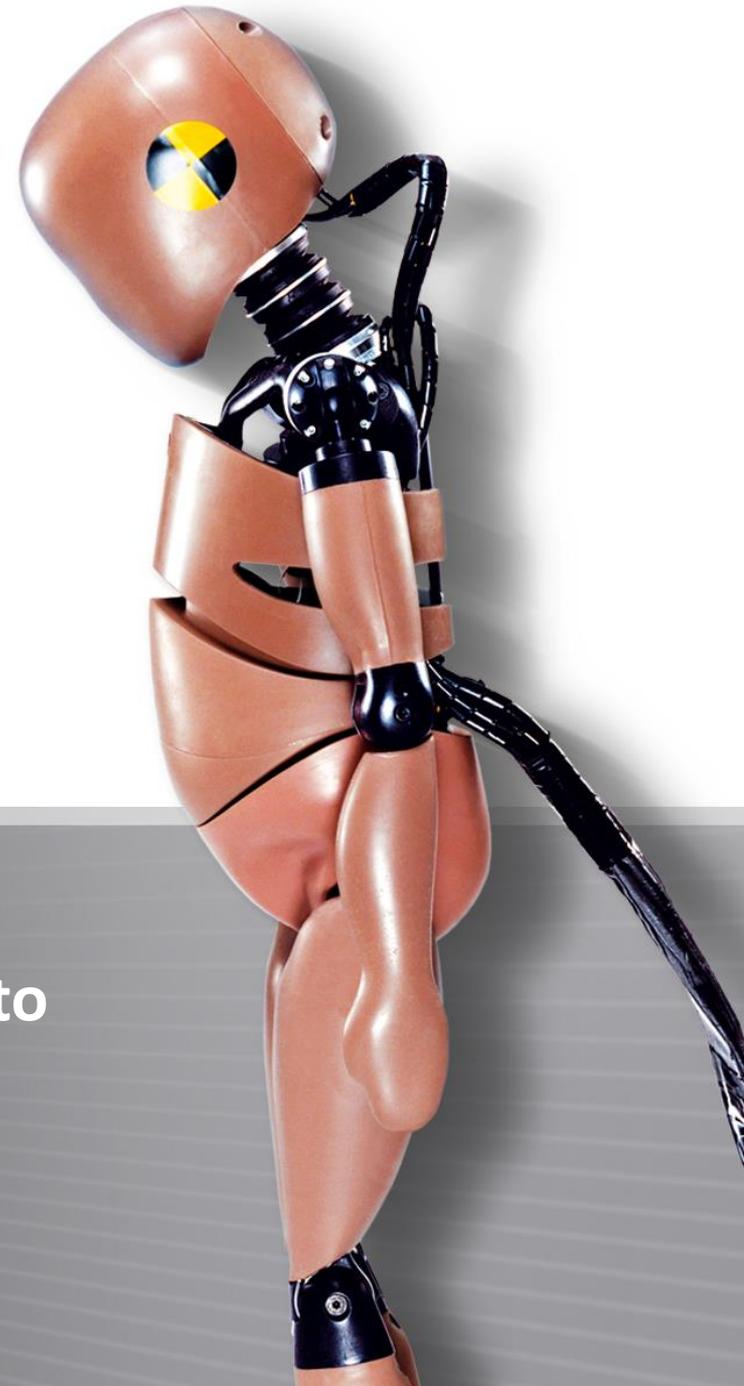




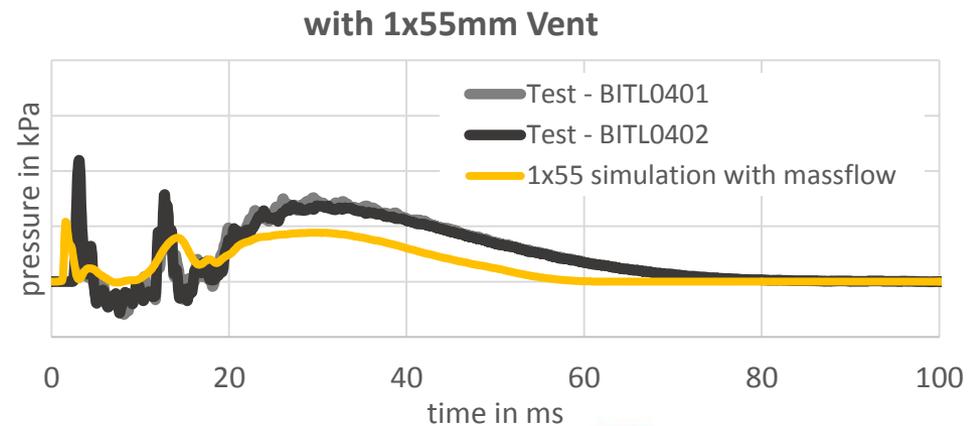
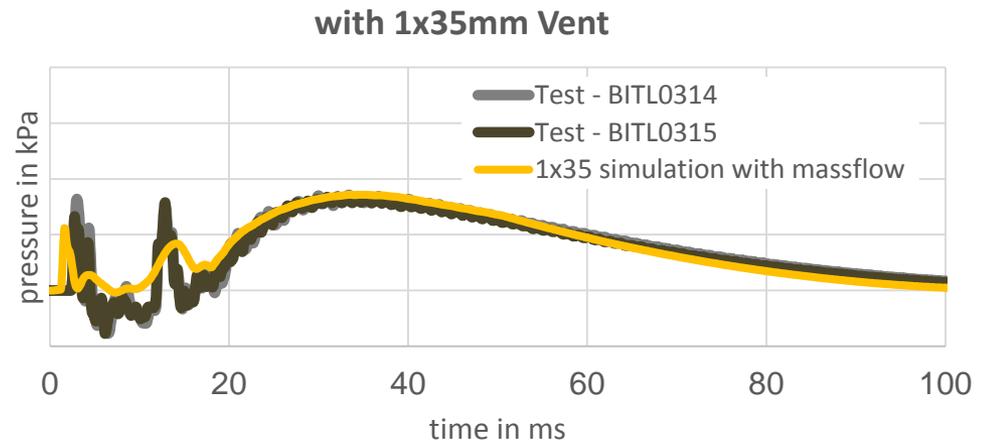
Simulation of coldgas inflators while taking the Joule – Thomson – Effect into account

Tilo Laufer - Takata AG Berlin, 10th October 2016



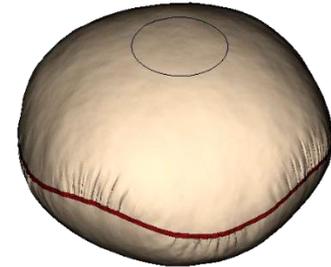
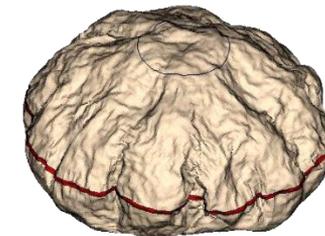
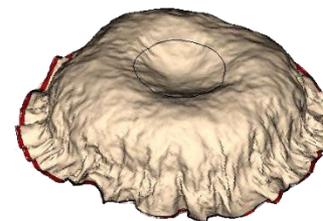
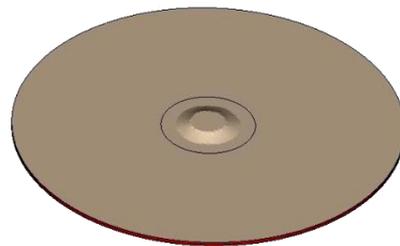
- problem definition
- conventional inflator model
- direct inflator simulation using the CP-Method
 - the Joule – Thomson – Effect
 - results of the tank test
 - final results in alternative system model
 - conclusion

- conventional inflator modelling:
 - Inflator mass-flow and temperature derived from tank test
- Takata's standardized inflation test



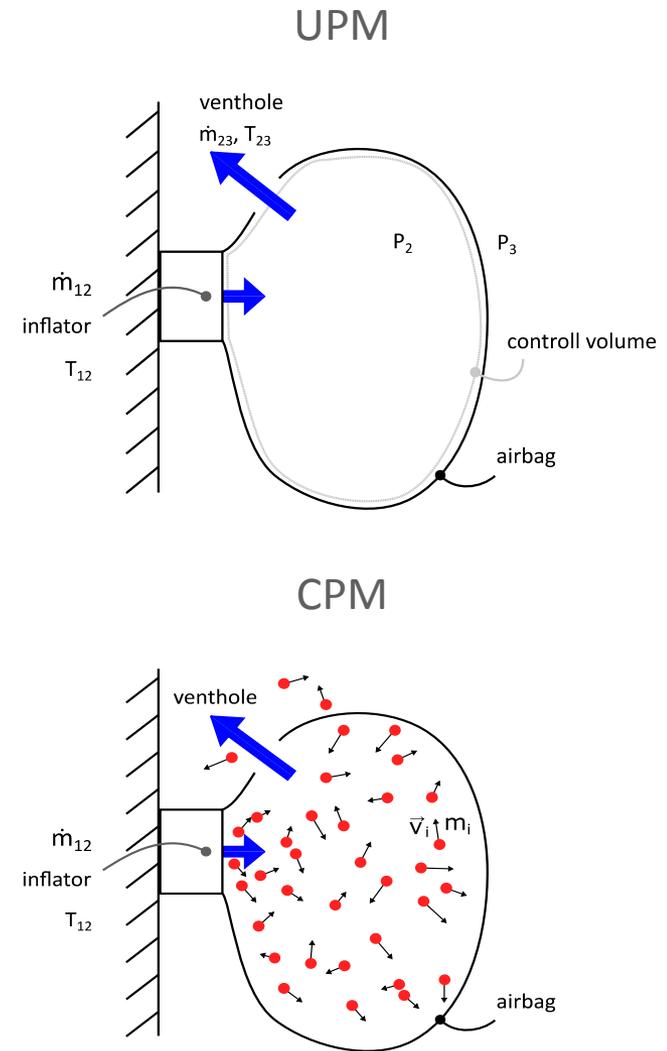
- Airbag simulation is a Fluid-Structure-Interaction (FSI) problem
- full FEA simulation coupled with CFD simulation is often too complex
 - Very long computing times
- Wang and Nefske [1] offered simplified approach to model the fluid behavior:

$$\frac{d}{dt} (mu)_{cv} = \sum \dot{m}_i h_i - \sum \dot{m}_o h_o - \dot{W}_{cv} - \dot{Q}_{cv}$$



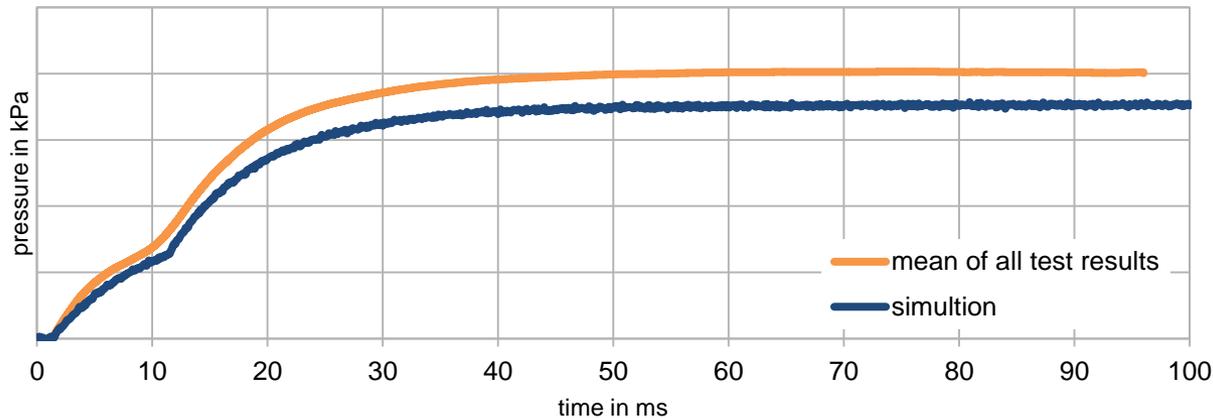
[1] Wang, J.T. and Nefske, D.J.: "A new cal3d airbag inflation model", SAE Technical Paper, 1988

- With the beginning of the 21st century NHTSA introduced the out of position requirement
- This lead to the need of a more accurate modeling of the fluid domain
 - LS-Dyna introduced the Corpuscular-Particle-Method (CPM)
 - With an acceptable increase of computation time the airbag deployment improved a lot
- But the CPM still needs a prescribed mass flow and temperature curve
 - By default derived from a tank test
 - The tank test brings by default a lot of assumptions in to the model



- Idea:
 - Direct modeling of cold gas inflators
 - Prefill the inflator with the correct mass, at the right pressure
 - Test the proper inflator output by simulating the tank test

pressure characteristic

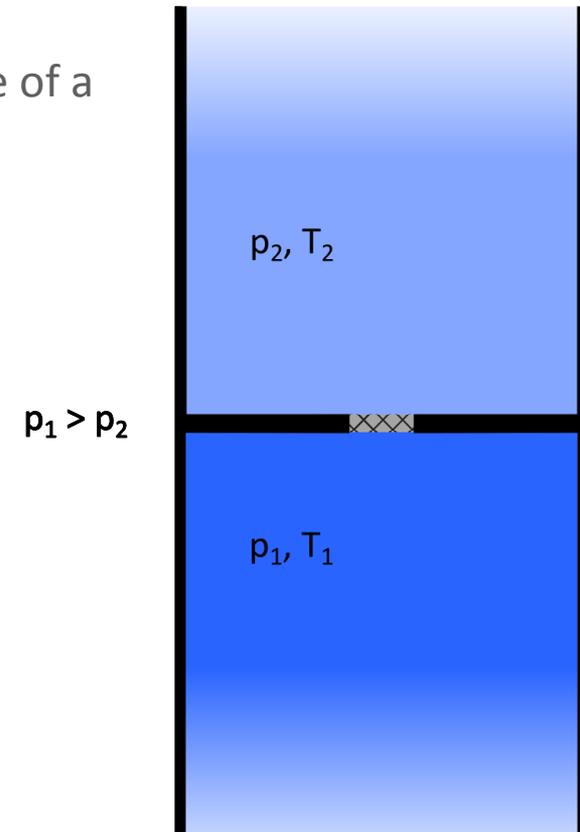


- The Joule-Thomson-Effect describes the temperature change of a fluid during a throttling process

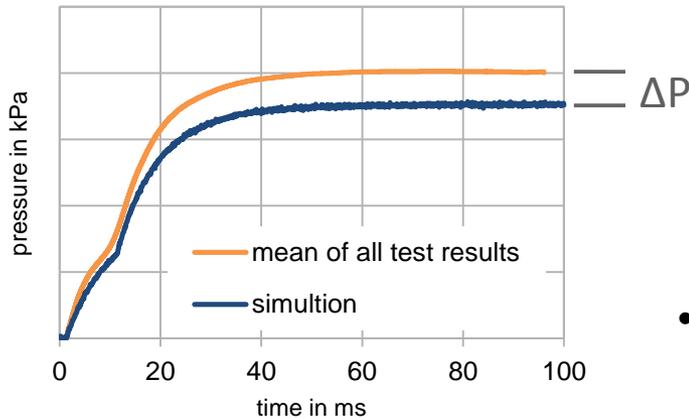
$$\mu_{JT} = \left(\frac{\partial T}{\partial p} \right)_H$$

- if $\mu_{JT} > 0$: the fluid will cool down at a throttling process
- if $\mu_{JT} < 0$: the fluid will heat up at a throttling process

$$\mu_{JT} = \mu_{JT}(p, T)$$



pressure characteristic



- Difference between test and simulation:
 - The inflator output in simulation is lower than in test
 - Reason: missing of Joule-Thomson-Effect
- JT-Effect describes the temperature change of a real gas or liquid during an adiabatic throttling process

$$\mu_{JT}(T) = \frac{b - 2a/RT}{C_p(T)}$$

$$= \frac{b - 2a/RT}{C_{p0} + C_{p1}T + C_{p2}T^2 + C_{p3}T^3}$$

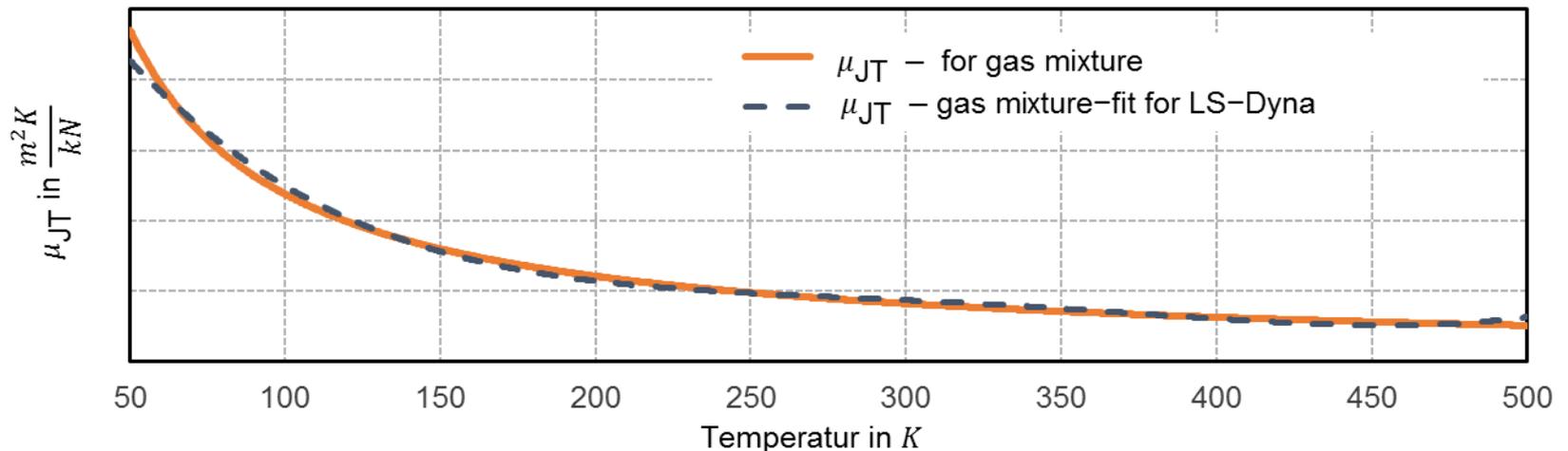
- Real gas (Van der Waals equation)
 - a - cohesion pressure
 - b - co-volume

[2] O. Babel.: "Joule-thomson-effekt", internet, 2014

- Calculate Joule-Thomson-coefficient:
 - Using the van-der-Waals coefficients from the table [2]
 - Approximate the Joule-Thomson-coefficient with polynomial of order 4

Name of gas	a in Nm ⁴ / kmol ²	b in m ³ /kmol
carbon dioxide	365585,65	0,0428
nitrogen	136777,05	0,0386
hydrogen	24645,79	0,0267
helium	3468,81	0,0238
air	135467,72	0,0365

Joule-Thomson-coefficient



- How to model the Joule-Thomson-Effect in LS-Dyna
 - Two new Keywords required
 - First (***DEFINE_CPM_VENT**) is referenced in Card 8 of the Airbag Particle keyword

```

*DEFINE_CPM_VENT
$:   label          c23      lctc23      lcpc23      enh_v      ppop      c23up
      1020&C23_VENT      1120
      1120
$:   jt            ids1  ->  ids2      iopt1
      2             1010      1012
    
```

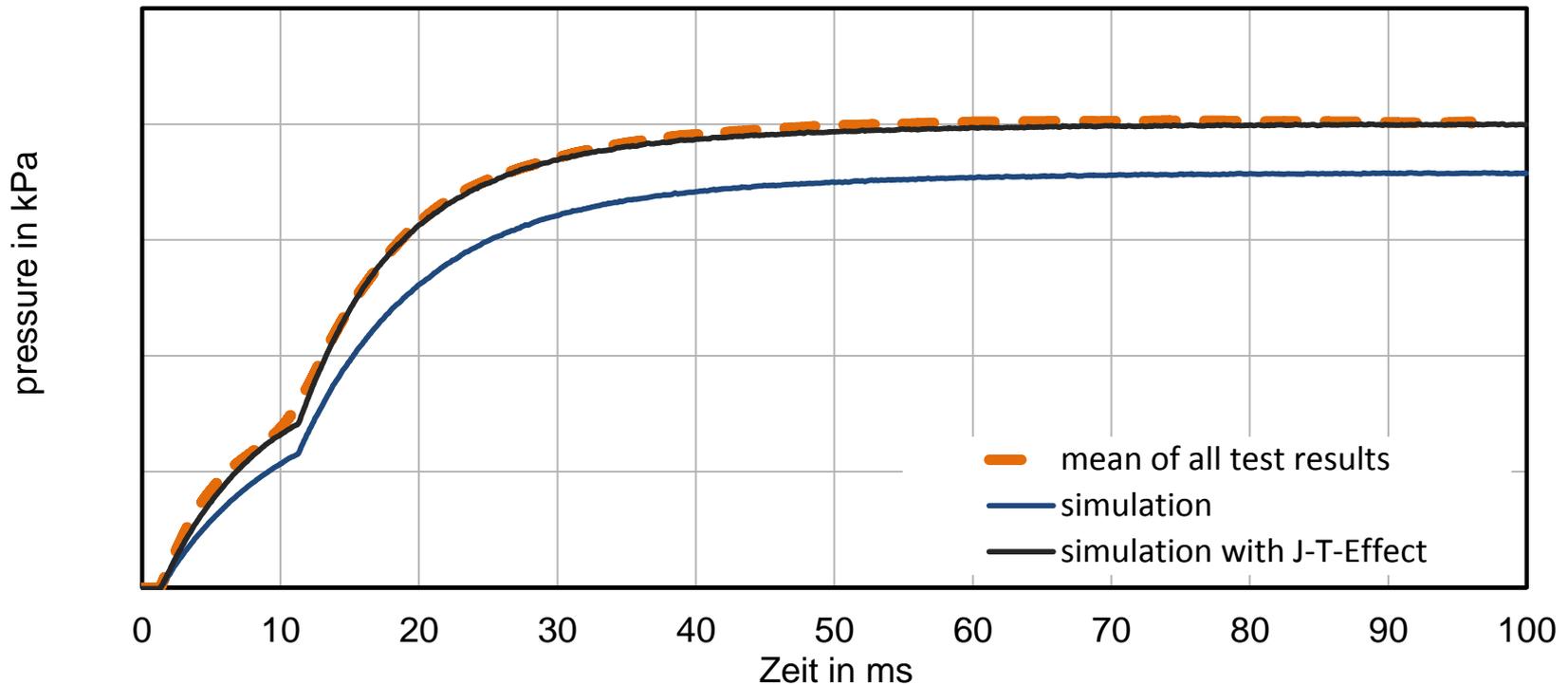
- Second (***DEFINE_CPM_GAS_PROPERTIES**) is referenced in Card 9 and 11 of the Airbag Particle keyword

```

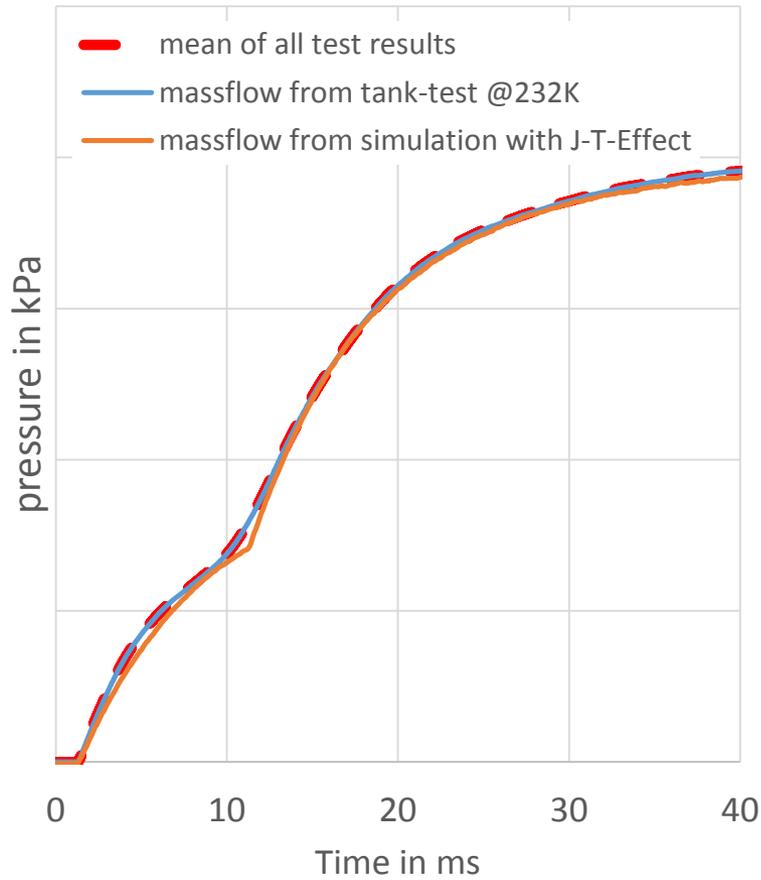
*DEFINE_CPM_GAS_PROPERTIES
$   ID           Xmm           Cp0           Cp1           Cp2           Cp3           Cp4
      1301 0.0040026      20.8           0             0             0             0
$   mu_t0        mu_t1        mu_t2        mu_t3        mu_t4        Chm_ID        Vini
      111.809 -13.3453 0.0617413-1.2798e-49.72665e-8
    
```

- Results of tank test simulation:
 - The Joule-Thomson-Effect closed the pressure gap
 - good correlation between test and simulation

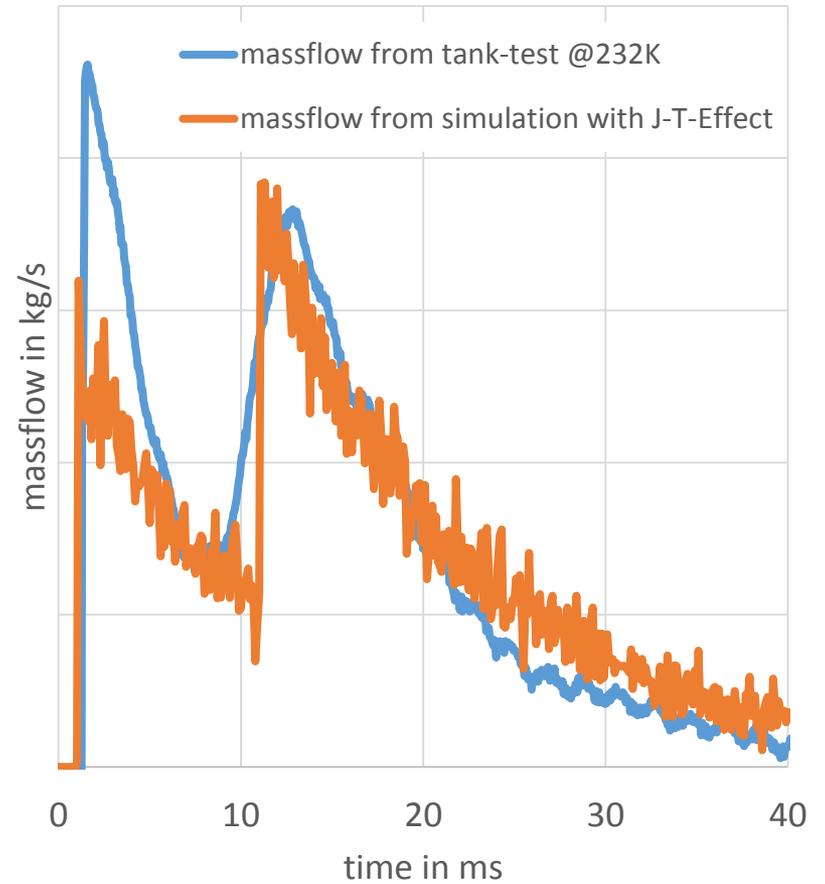
pressure characteristic



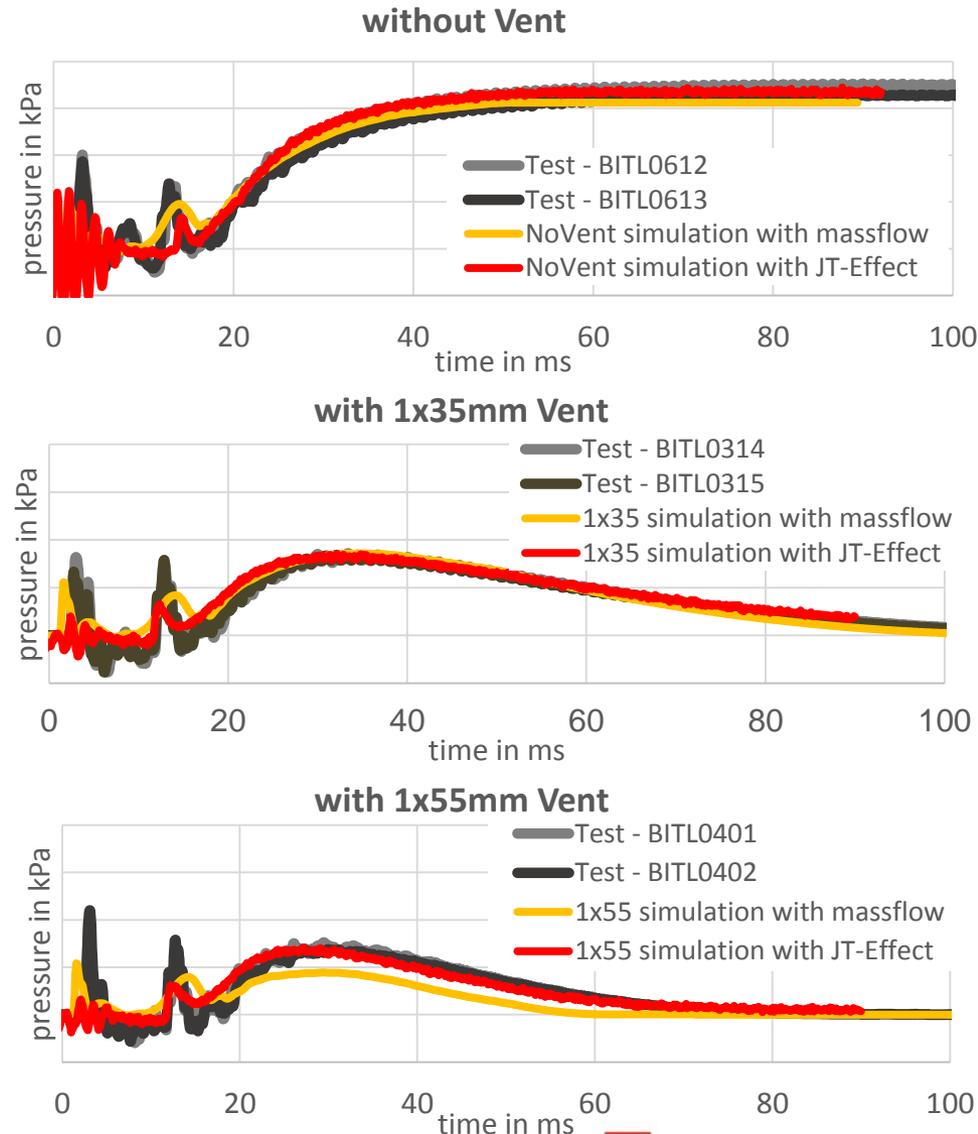
Tank-Test pressure comparison



massflow comparison



- Using the direct cold gas inflator modeling to simulate various system load-cases
 - System load-cases are represented by Takata's standardized inflation test with various vent sizes



- With LS-Dyna CP-Method it is possible to simulate cold-gas-inflators without tank-test assumptions
- The treatment of the Joule-Thomson-Effect is possible for more accurate results
- Open points
 - compare LS-Dyna Versions, currently:
`mpp.s.R7.1.2.95028.95028_dmp_sp`
 - Vary number of CPM-particles, currently: 100.000 particle
 - Analyze pressure dependency in Joule-Thomson-Coefficients, currently: $\mu_{JT} = \mu_{JT}(T)$



Thank you for your attention

Tilo Laufer

Takata AG, Safety Systems Numerical Simulation

tilo.laufer@eu.takata.com

030-47407-4237



This presentation and the information included therein was compiled with the greatest care possible. This presentation serves as general information and does not contain any offer, acceptance, or contract of any kind. The information contained in this presentation can only become contractually binding when it is included in a written contract by or with TAKATA or its affiliates.

The information contained in this presentation is confidential and is the sole property of TAKATA or its affiliates. Any use, divulgement, publication, or reproduction of any kind of any of the information contained in this presentation, whether in whole or part, requires prior written approval from TAKATA or its affiliates.