

Material Models of Polymers for Crash Simulation

An overview with focus on the dynamic test setup
Impetus by 4a engineering

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Laboratory of Mechanics - Equipment



- Hardware / Software
 - Clusters of Xeon, Intel Dual-Core and Quad-Core, 8CPUs parallel
 - FE Packages: LS-Dyna, Radioss, Nastran
 - Pre and Postprocessor: Hyperworks, LS-PrePost

- Experimental Setups
 - Quasi-static tensile and compression tester by Instron
 - Dynamic testing system “4a Impetus II”, movable devices for compression and bending tests, range of velocities: 500-4500mm/s
 - Dynamic test setup for impact tests on windshields
 - Drop tower



► Outline



- Parameter based Input vs. Tabulated Input
- Rubberlike Materials
 - Finite Elasticity
 - Blatz-Ko Rubber (Mat_7)
 - Simplified Rubber (Mat_181)
- Foams
 - Fu Chang Foam (Mat_83)
 - Simplified Rubber (Mat_181)
- Plastics
 - Piecewise Linear Plasticity Mat_24
 - Schmachtenberg / Johnson Cook



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Parameter based Input versus Tabulated Input



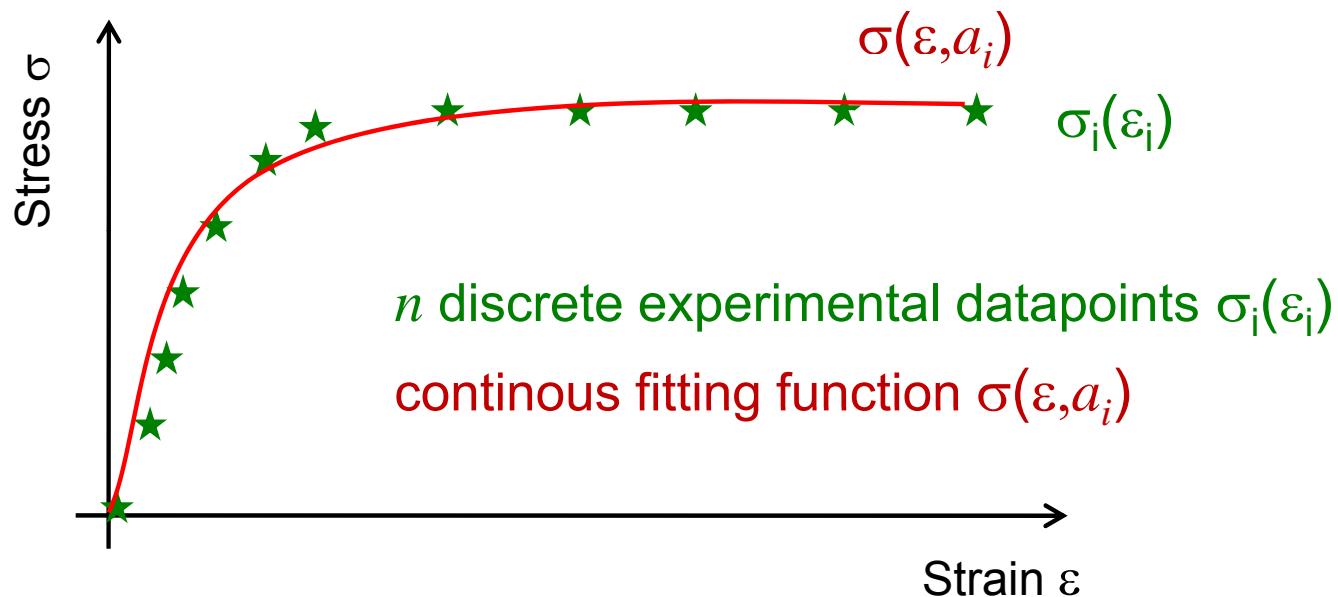
- Input of stress-strain relations in a tabulated way are very popular in commercial crash-codes
- The (more or less) direct input of experimental data obtained by tensile tests is the major benefit of those approaches
- This advantage fails in the validation and verification process where the stress-strain-curves have to be fitted to experimental results
- Parameter based stress-strain relations have therefore a huge advantage in reverse engineering (fitting of parameters, e.g. by LS-OPT, instead of the entire stress-strain-datapoints)



Parameter based Input versus Tabulated Input



- Parametrized Formulation



- Usually via suitable ansatz $\sigma(\varepsilon, a_i)$ in dependence of the material under consideration, where a_i are material parameters



Parameter based Input versus Tabulated Input



- Parameters may then be identified, e.g. by least square fit:

$$S(a_i) := \sum_{k=1}^n [\sigma_k(\varepsilon_k) - \sigma(\varepsilon, a_i)]^2 \rightarrow MIN$$

$$\Rightarrow \partial_{a_1} S(a_1) = \partial_{a_2} S(a_2) = \dots = \partial_{a_n} S(a_n) = 0$$

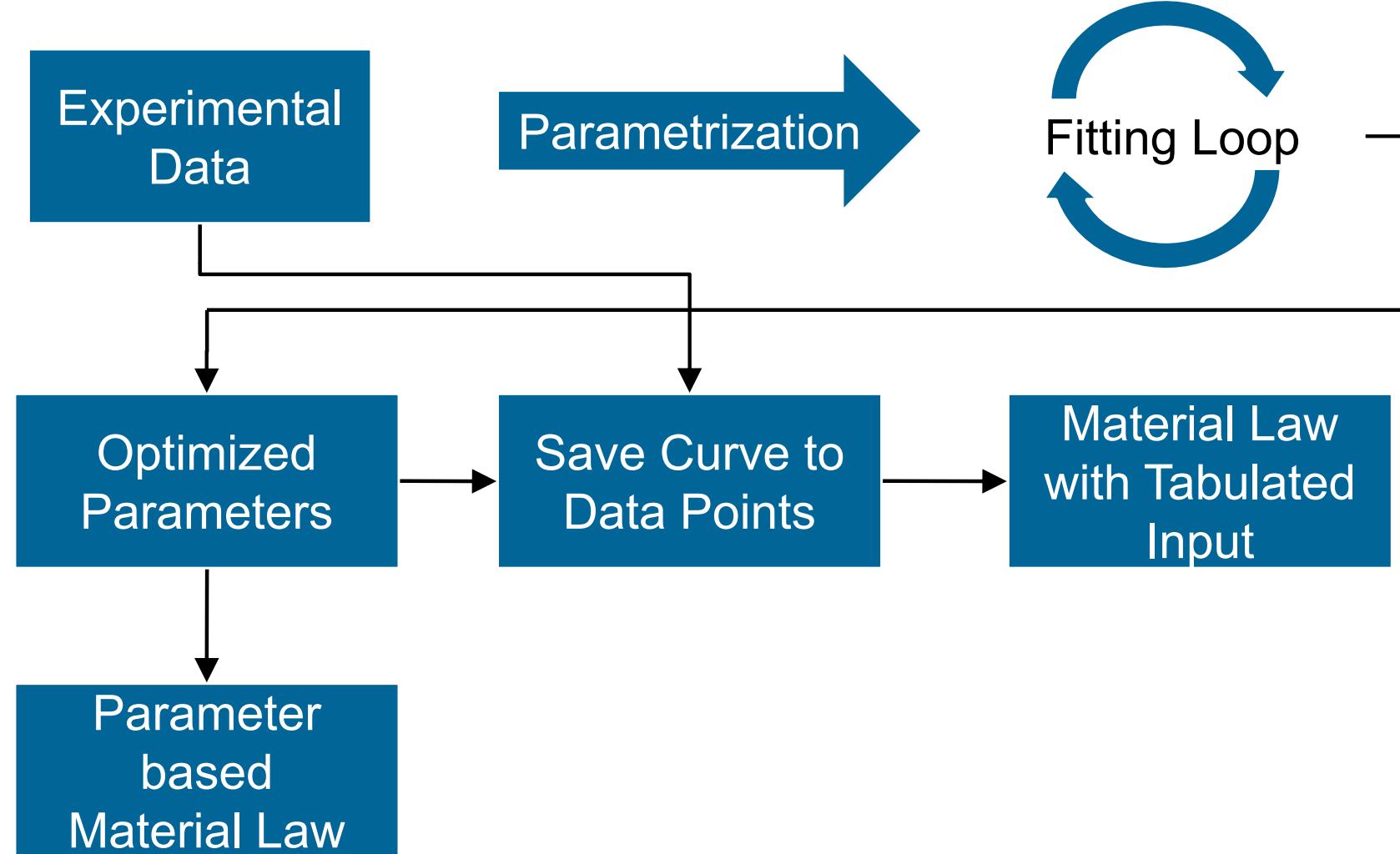
which leads to a nonlinear system of equations in general

- Alternatively LS-OPT can also be used



Parameter based Input versus Tabulated Input

- Procedures of Material Card Generation



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Hyperelasticity



Right Cauchy-Green Tensor $\mathbf{C} = \mathbf{F}^T \mathbf{F}$ with $\mathbf{F} = \text{Grad } \mathbf{x}$

$$\text{2. PK: } \mathbf{S} = 2 \frac{\partial W}{\partial \mathbf{C}} \xrightarrow{\text{Cauchy stress tensor}} \sigma = \frac{1}{J} \mathbf{F} \mathbf{S} \mathbf{F}^T \quad J = \det \mathbf{F}$$

Strain energy density in terms of invariants: $W = \hat{W}(I_C, II_C, III_C)$

$$I_C = \mathbf{1} : \mathbf{C} = \text{tr } \mathbf{C}, \quad II_C = \frac{1}{2} (I_C^2 - \mathbf{C} : \mathbf{C}), \quad III_C = \det \mathbf{C}$$

Derivative:

$$\mathbf{S} = 2 \frac{\partial W}{\partial I_C} \mathbf{1} + 2 \frac{\partial W}{\partial II_C} (I_C \mathbf{1} - \mathbf{C}) + 2 \frac{\partial W}{\partial III_C} III_C \mathbf{C}^{-1}$$



Rubber Laws in LS-DYNA



Law	
7	MAT_BLATZ-KO_RUBBER
27	MAT_MOONEY-RIVLIN_RUBBER
31	MAT_FRAZER-NASH_RUBBER
77	MAT_GENERALIZED_RUBBER
77	MAT_OGDEN_RUBBER
181	MAT_SIMPLIFIED_RUBBER



One-Parameter Law: Blatz-Ko Energy Function



- General form for polyurethane foam rubbers (1962):

$$W = \frac{G}{2} \left[I_1 + \frac{1}{\alpha} (I_3^{-\alpha} - 1) - 3 \right] + \frac{G}{2} (1 - \beta) \left[\frac{I_2}{I_3} + \frac{1}{\alpha} (I_3^\alpha - 1) - 3 \right]$$

$$\alpha = \frac{\nu}{1 - 2\nu}$$

- Implemented as material law no. 7 in LS-DYNA:

$$\beta = 1, \quad \nu = 0.463$$

$$W = \frac{G}{2} \left[I_1 - 3 + \frac{1}{\alpha} (I_3^{-\alpha} - 1) \right] \quad \Rightarrow \quad \sigma = G \left(\frac{1}{J} \mathbf{F} \mathbf{F}^T - J^{-2\alpha-1} \boldsymbol{\delta} \right)$$

$$\alpha = \frac{\nu}{1 - 2\nu} \Rightarrow -2\alpha - 1 = -2 \frac{\nu}{1 - 2\nu} - 1 = \frac{-1}{1 - 2\nu}$$



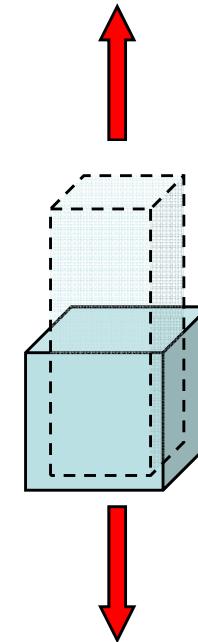
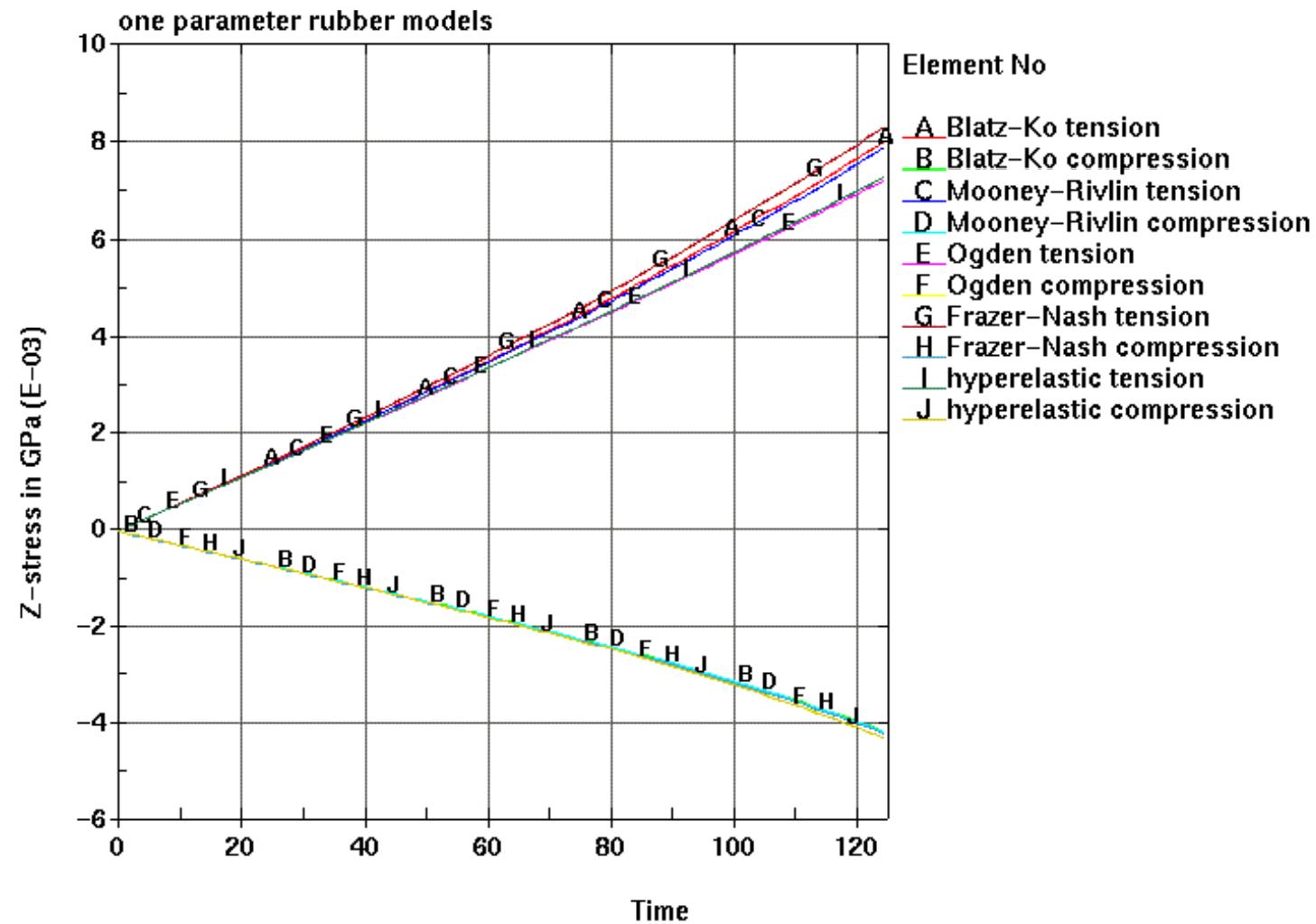
Equivalent One-Parameter Models



7	27	77 Ogden	31	77
G	$A = \frac{G}{2}$	$\mu_1 = G$ $\alpha_1 = 2$	$C100 = \frac{G}{2}$	$C10 = \frac{G}{2}$



Equivalent One-Parameter Models





Limitations of Low Order Models



- Fitting of a higher curvature in the stress-strain curve for large deformations will not work
- Optimization software will not help
- Multiple parameter models, e.g. Ogden's energy function (Mat_77) allow for fitting stress-strain curves with higher curvature

$$W = \sum_{i=1}^3 \sum_{j=1}^n \frac{\mu_j}{\alpha_j} \left(\lambda_i^{*\alpha_j} - 1 \right) + K(J - 1 - \ln J)$$

$$J = \lambda_1 \lambda_2 \lambda_3, \quad \lambda_i^* = \lambda_i J^{-1/3} = \frac{\lambda_i}{J^{1/3}}$$

- Tabulated version available in MAT_SIMPLIFIED_RUBBER

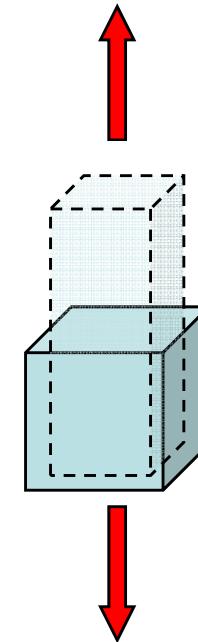
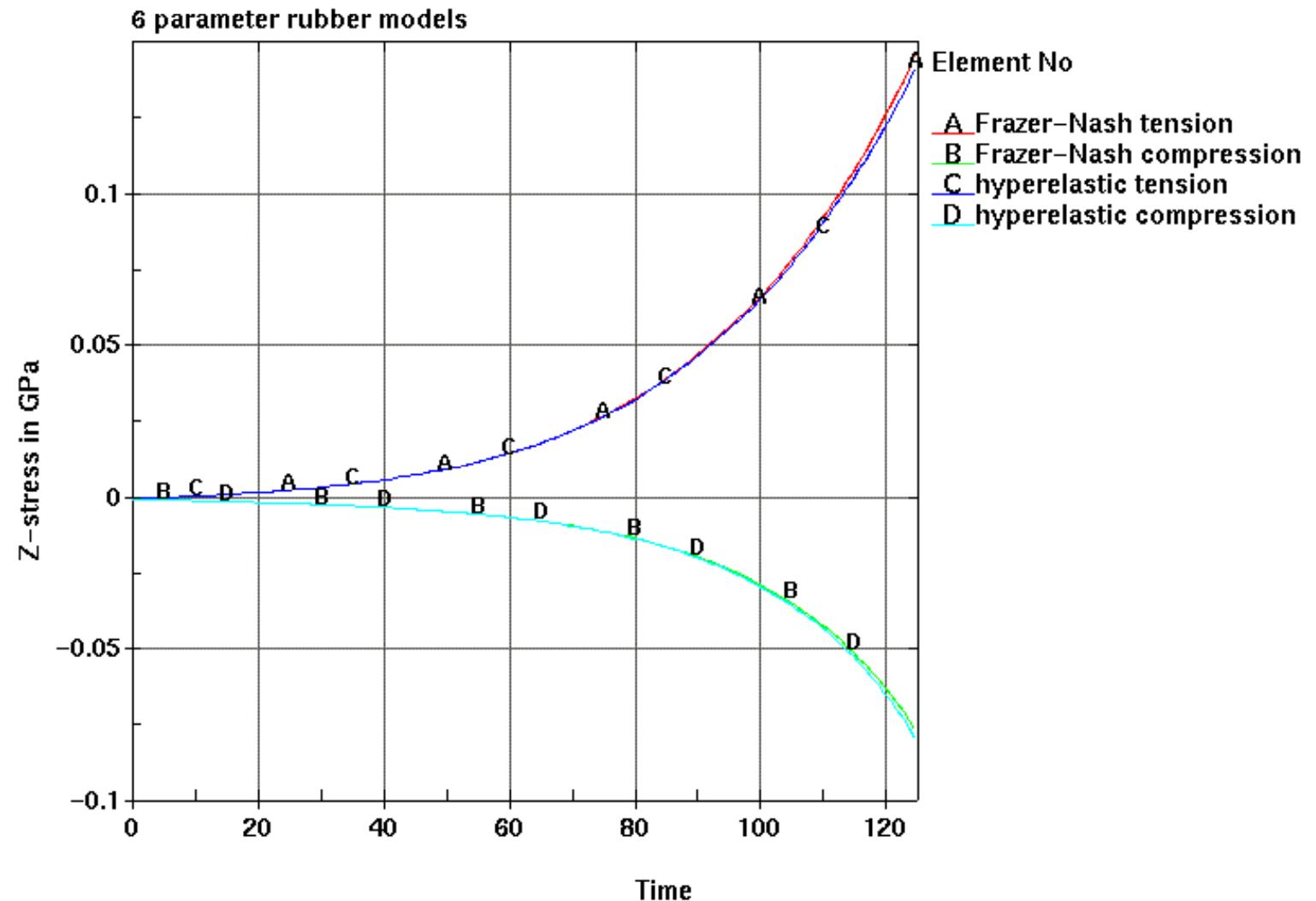


Equivalent Multiple-Parameter Models



31		77
C100	I_1	C10
C200	I_1^2	C20
C300	I_1^3	C30
C400	I_1^4	
C110	$I_1 I_2$	C11
C210	$I_1^2 I_2$	
C010	I_2	C01
C020	I_2^2	C02

Equivalent Multiple-Parameter Models



Outline



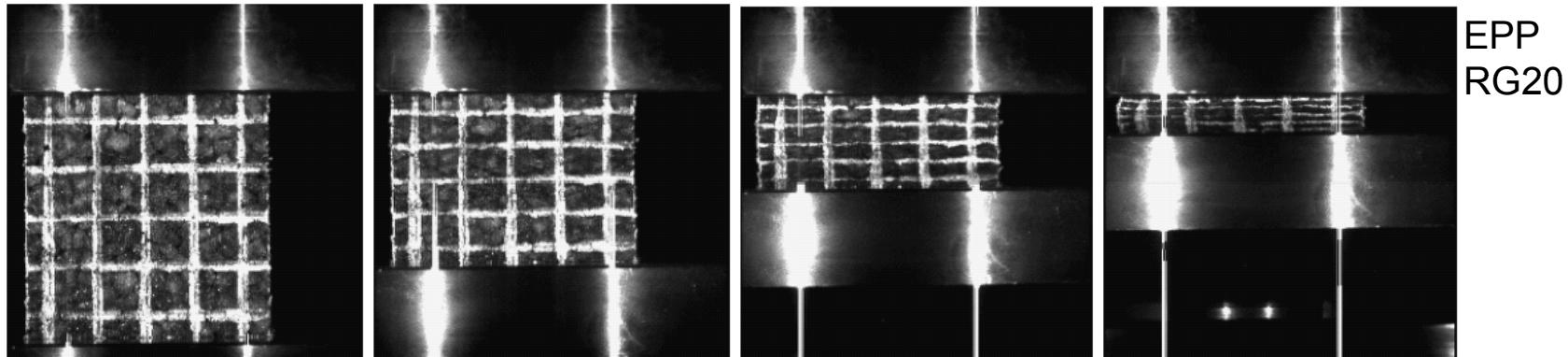
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➤ Introduction - What are Foams?



- Material scientist: any material manufactured by some expansion process
- (Crash-) Numericist: a material with Poisson's ratio close to zero



- Both definitions coincide only for low density foams, roughly below 200g/l
- High density (>200g/l) structural foams exhibit a non-negligible Poisson effect



Material Laws for Elastic Foams in LS-DYNA



No.	keyword	formulation	input
38	MAT_BLATZ_KO_FOAM	hyperel., $\nu = 0.25$	1 parameter
57	MAT_LOW_DENSITY_FOAM	hyperel. + viscoel.	LC + parameter
62	MAT_VISCOUS_FOAM	hyperel. + viscoel. ν variable	parameter
73	MAT_LOW_DENSITY_VISCOUS_FOAM	hyperel. + 6 viscoel. dampers	LC + parameter
83	MAT_FU-CHANG_FOAM	hyperel. + strain-rate	LC/ table
177	MAT_HILL_FOAM	hyperel., ν variable	LC
178	MAT_VISCOELASTIC_HILL_FOAM	= 177 + viscoel	LC + parameter
179	MAT_LOW_DENSITY_SYNTHETIC_FOAM	hyperel. pseudo-damage	LC LC
180	MAT_LOW_DENSITY_SYNTHETIC_FOAM_ORTHO	no damage orthogonal load direction	LC
181	MAT_SIMPLIFIED_RUBBER/FOAM	hyperel. + strain-rate	LC/ table
183	(WITH_FAILURE) / _WITH_DAMAGE	ν variable	

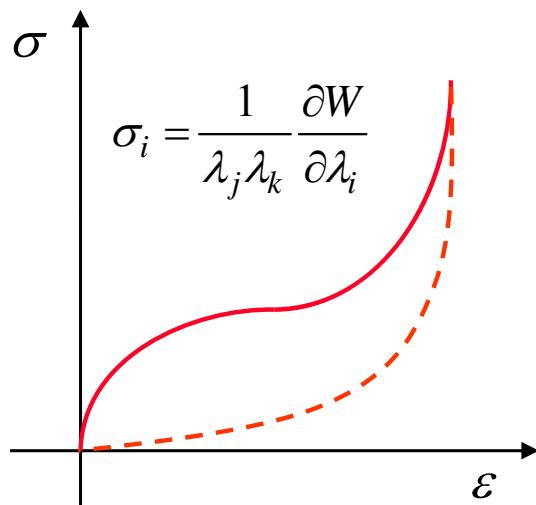


Material Laws for Elastic Foams (no Poisson Effect)



strainrate dependent hyperelastic

visco-hyperelastic

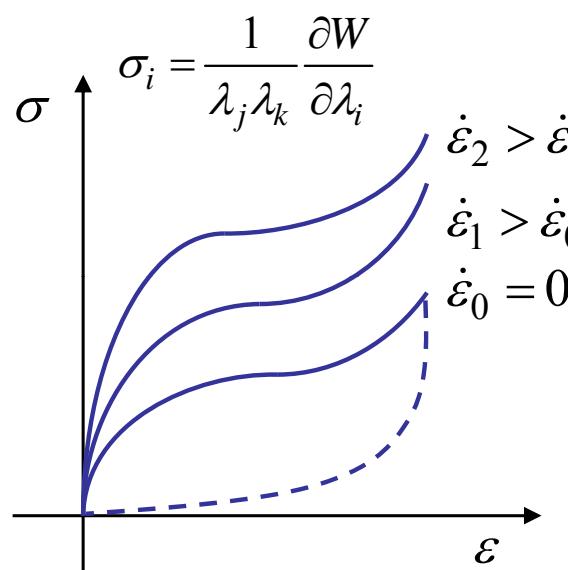


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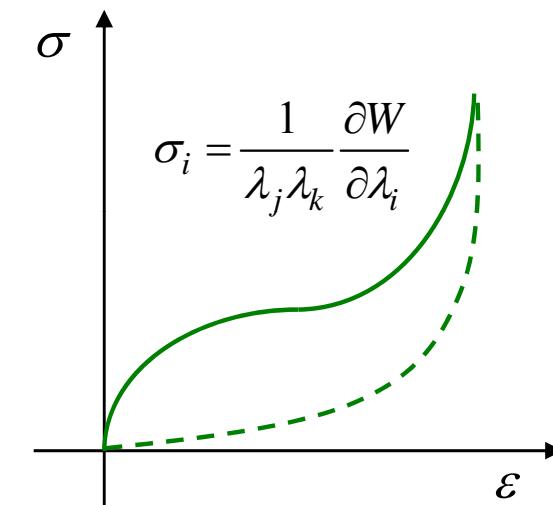
elastic damage



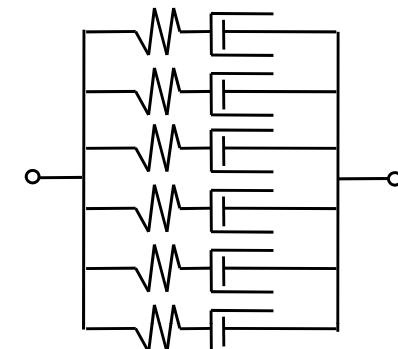
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elastic damage

visco-hyperelastic



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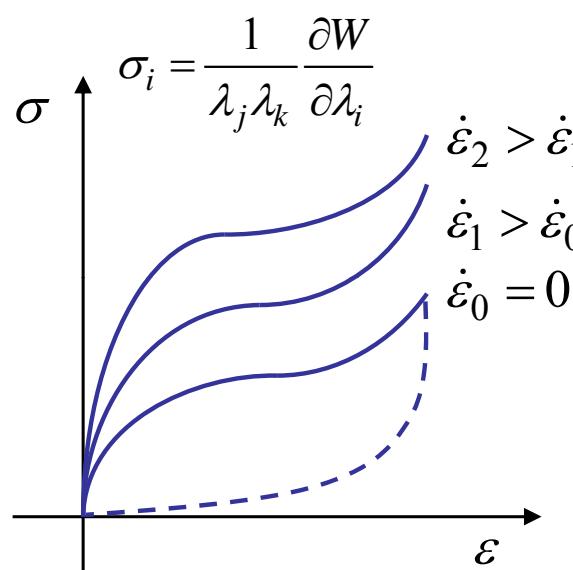
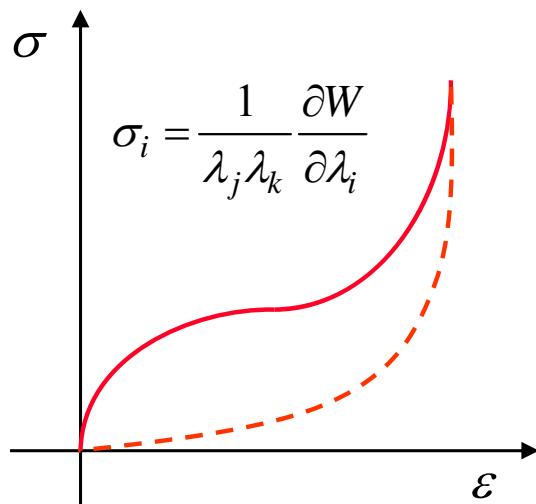


Material Laws for Elastic Foams (no Poisson Effect)

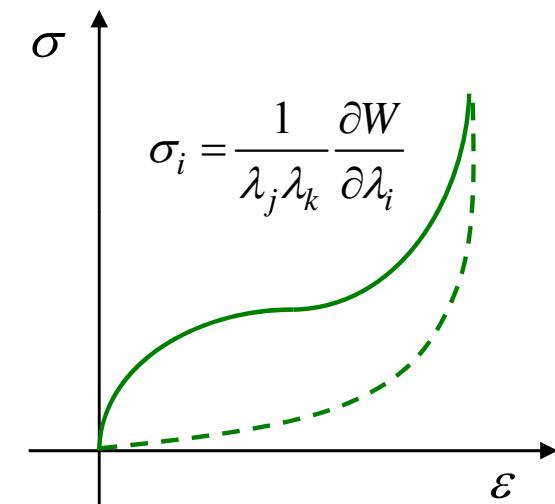


strainrate dependent hyperelastic

visco-hyperelastic



visco-hyperelastic



MAT_83
MAT_FU-CHANG_FOAM

MAT_57
MAT_LOW_DENSITY_FOAM

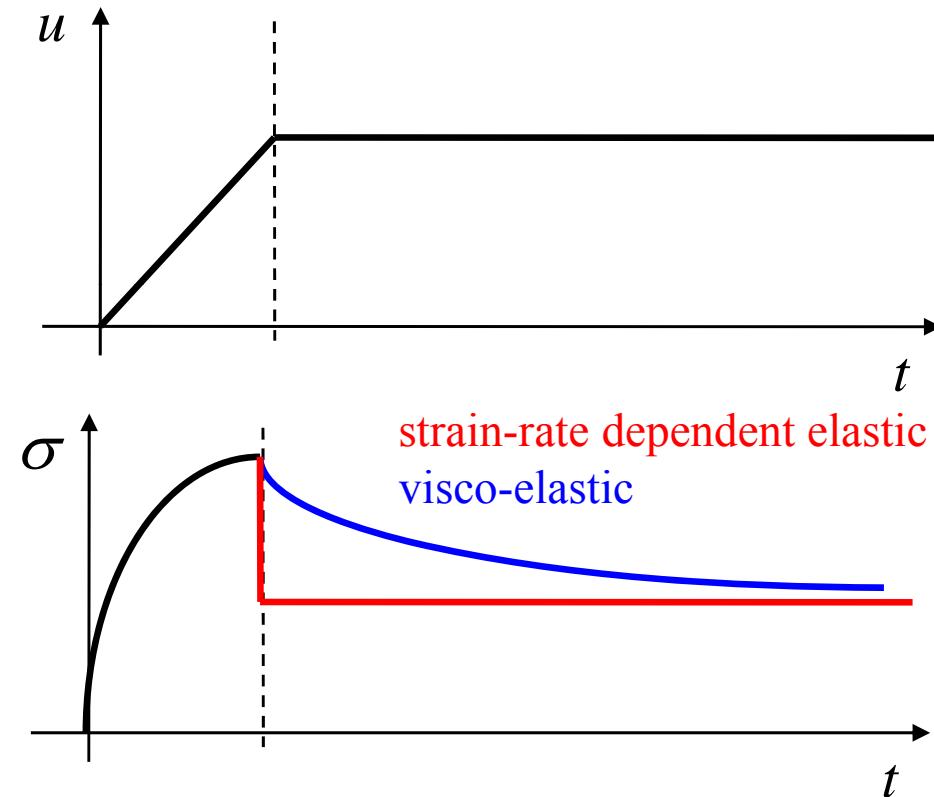
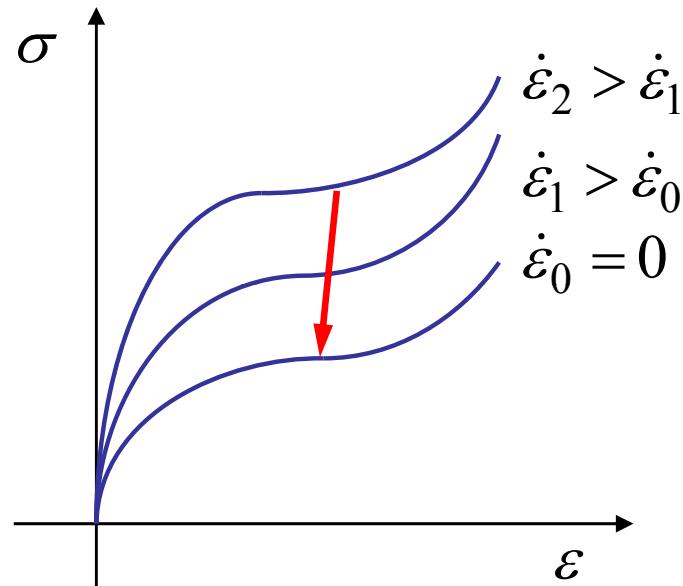
MAT_62
MAT_LOW_DENSITY_FOAM



Rate-Dependent Hyperelasticity versus Visco-Elasticity



Relaxation Test



- rate-independent unloading
- numerically stable
- unrealistic, potential problem for foams with high damping
(e.g. confor foam)

MAT_83
MAT_73



Complete Input-Data: Unloading



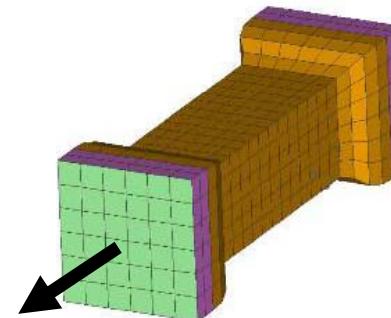
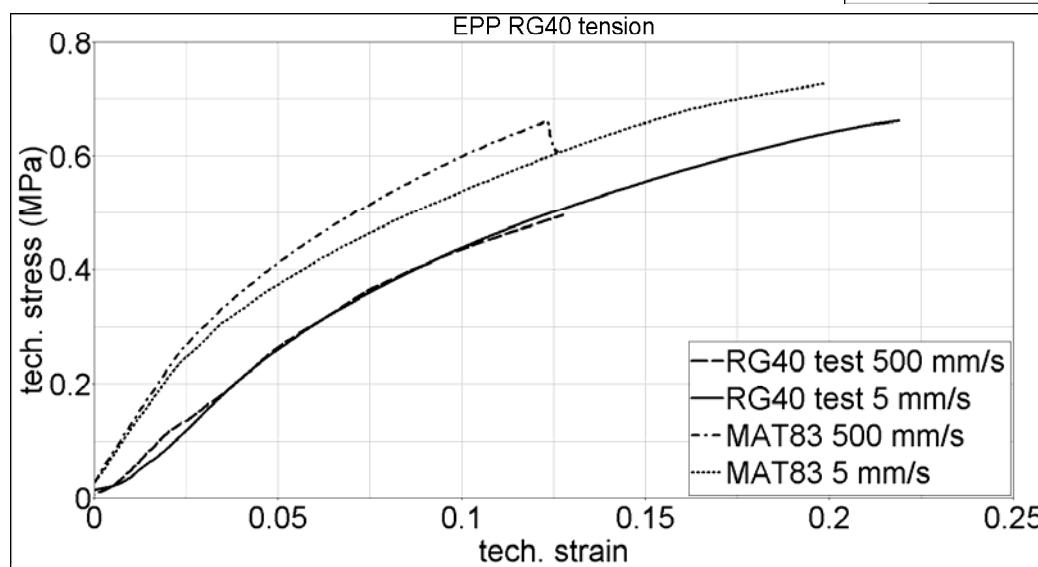
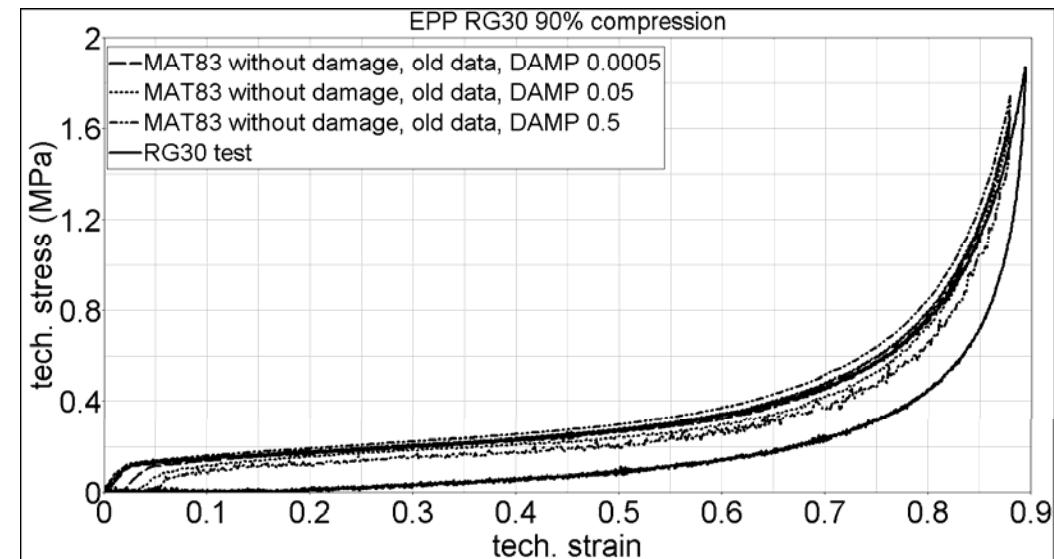
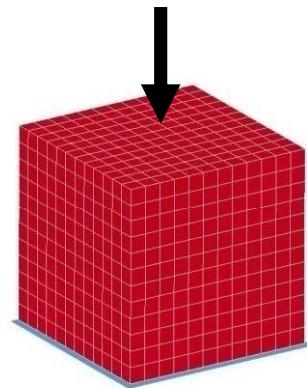
- Define an additional curve for unloading (strain rate zero in TABLE), this should correspond to the quasistatic unloading path
- Unloading always follows the curve with lowest strain rate and is detected by

$$\varepsilon_i \cdot \dot{\varepsilon}_i \begin{cases} \leq 0 & \rightarrow \text{unloading: strain rate is set to zero} \\ > 0 & \rightarrow \text{loading: strain rate dependence} \end{cases}$$

- This may lead to numerical problems that can be avoided by an elastic damage formulation
- Furthermore, no rate dependency upon unloading

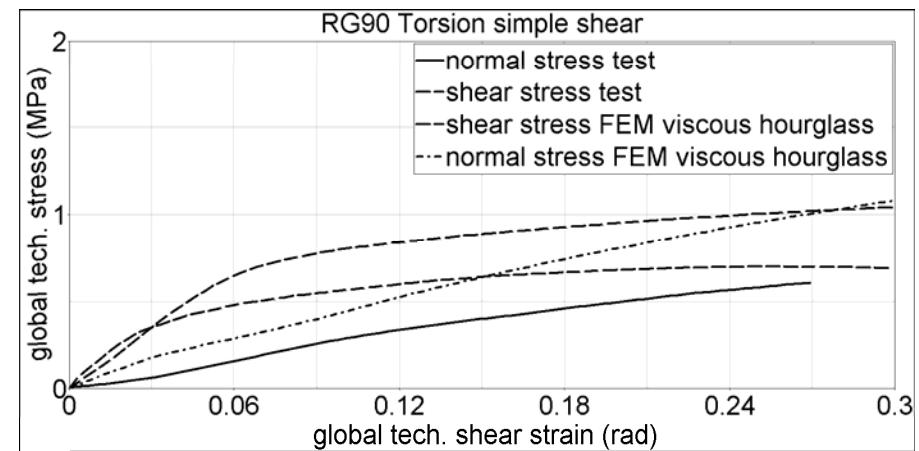
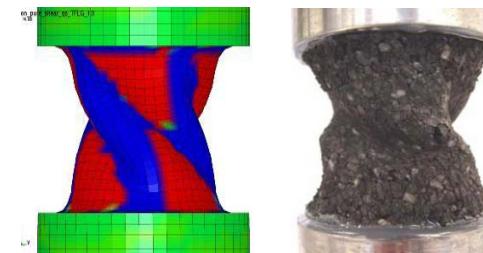
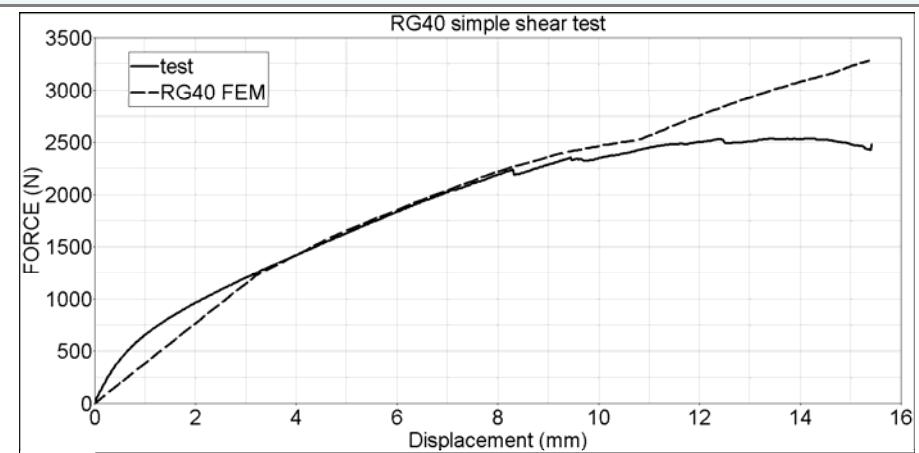
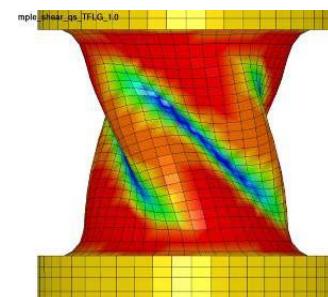
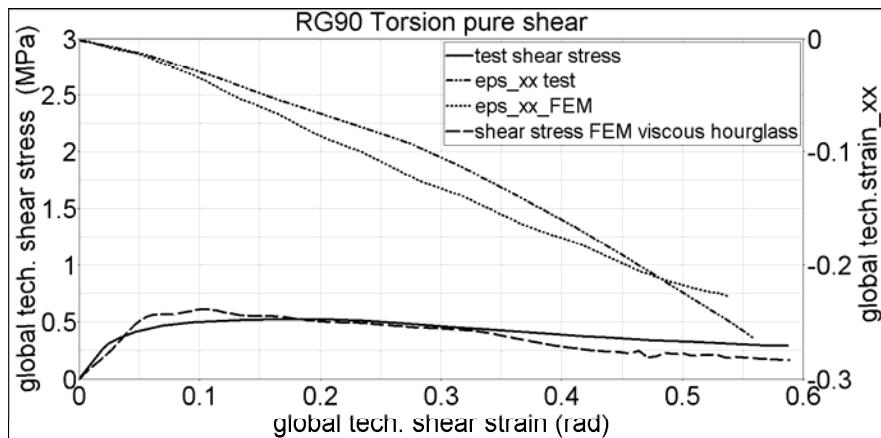
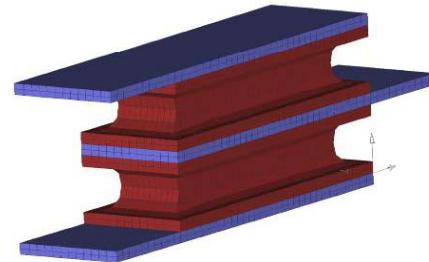


Some Validation Tests – How Accurate is MAT_83?





Some Validation Tests – How Accurate is MAT_83?

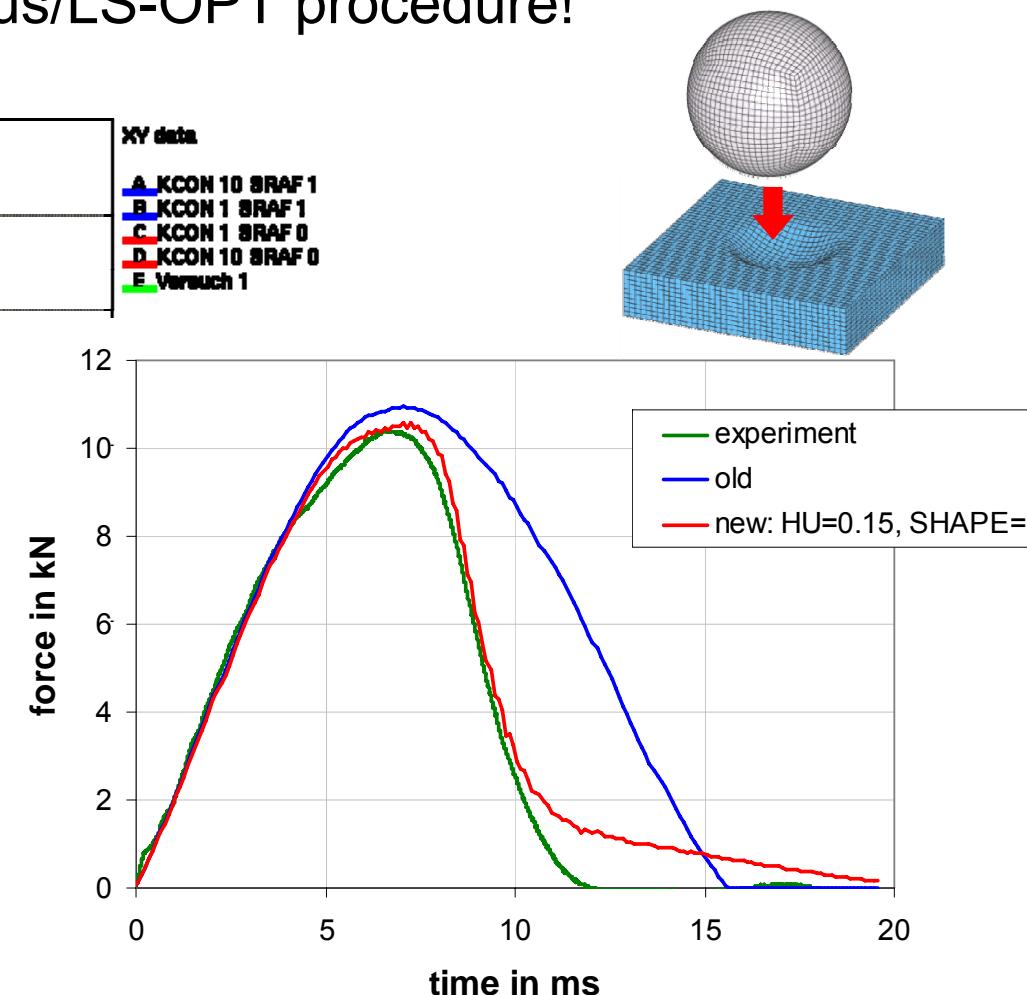
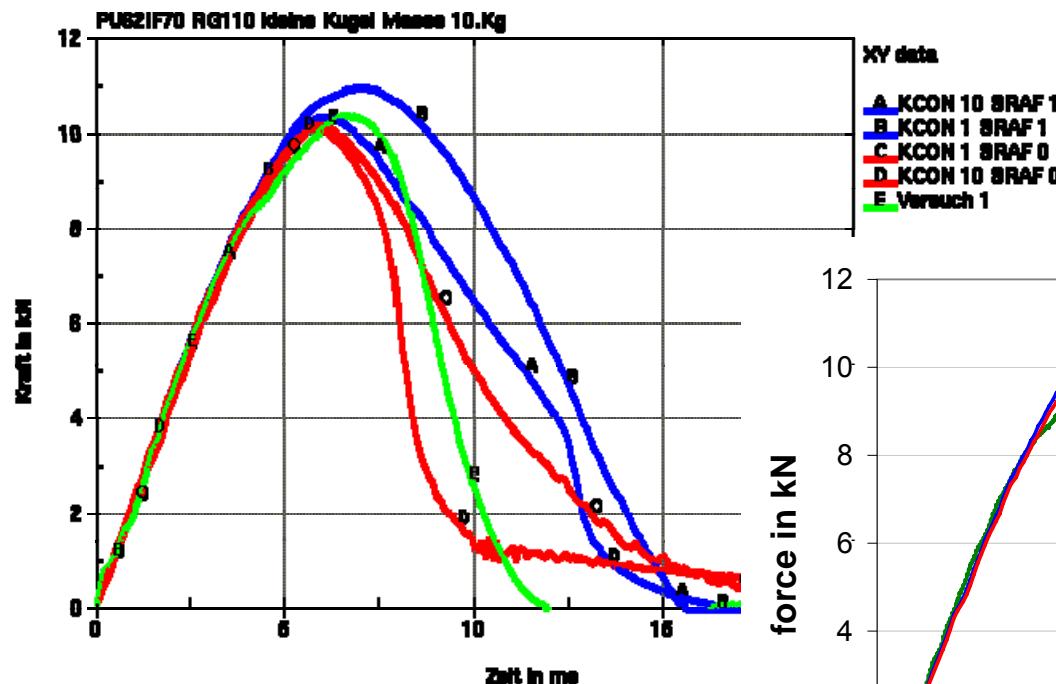




Some Validation Tests – How Accurate is MAT_83?



- Damage formulation a further improvement and can also be identified during the Impetus/LS-OPT procedure!





Material Law for Elastic Foams with Poisson Effect



- Uses Hill instead of Ogden functional (incompressible case, rubber):

$$W = \sum_{j=1}^m \frac{C_j}{b_j} \left[\lambda_1^{b_j} + \lambda_2^{b_j} + \lambda_3^{b_j} - 3 + \frac{1}{n} (J^{-nb_j} - 1) \right]$$

where C_j , b_j and n are material constants and $J = \lambda_1 \lambda_2 \lambda_3$

The nominal stresses (force per unit undeformed area) are

$$S_i = \frac{1}{\lambda_i} \sum_{j=1}^m C_j \left[\lambda_1^{b_j} - J^{-nb_j} \right] \quad i=1,2,3$$

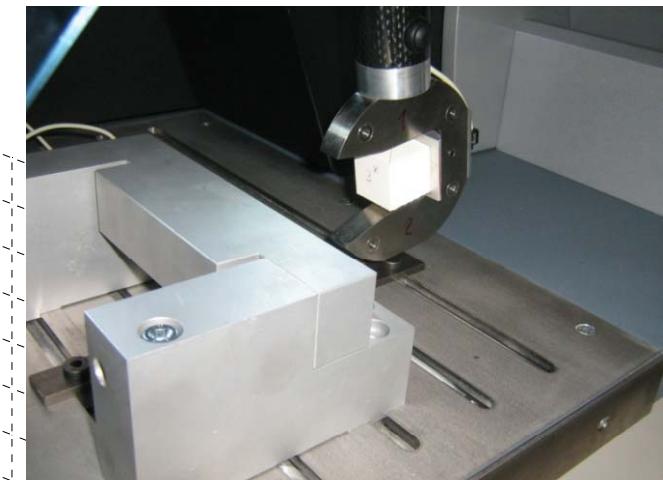
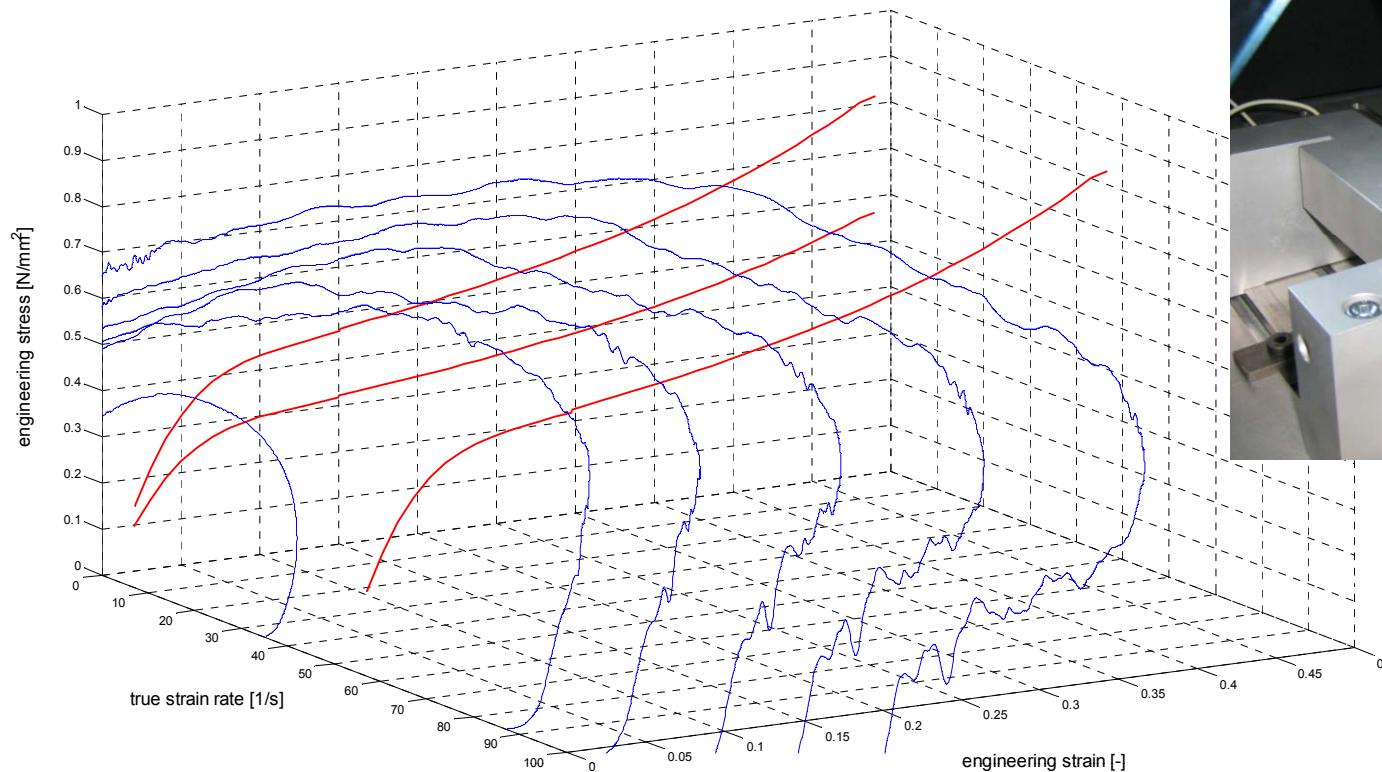
- Allows for a fully tabulated input implemented as
MAT_SIMPLIFIED_RUBBER/FOAM in 2004



Example: Rubberlike Foam for Sensomotoric Inlays



- In pendulum impact tests (Impetus) stress can be plotted as a function of strain and the strain rate: $\sigma = \sigma(\varepsilon, \dot{\varepsilon})$
- A fitted surface leads then to stress-strain relations for tabulated input
- Neuronal network in LS-OPT works similar

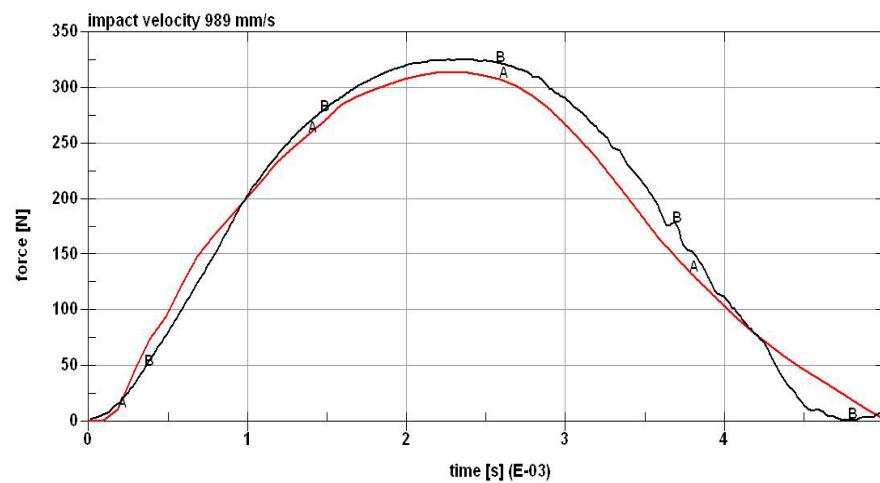
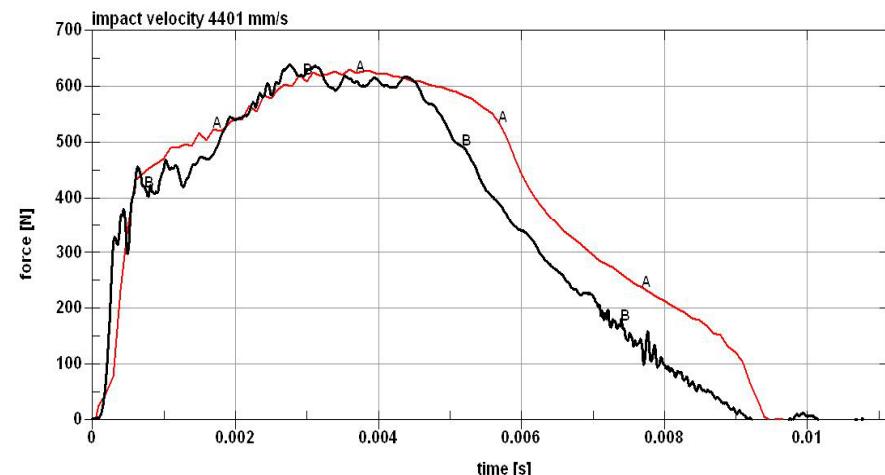
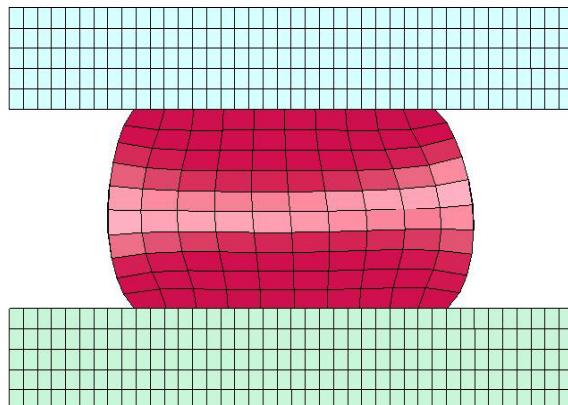
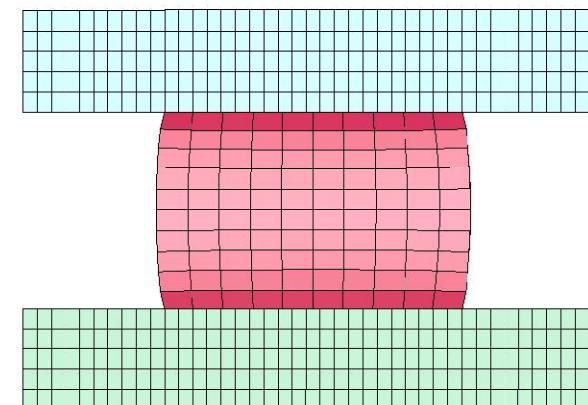




Validation: MAT181 vs. MAT183



- Hill's functional in MAT181 allows for a proper consideration of Poisson's ratio ($\nu=0.25$) and yields to a better agreement to the experiment

MAT 183, $\nu = 0.4998$ MAT 181, $\nu = 0.25$ 

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Material Models: Elasto-Plasticity



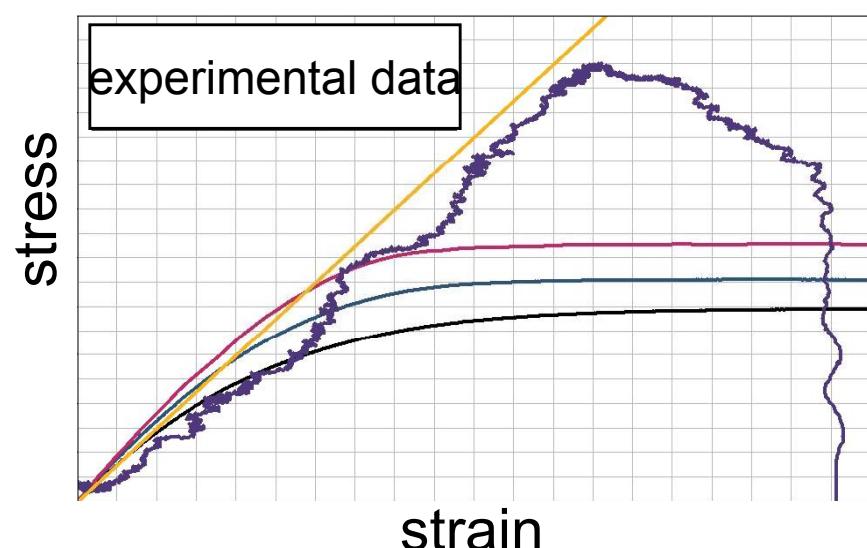
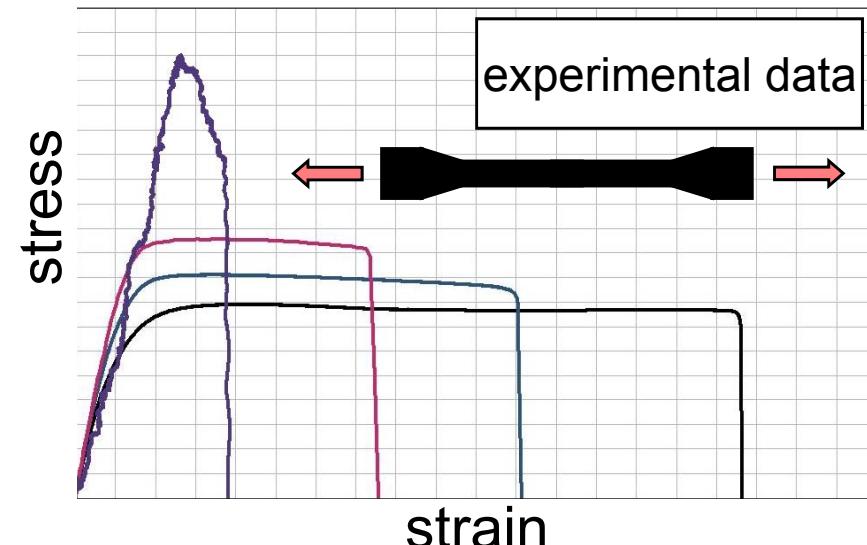
- Although thermoplastics do not show a strict transition from elasticity to plasticity, a elasto-(visco)plastic model is (so far) the best choice:
 - permanent deformation, implemented for shell elements
 - von Mises yield surface still standard for simulation of plastics
 - stable simulation; user-friendly input data (e.g. MAT24 in LS-DYNA)
 - High sophisticated models (SAMP, MF Polymers, ...) available now
- In what follows, the validation and verification process (e.g. reverse engineering) is demonstrated for
MAT_PIECEWISE_LINEAR_PLASTICITY (MAT_24)



V&V Step 1: Revision of the Test Data; Young's Modulus



- Test data has to be available as engineering stress vs. engineering strain (Excel / ASCII)
- Visual inspection of the data is necessary first. The goal is to obtain a single sufficiently smooth, i.e. non-oscillatory curve for each strain rate:
 - Eliminate strong oscillating curves
 - Scattering at the same strain rate ?
 - If yes: take the average of selected curves at the same strain rate, i.e. eliminate outliers
 - If no: take the average of all tests at the same strain rate
- Determine average Young's modulus





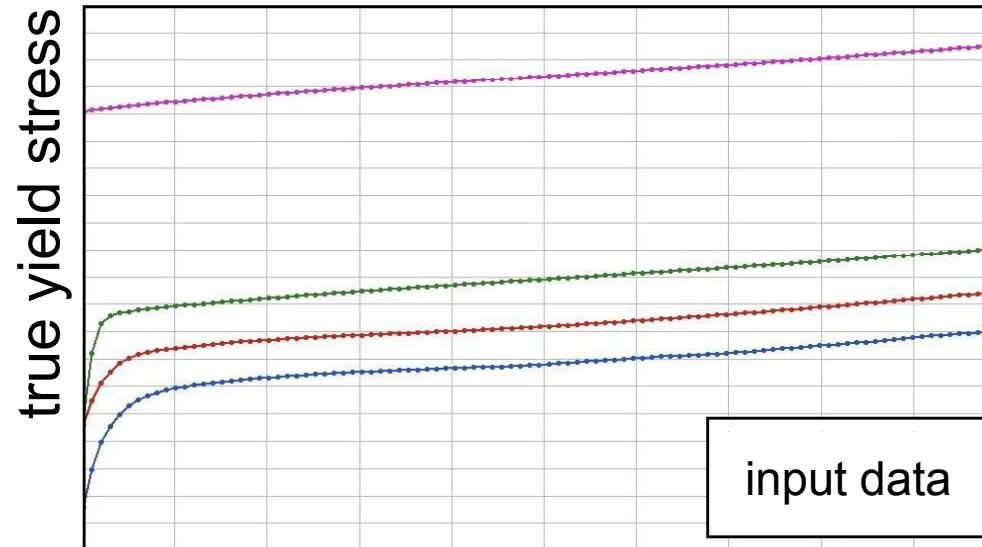
V&V Step 2: Conversion, Smoothing and Sampling



- True strain $\varepsilon = \ln(1 + \varepsilon_0)$
- true stress $\sigma = \sigma_0(1 + \varepsilon_0)$
- This step may be skipped if (local) true stress-strain data is available
- Compute yield curves for each strain rate true plastic strain
- 100 data points are required in the input, thus sampling of the data is necessary:

$$\varepsilon^1 = 0$$

$$\varepsilon^n = \varepsilon^{\frac{nN}{100}}, \quad n = 2, 3, \dots, 100$$



$$\sigma^1 = 0$$

$$\sigma^n = \frac{1}{k_e - k_b + 1} \sum_{i=k_b}^{k_e} \sigma^i, \quad n = 2, 3, \dots, 100$$

$$k_e = \min\left(N, \frac{N}{50}(i+1)\right), \quad k_b = \max\left(1, \frac{N}{50}(i-1)\right)$$



V&V Step 3: Extrapolation after Necking



- Derive the smoothed curve (that is obtained in step 2) numerically by central difference scheme

$$\left. \frac{d\sigma}{d\varepsilon} \right|_n = \frac{\sigma_{n+1} - \sigma_{n-1}}{\varepsilon_{n+1} - \varepsilon_{n-1}}$$

- Identify the onset of the material instability (necking), i.e. find

$$\sigma - \frac{d\sigma}{d\varepsilon} = 0 \Rightarrow \varepsilon^*$$

where ε^* is the strain where necking occurs.

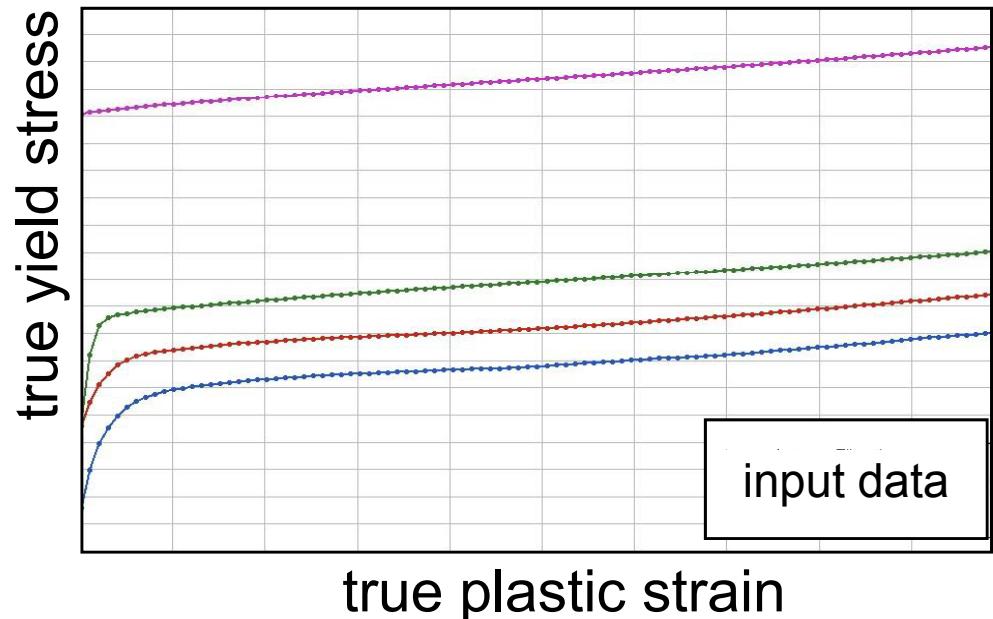
If there is an intersection,
compute for each strain $\varepsilon > \varepsilon^*$:

$$\sigma = \sigma^* e^{(\varepsilon - \varepsilon^*)}$$

where $s^* = s(\varepsilon^*)$

Else Compute the hardening curve:

$$\sigma_y = \sigma, \quad \varepsilon^p = \varepsilon - \frac{\sigma}{E}$$

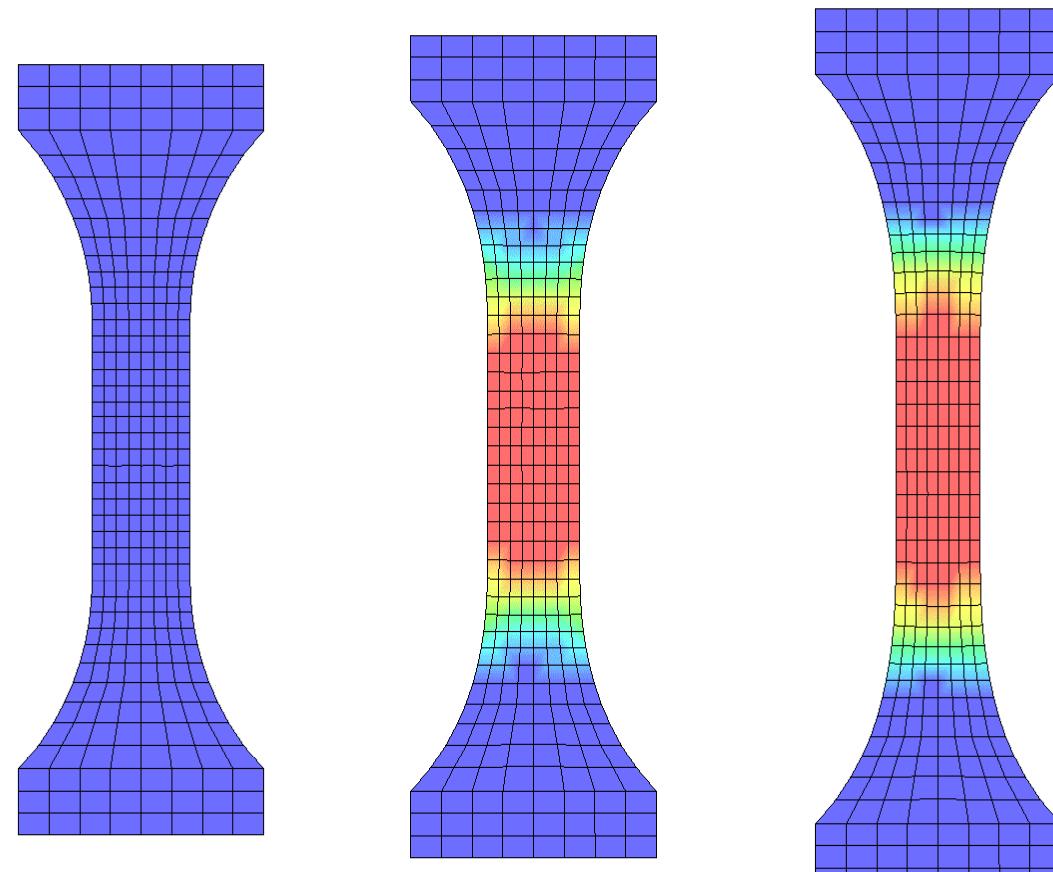




V&V Step 4: Tensile Test Simulation



- Von Mises (piecewise linear) plasticity, linear elastic visco-plastic,
- Generally good representation of tensile responses

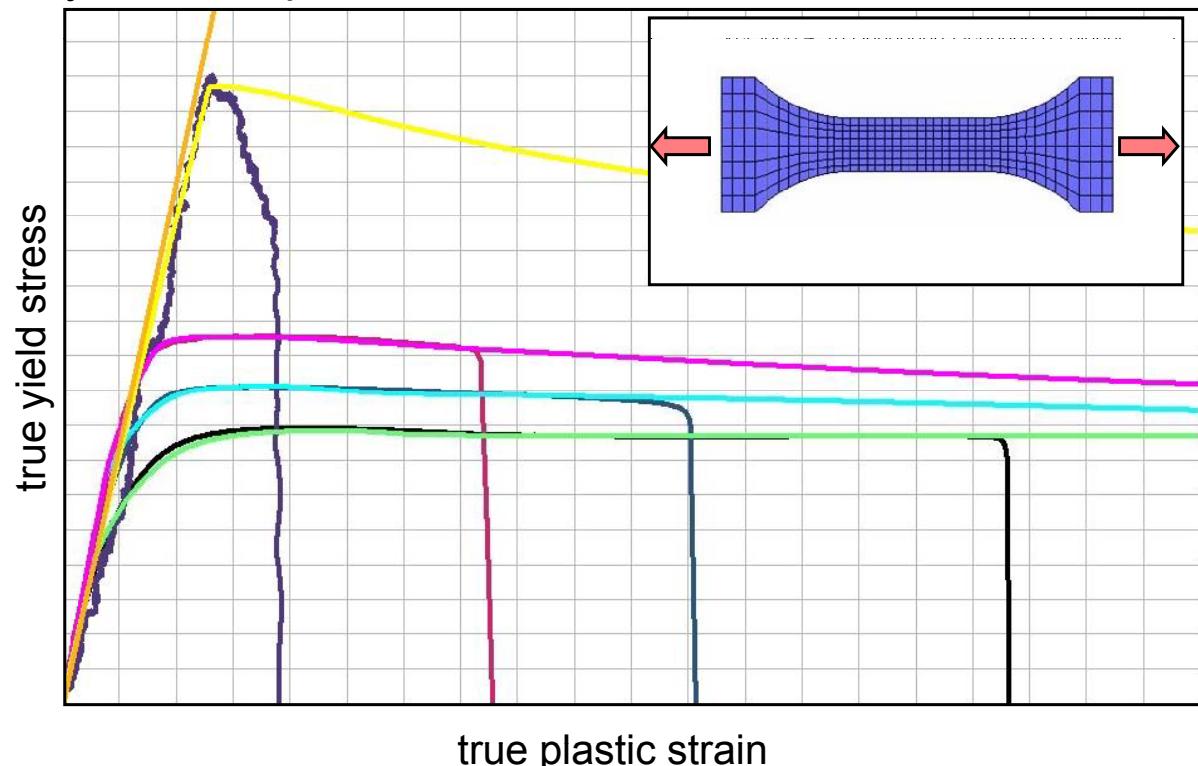
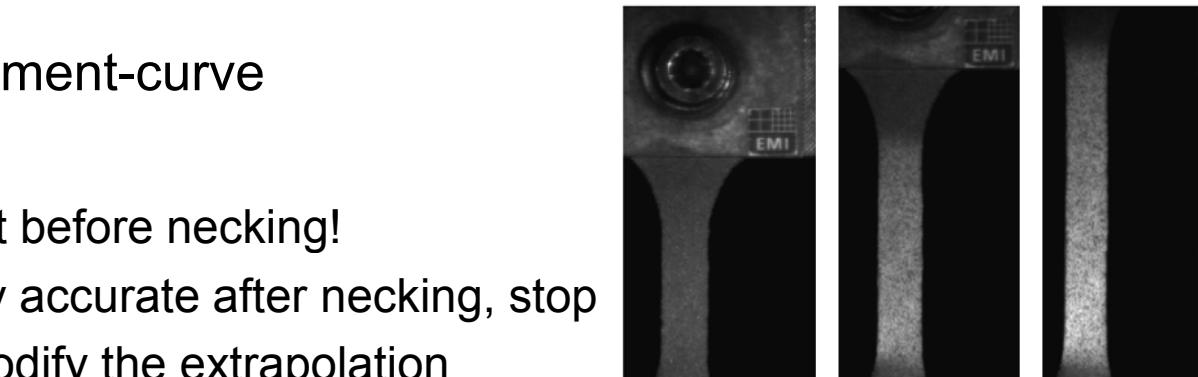
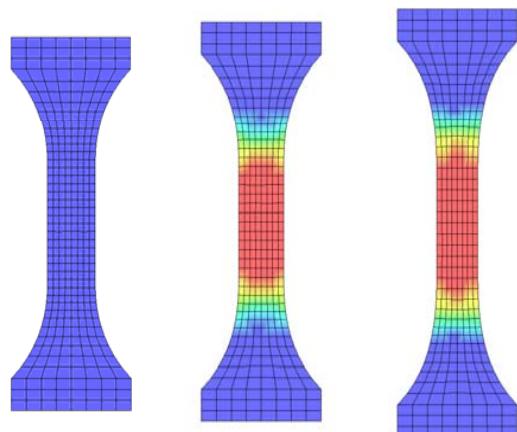




V&V Step 4: Tensile Test Simulation (Loop!)



- Compare force-displacement-curve for each strain rate:
 - Correlation must be exact before necking!
 - If correlation is sufficiently accurate after necking, stop
 - If not, go to step 3 and modify the extrapolation
(e.g. automatically by optimization software)



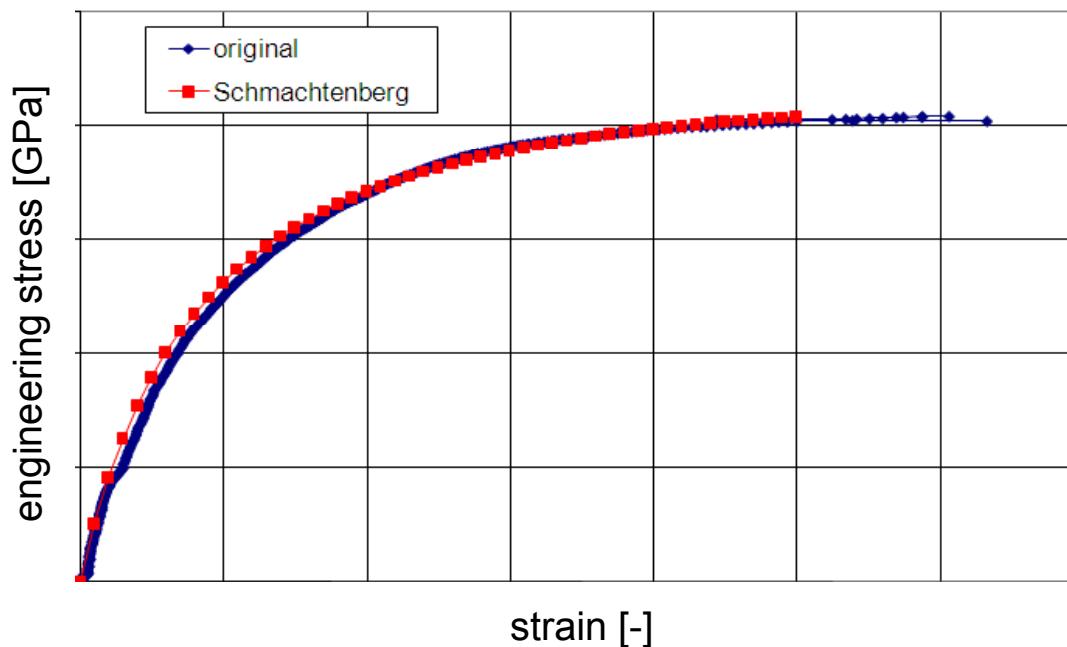
Parameter based Material Laws



- Stress-Strain Relation by Schmachtenberg

$$\sigma = E \varepsilon \frac{1 - D_1 \varepsilon}{1 + D_2 \varepsilon}$$

- Example:
Tensile test



Parameter based Material Laws



- Parameter-identification performed by least square fit

$$S(E, D_1, D_2) := \sum_{k=1}^n \left[\sigma_k(\varepsilon_k) - E \varepsilon \frac{1 - D_1 \varepsilon}{1 + D_2 \varepsilon} \right]^2 \rightarrow \text{MIN}$$

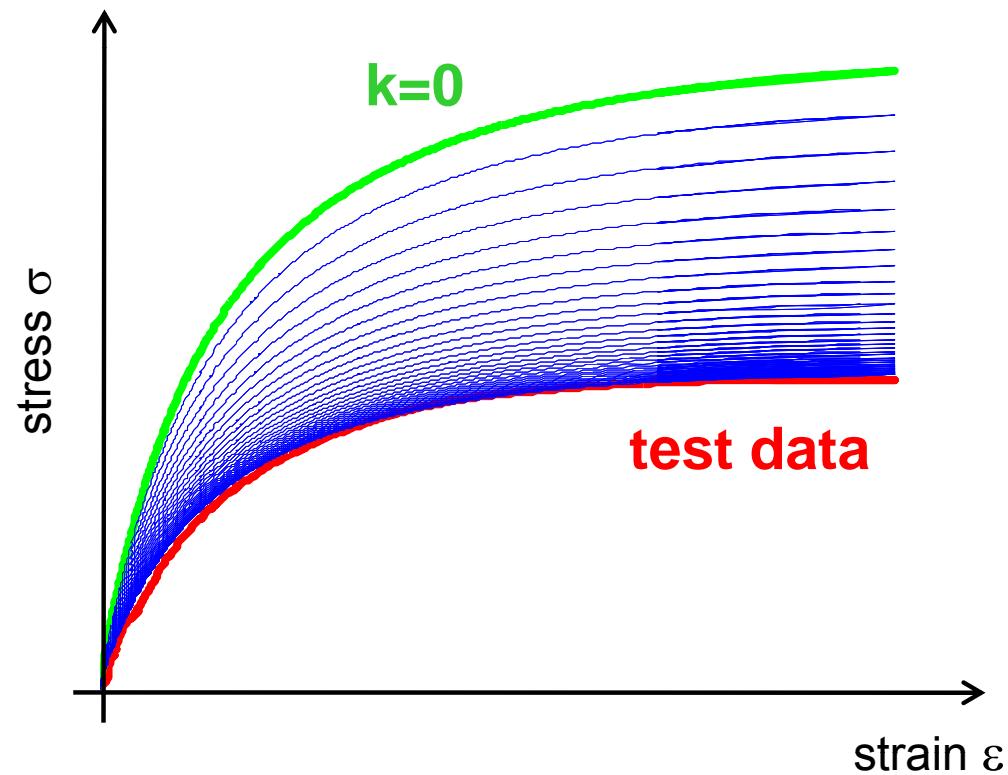
with a gradient method

$$x^{k+1} = x^k - \alpha \nabla S^k$$

$$x^k = [E, D_1, D_2]^k,$$

$$\nabla S^k = \left[\frac{\partial S}{\partial E}, \frac{\partial S}{\partial D_1}, \frac{\partial S}{\partial D_2} \right]^k$$

α = damping parameter





Parameter based Material Laws

- Strain-Rate Dependency by Johnson Cook

$$\sigma_y(\dot{\varepsilon}, \varepsilon_p) = \sigma_y(0, \varepsilon_p) \left[\frac{1 + \ln\left(\frac{\dot{\varepsilon}}{C}\right)}{p} \right]$$

Compute curves
for each strain rate



tabulated input in MAT_24

- Cowper Symonds

$$\sigma_y(\dot{\varepsilon}, \varepsilon_p) = \sigma_y(0, \varepsilon_p) \left[1 + \left(\frac{\dot{\varepsilon}}{C} \right)^{\frac{1}{p}} \right]$$

Parameters C, p can
be used directly
in the MAT_24 card

