

Evaluation of Equivalent Radiated Power with LS-DYNA®

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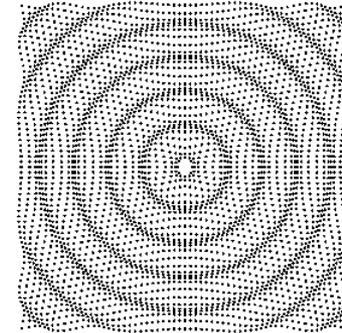
Livermore Software Technology Corporation

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- 1) Introduction
- 2) Brief theory about ERP
- 3) Keywords
- 4) Examples and post-processing
- 5) Conclusions





Animation courtesy of Dr. Dan Russell,
Grad. Prog. Acoustics, Penn State

Acoustic intensity

Sound pressure $P = p e^{i\omega t}$

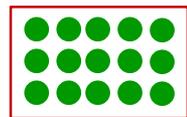
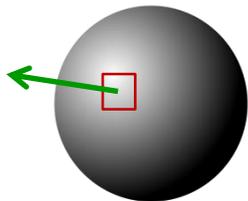
Particle velocity $V = v e^{i\omega t}$

Acoustic intensity $I = \langle PV \rangle_t = \frac{1}{T} \int_0^T PV dt \quad \rightarrow \quad I = \frac{1}{2} \text{Re}\{p \bar{v}_n\}$

Acoustic power

$$W = \int_S I_n dS$$

Time-harmonic wave



Acoustic solvers in LS-DYNA

Time domain acoustics

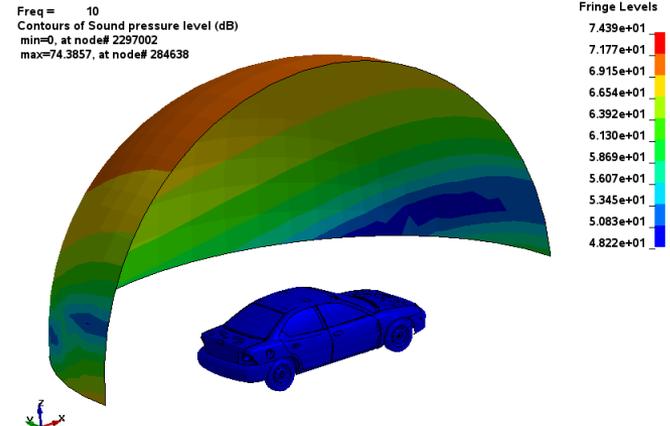
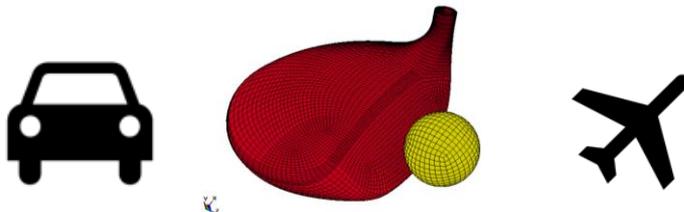
- using **MAT_ACOUSTIC** and **SOLID** formulation 8 and 14

Frequency domain acoustics

- Boundary element method
 - * **FREQUENCY_DOMAIN_ACOUSTIC_BEM**
- Finite element method
 - * **FREQUENCY_DOMAIN_ACOUSTIC_FEM**

Application

- Vehicle NVH
- Acoustic design of sports products
- Transportation acoustics
- Noise control

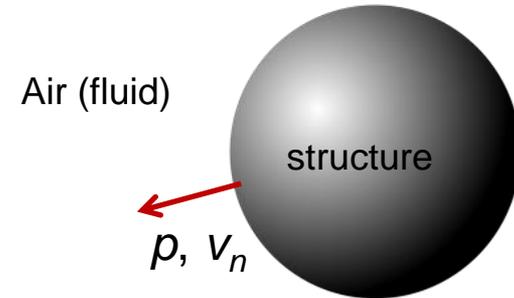


Why ERP is needed?

- Traditional methods, like FEM and BEM, are generally slow and memory intensive.
- ERP is a simple way to characterize structural borne noise
 - ✓ provide information about maximal possible acoustic radiation of panels for specific frequency domain excitation
 - ✓ No air modeling required
 - ✓ No equation systems to solve
 - ✓ good for acoustic optimization based on topology / geometry (direct surface integral)
 - ✓ damping in acoustic volume can be considered in terms of radiation loss factor
 - ✓ acoustic panel contribution analysis
 - ✓ in LS-DYNA, works as a post-processing of SSD

Acoustic intensity $I = \frac{1}{2} \text{Re}\{p \bar{v}_n\}$

Usually, full fluid-structure coupling is needed, to compute the normal velocity and pressure

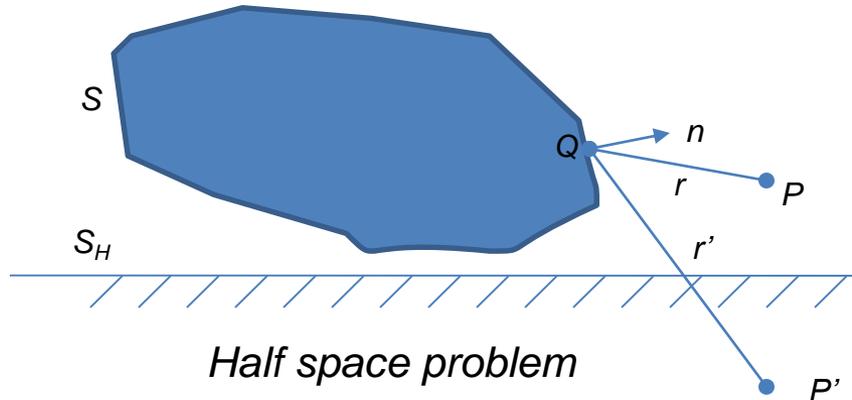


Equality of structure and particle velocity

In linear vibro-acoustics usually the coupling between structure and fluid assumes the equality of the normal velocity of structural surface and the particle velocity of the fluid on the structure surface

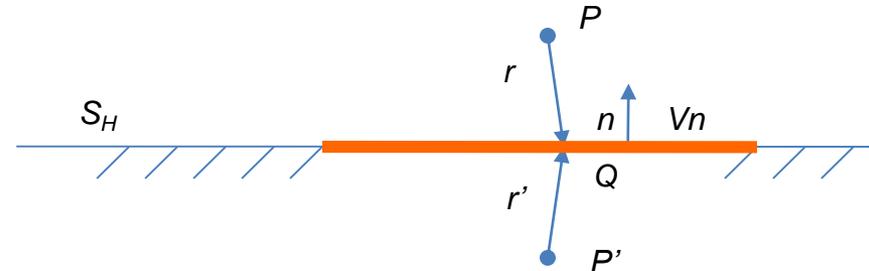
$$v_{S,n} = v_f$$

For rigid panels, one can get the acoustic pressure as follows.
(similar to the Rayleigh method)



Half space problem

Special case when the vibrating surface lies on the reflecting plane



Open space fundamental solution

$$G = \frac{e^{-ikr}}{4\pi r}$$

Half space fundamental solution

$$G_H = \frac{e^{-ikr}}{4\pi r} + \frac{e^{-ikr'}}{4\pi r'}$$

$$P(\omega) = -\int_S \left(i \rho \omega v_n(\omega) G_H + p(\omega) \frac{\partial G_H}{\partial n} \right) ds$$

$$G_H = 2G \quad \partial G_H / \partial n = 0$$

$$P(\omega) = -\int_S 2i \rho \omega v_n(\omega) G ds$$

Pressure

$$p = \frac{i \omega \rho}{2\pi} \int_S v_n \frac{e^{-ikr}}{r} dS$$



Still too much efforts!

$$k = \omega / c$$

The ERP calculation is based on plane wave assumption for the radiated acoustic waves. First we calculate the ERP density, defined as

$$I = \frac{1}{2} \operatorname{Re}\{p \bar{v}_n\} \quad \frac{p}{v_n} = Z = \rho c$$

Plane wave:

- described by only one dimension
- sound pressure and particle velocity are in phase

ERP density $ERP_{\rho} = \frac{1}{2} \rho c \operatorname{Re}[v_n \cdot \bar{v}_n]$

The ERP absolute value radiated from the vibrating panels is the integral of the ERP density over the whole surface and is given by

ERP absolute $ERP_{abs} = \int_S ERP_{\rho} dS = \frac{1}{2} \rho c \int_S \operatorname{Re}[v_n \cdot \bar{v}_n] dS$

ERP in dB $ERP_{dB} = 10 \log_{10} (ERP_{abs} / ERP_{ref})$

Acoustic density $w = \frac{I}{c} = \frac{1}{2} \rho \operatorname{Re}[v_n \cdot \bar{v}_n]$ *Sound energy density (Pa)*

3) Keywords

*FREQUENCY_DOMAIN_SSD_ERP

Card 1	1	2	3	4	5	6	7	8
Variable	MDMIN	MDMAX	FNMIN	FNMAX	RESTMD	RESTDP	LCFLAG	RELATV
Type	I	I	F	F	I	I	I	I

Card 2	1	2	3	4	5	6	7	8
Variable	DAMPF	LCDAM	LCTYP	DMPMAS	DMPSTF	DMPFLG		
Type	I	I	I	I	I	I		

Card 3	1	2	3	4	5	6	7	8
Variable			MEMORY	NERP	STRYP	NOUT	NOTYP	NOVA
Type			I	I	I	I	I	I

<u>VARIABLE</u>	<u>DESCRIPTION</u>
NERP	Number of ERP panels.

ERP Cards: additional Cards 3a and 3b are defined for ERP keyword option.

Card 3a	1	2	3	4				
Variable	RO	C	ERPRLF	ERPREF				
Type	F	F	F	F				

Repeat Card 3b NERP times (one card defines one ERP panel)

Card 3b	1	2						
Variable	PID	PTYP						
Type	I	I						

Card 4	1	2	3	4	5	6	7	8
Variable	NID	NTYP	DOF	VAD	LC1	LC2	LC3	VID
Type	I	I	F	F	I	I	I	I

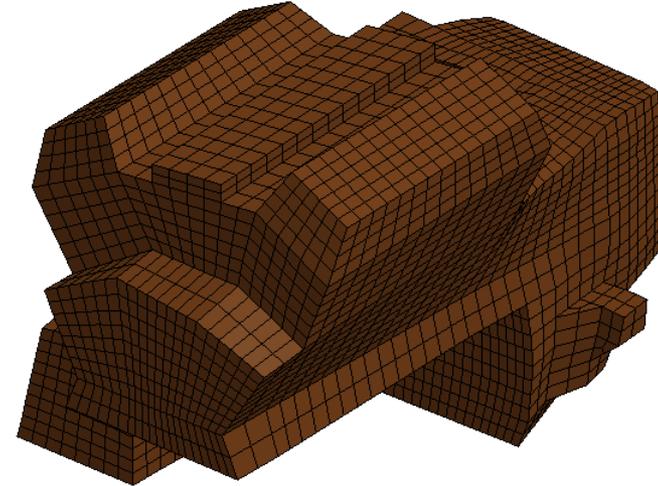
VARIABLE	DESCRIPTION
RO	Fluid density
C	Sound speed of the fluid
ERPRLF	ERP radiation loss factor
ERPREF	ERP reference value. This is used to convert the absolute ERP value to ERP in decibels (dB)
PID	Part, part set, or segment set ID for ERP computation

A typical keyword input

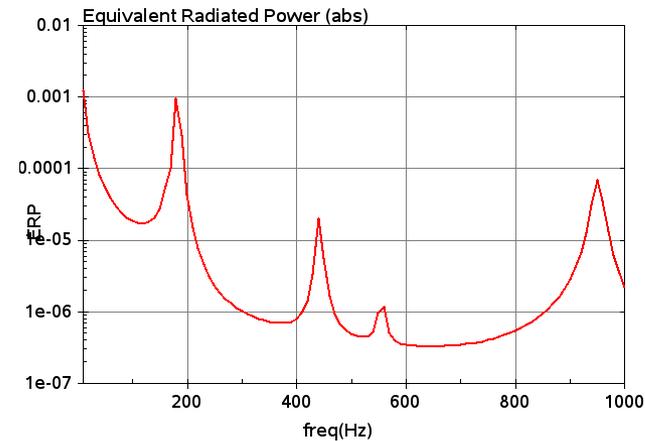
***FREQUENCY_DOMAIN_SSD_ERP**

\$#	MDMIN	MDMAX	FNMIN	FNMAX	RESTMD	RESTDP	LCFLAG	RELATV
	1	100	0.0	2000.0				
\$#	DAMPF	LCDAM	LCTYP	DMPMAS	DMPSTF	DMPFLG		
	0.01							
\$#			MEMORY	NERP	STRTYP	NOUT	NOTYP	NOVA
				1				
\$#	RO	C	ERPRLF	ERPREF				
	1.21	340.0	1.	5.E-13				
\$#	PID	PTYP						
	1							
\$#	NID	NTYP	DOF	VAD	LC1	LC2	LC3	VID
	131	0	3	0	100	200		

For a simplified engine model, a constant horizontal acceleration $0.02g$ is given on the base, for the range of frequency 10-1000 Hz.

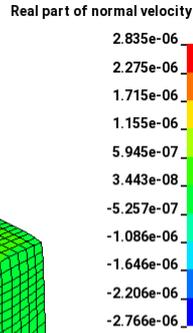
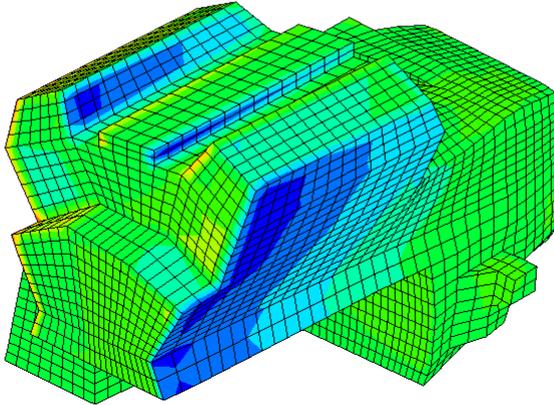


No. of solids: 13484
No. of nodes: 16041
No. of surface segments: 4880

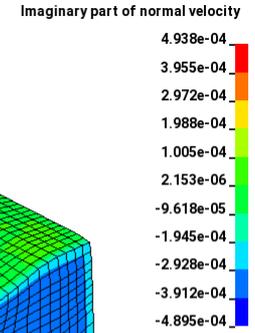
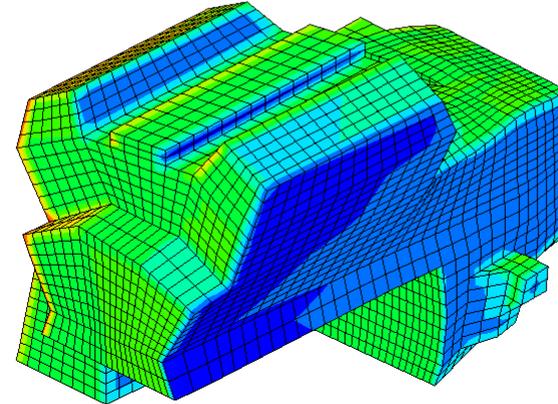


ERP_abs

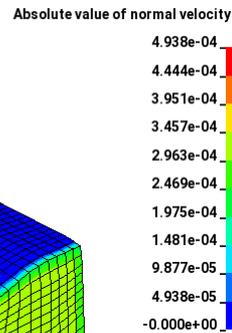
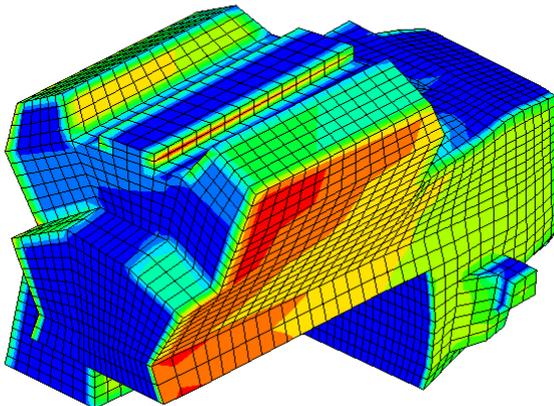
Freq = 100
 Contours of Real part of normal velocity
 min=-2.76615e-06, at node# 2862665
 max=2.83501e-06, at node# 2862599



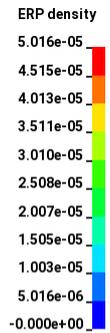
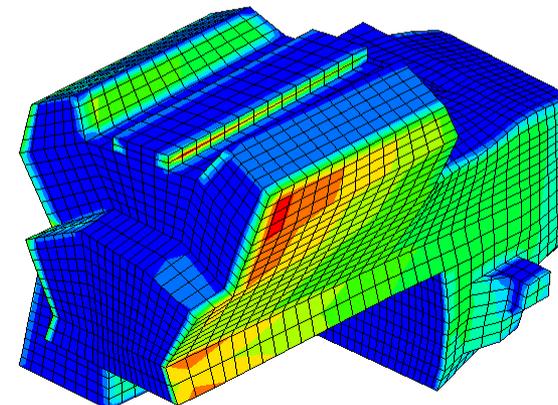
Freq = 100
 Contours of Imaginary part of normal velocity
 min=-0.000489514, at node# 2862665
 max=0.00049382, at node# 2862599



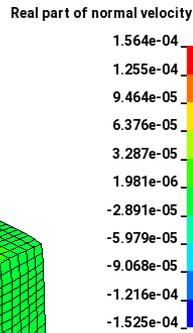
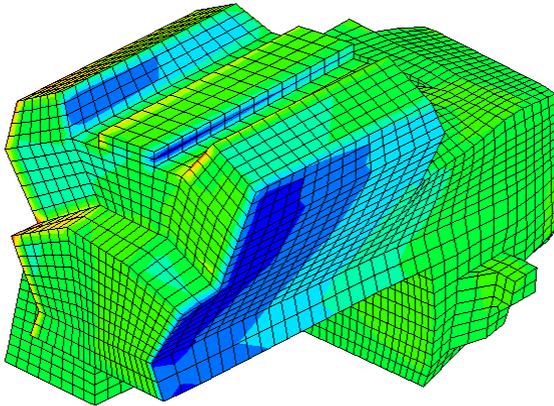
Freq = 100
 Contours of Absolute value of normal velocity
 min=-0, at node# 2849013
 max=0.000493828, at node# 2862599



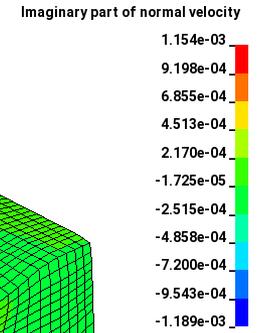
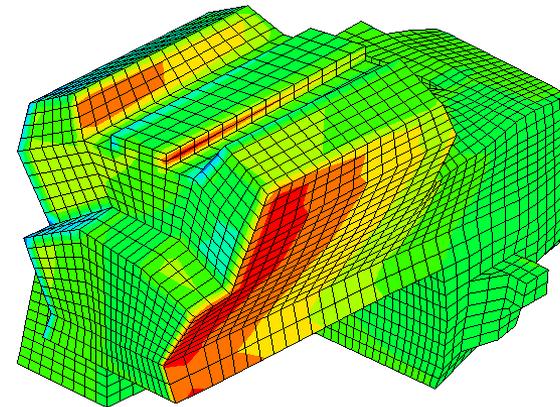
Freq = 100
 Contours of ERP density
 min=-0, at node# 2849013
 max=5.01634e-05, at node# 2862599



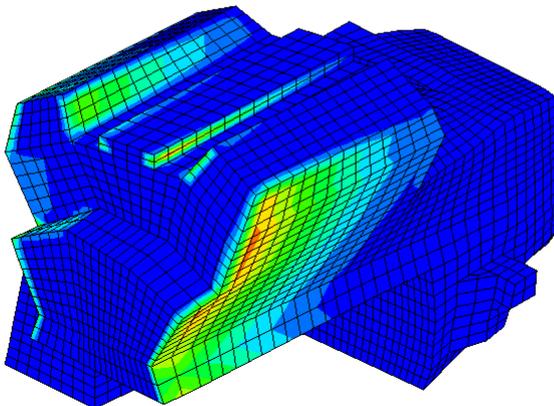
Freq = 200
 Contours of Real part of normal velocity
 min=-0.000152456, at node# 2862665
 max=0.000156418, at node# 2862599



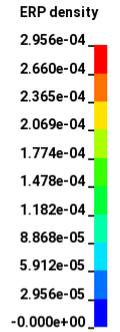
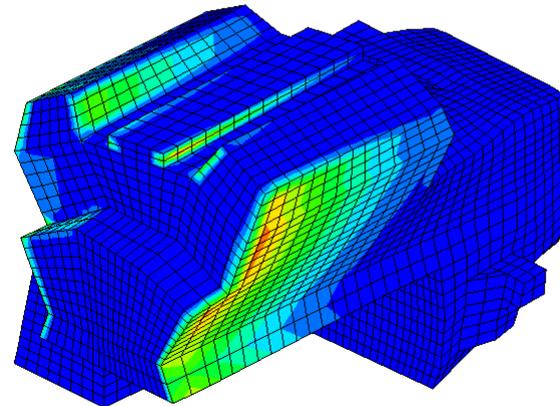
Freq = 200
 Contours of Imaginary part of normal velocity
 min=-0.00118853, at node# 2862665
 max=0.00115402, at node# 2862665



Freq = 200
 Contours of Acoustic density
 min=-0, at node# 2849013
 max=7.1854e-07, at node# 2862599



Freq = 200
 Contours of ERP density
 min=-0, at node# 2849013
 max=0.000295607, at node# 2862599



GUI for post-processing of d3erp

LS-PP LS-PrePost(R) V4.3 - 27Sep2016(10:00)-64bit ing/2012_detroit/exercises/test/5.4/erp/2016.08.31.new/d3erp

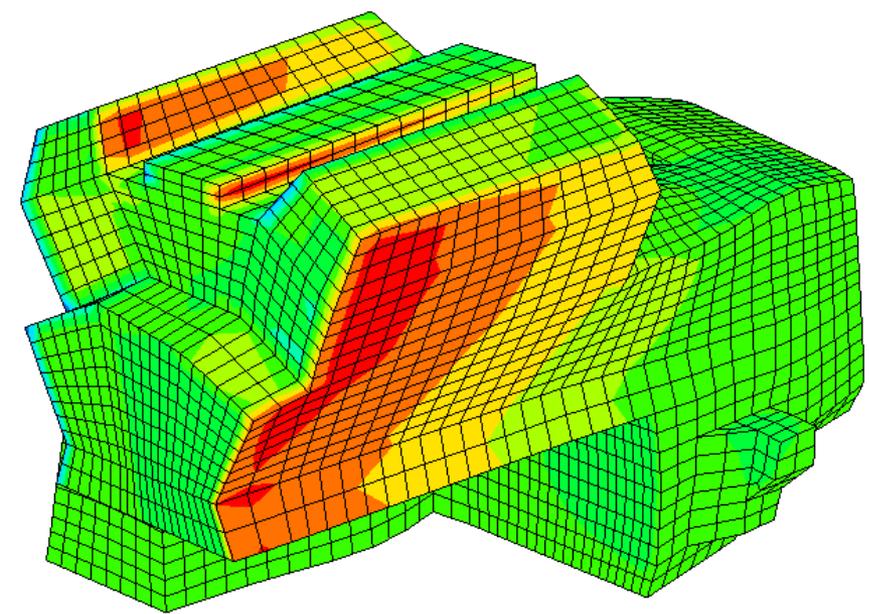
File Misc. View Geometry FEM Application Settings Help

A simplified engine model for SSD ERP c
 Freq = 10
 Contours of Real part of normal velocity
 min=-0.000802643, at node# 2862599
 max=0.000784196, at node# 2862665

Real part of normal velocity



7.842e-04
 6.255e-04
 4.668e-04
 3.081e-04
 1.495e-04
 -9.223e-06
 -1.679e-04
 -3.266e-04
 -4.853e-04
 -6.440e-04
 -8.026e-04



LS-PP NVH Fri...mponent

- D3SSD Real part of normal veloc
- D3SPCM Imaginary part of normal
- D3PSD Absolute value of normal
- D3RMS Acoustic density
- D3RMS ERP density
- D3FTG
- D3ACS
- D3ATV
- D3EIGV_AC
- D3ERP

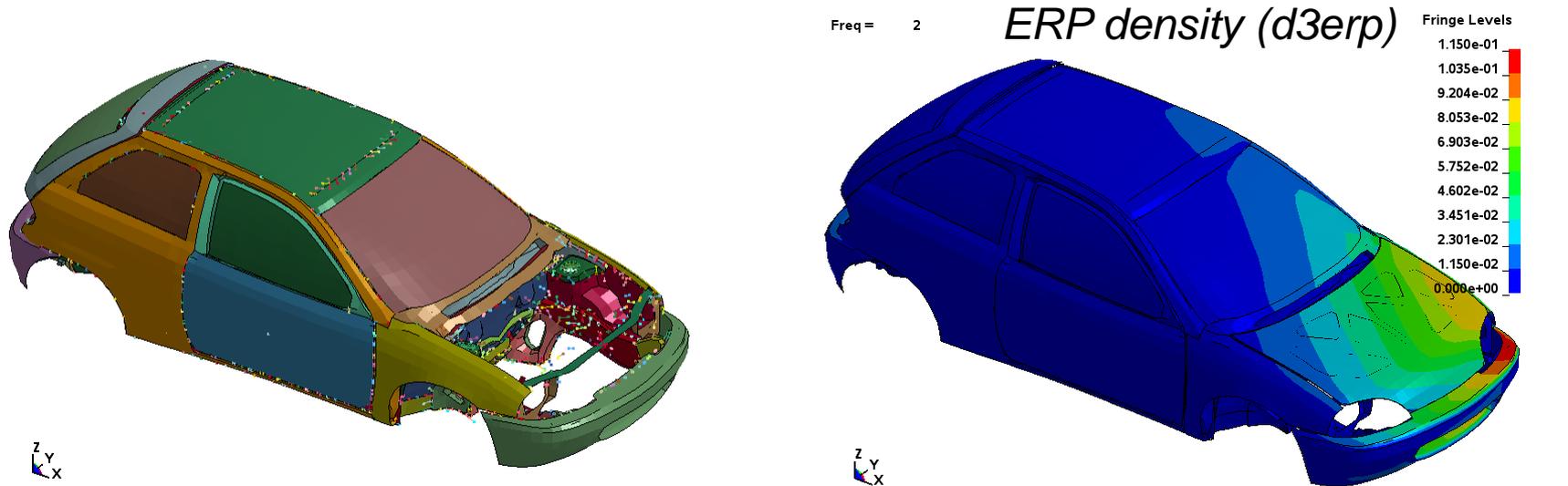
Frin ▾
 Max ▾
 Glob ▾

Done

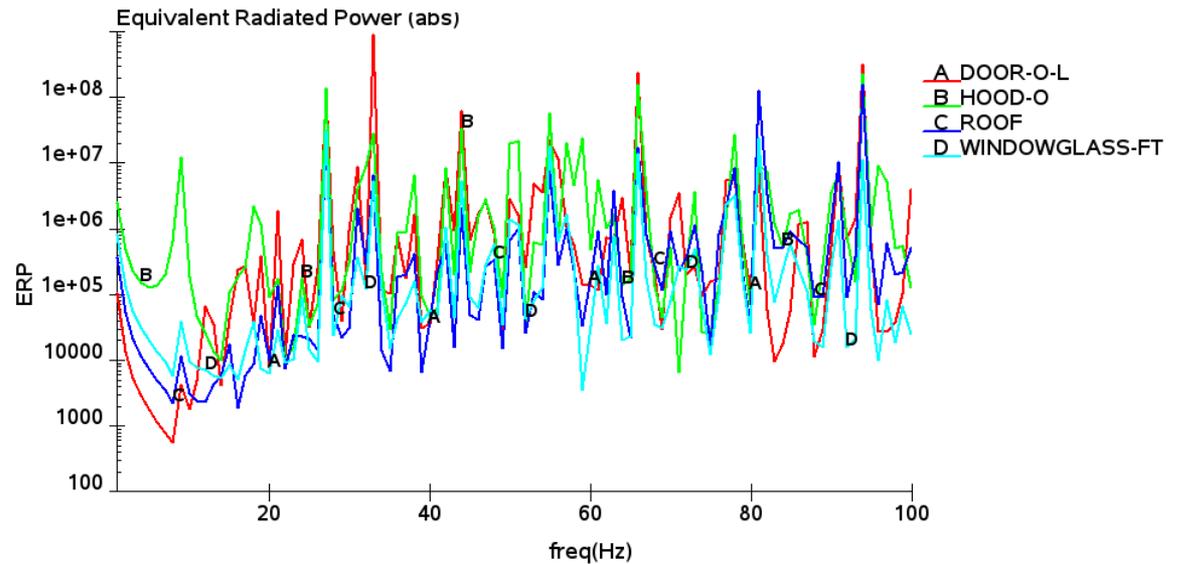
fringe

Select a fringe component

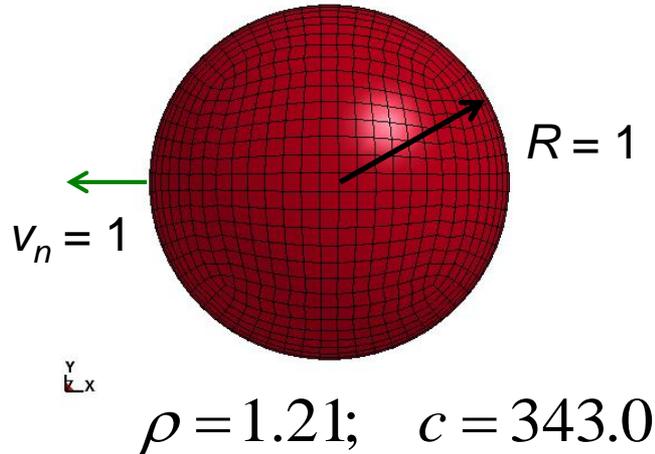
Normal field...



With ERP, one can study the contribution to the radiated noise from each panel.



Pulsating sphere



$$W = \frac{1}{2} \rho c \int_S \text{Re}[v_n \cdot \bar{v}_n] dS$$

$$= \frac{1}{2} \rho c \int_S dS$$

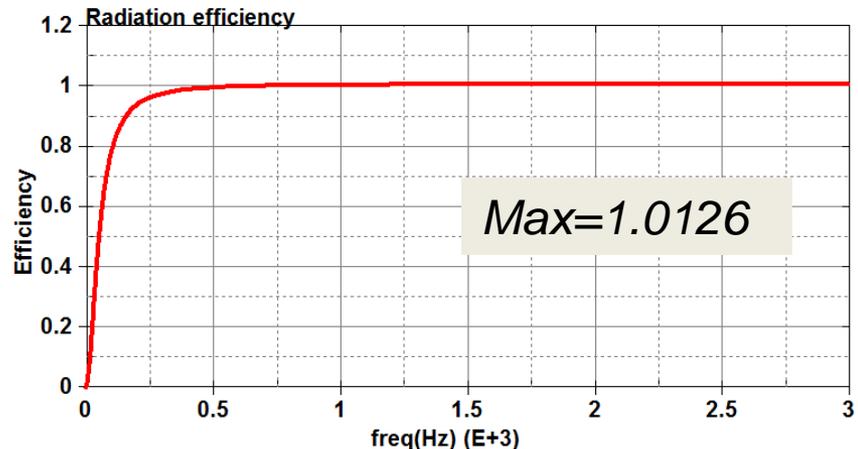
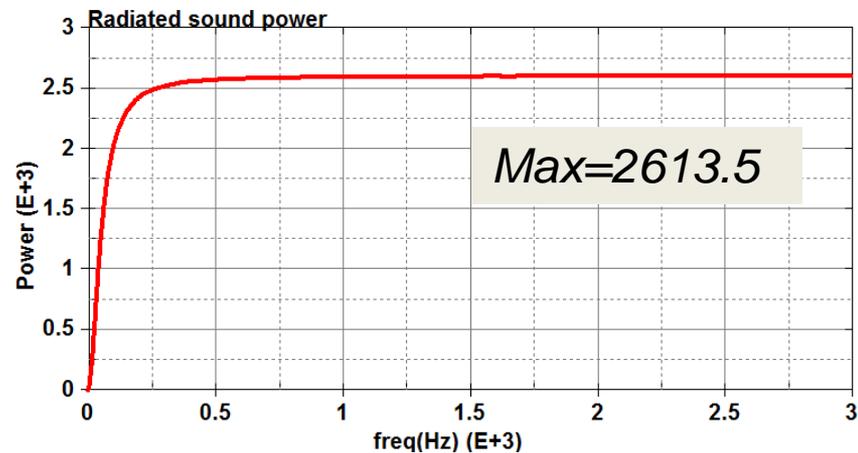
$$= 2\pi \rho c R^2$$

$$= 2607.7$$

* FREQUENCY_DOMAIN_ACOUSTIC_BEM

Press_Power

Press_radeff



5) *Conclusions*

- A feature for ERP calculation is implemented
- Post-processing of the calculation results is provided
- Need to add element ERP absolute to d3erp
- More validation / testing
- Optimization based on ERP?

THANK YOU!

for your attention

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