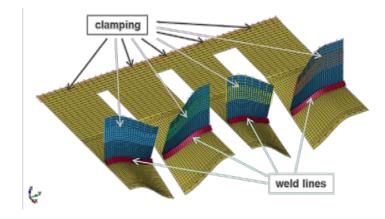
#### **Thermal Run:**



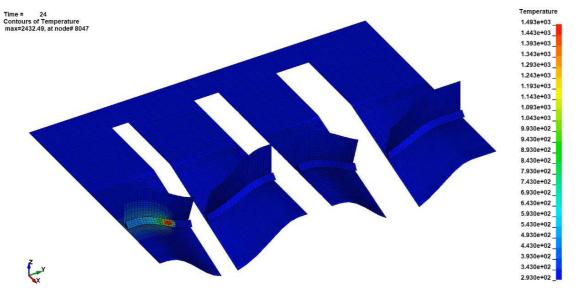




	1	2	3	4	5	6	7	8					
Card 1	DEFTEMP												
Card 2		Filename											
Card 3	START	TSF											



#### **Structure Run with partial overlap and DEFTEMP:**



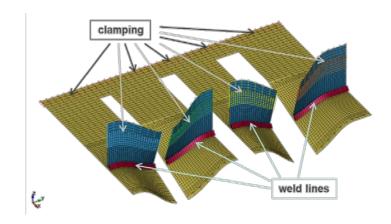


Temperature von Mises stress

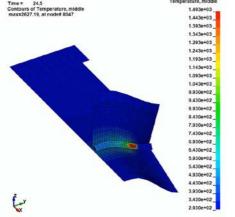


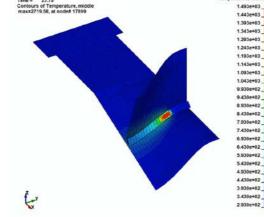
	1	2	3	4	5	6	7	8					
Card 1	DEFTEMP												
Card 2		Filename											
Card 3	START	TSF											

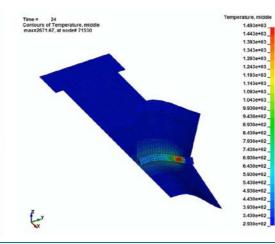
- Multiple thermal runs can be read in
- Each thermal run with time offset START
- Compensation for a scaling in time with TSF

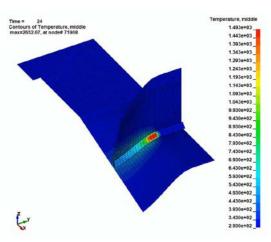


#### **Thermal Runs:**





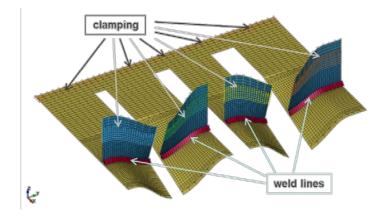






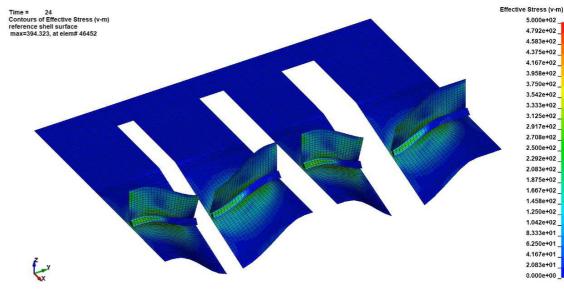


	1	2	3	4	5	6	7	8				
Card 1	DEFTEMP											
Card 2		Filename										
Card 3	START	TSF										



#### **Structure Run:**

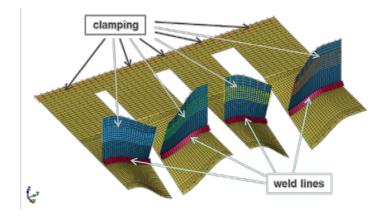




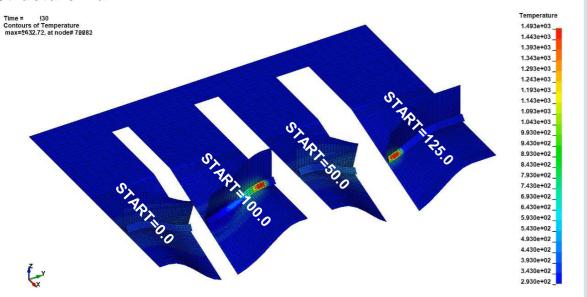
Temperature von Mises stress

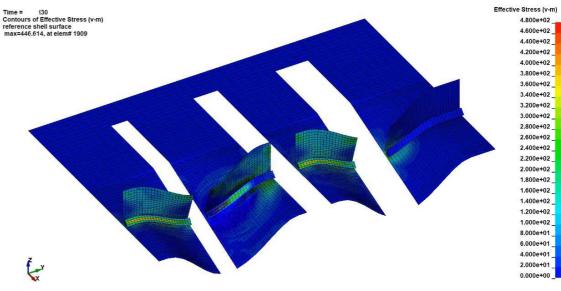


	1	2	3	4	5	6	7	8				
Card 1	DEFTEMP											
Card 2		Filename										
Card 3	START	TSF										



#### **Structure Run:**





von Mises stress



**Temperature** 

### Content

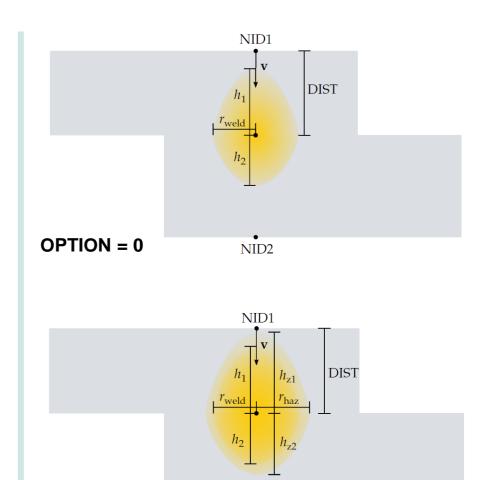
- 2-way coupled approaches
  - Data transfer and simulation principles
  - Application example: laser cutting
- Interlude: Hot forming and press-hardening simulations
- 1-way coupled approaches
  - Motivation and data transfer
  - New keyword \*LOAD\_THERMAL\_BINOUT
  - Application example
- Prescribed thermal loading conditions for resistance spot welding
  - Motivation
  - New keyword \*LOAD\_THERMAL\_RSW





## **Motivation**

- Modelling approaches for resistance spot welding (RSW)
  - Use a detailed and coupled (EM, thermal, structure) simulation
  - Use an equivalent heat source and calibrate power
- For large assemblies and hundreds of spot welds neither approach is feasible
- \*BOUNDARY\_TEMPERATURE\_RSW
  - Direct temperature definition (Dirichlet condition) for the weld nugget and the heat affected zone for the **thermal** solver
  - Constraint condition only active during the welding
  - Very good prediction of deflections in large assemblies
  - A HAZ can be additionally accounted for



NID2

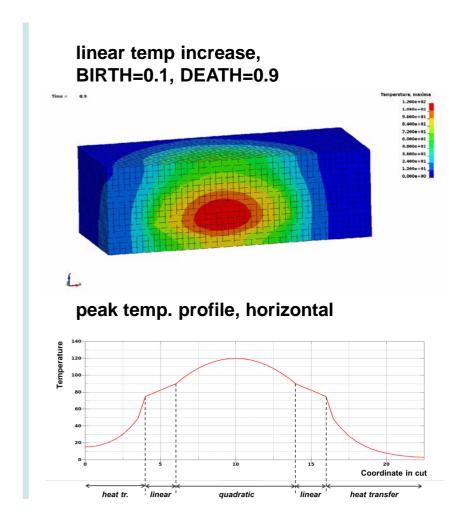




OPTION = 1

#### **Motivation**

- temperature in the weld nugget
  - prescribed at the center, boundary of nugget, and boundary of HAZ
  - quadratic approximation inside the nugget
  - linear approximation in the HAZ
- boundary condition active between BIRTH and DEATH times
- load curve input (LCIDT) for temperature scaling factor as function of normalized time
- To further increase the efficiency, customers want to imprint the temperature field in the weld nugget directly as thermal load in the **structure** solver

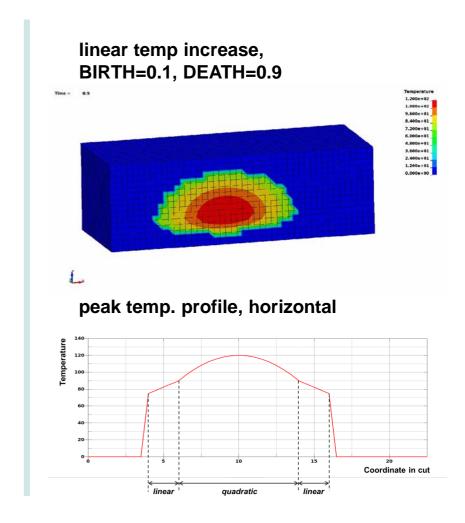






## \*LOAD\_THERMAL\_RSW

- Thermal loading condition for the structure solver, i.e. no thermal calculation required
- Temperature profile in the weld nugget same as in the temperature boundary condition
  - Prescribed at the center, boundary of nugget, and boundary of HAZ
  - Quadratic approximation inside the nugget
  - Linear approximation in the HAZ
- Outside the HAZ a default temperature is assumed







## **Summary and Outlook**

- Presented thermal-mechanical coupling schemes realized within LS-DYNA for assembly simulations
- Discussed new functionality \*LOAD\_THERMAL\_BINOUT for 1-way coupled simulation
  - Uses the flexible and open LSDA data format for data transfer
  - Mapping of temperatures based on user node ID requires only partial overlap of meshes
  - Time offset and scaling functionality for easy modification of weld ordering
- Future work
  - Provide a tailored LSDA output file to reduce memory consumption
  - Implement thermal loading conditions for further line welding process simulations in early design phases





# FIN

