# Recent Developments of LS-DYNA® in Stamping Simulation

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#### **Outline**

- Implicit method
- Material work-hardening
- Some new keywords
- Surface low detections
- Conclusions

## Improvements in Implicit Method



### Implicit Method

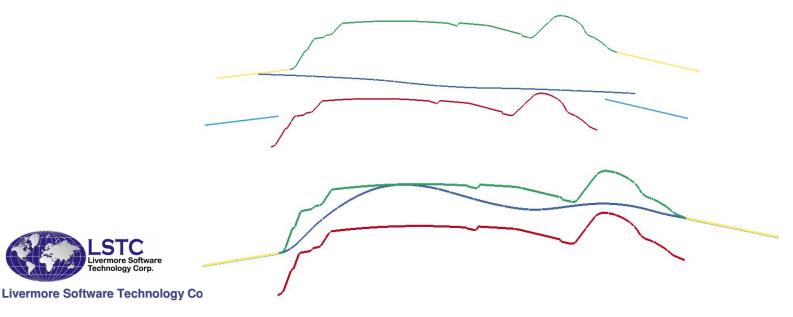
## Implicit method has been gaining more popularities in sheet stamping simulation

- Initial application was mainly limited to springback predictions
- Gravity loading simulation has been proven to be robust with implicit method
- Implicit method has also shown great potentials in flanging simulation.

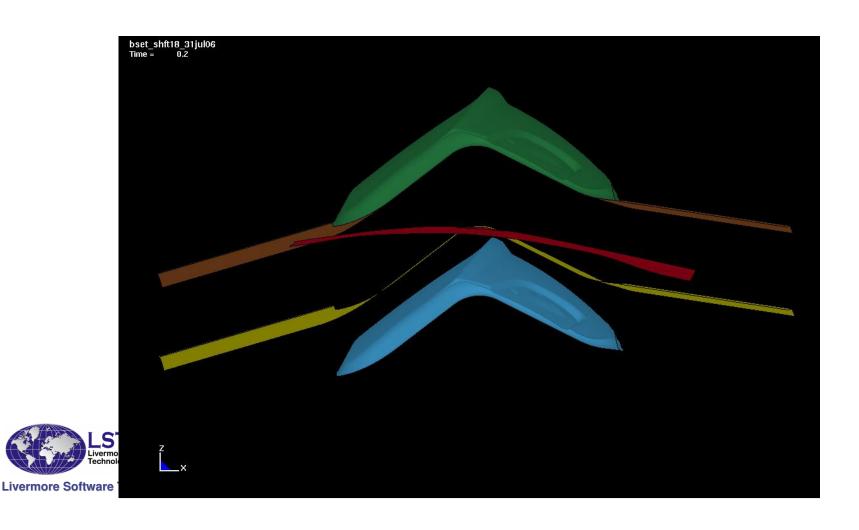
- Binder wrapping characteristics
  - Large blank movements, large dynamic effect
  - Small plastic deformation
  - Element is relatively coarse, and the number of element is small
  - Implicit method might be the preferred one
- Recent developments
  - New implicit contact
  - New features to help convergence
  - An extensive parameter studies
    - The default parameters are suitable for most of the binder wrapping process
    - It is easy to use
    - User in-dependent results can be obtained



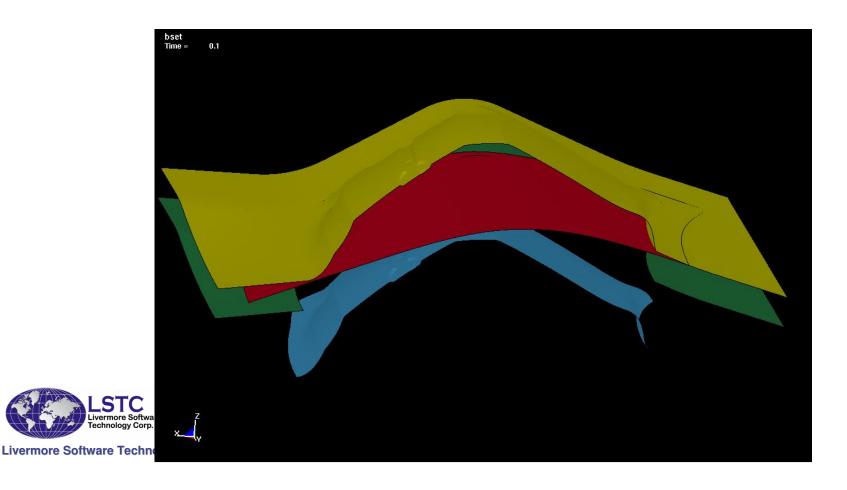
- The proper step size for a typical binder wrapping process
  - The new algorithm allow large time step. While the old time will limit the time step size
  - Too small time step will require many time steps
  - Too large time step will require more iterations, sometimes, the result is not good



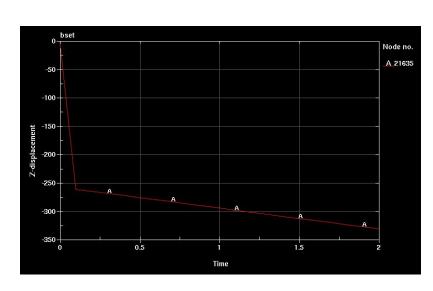
Too small time step might have convergence problem

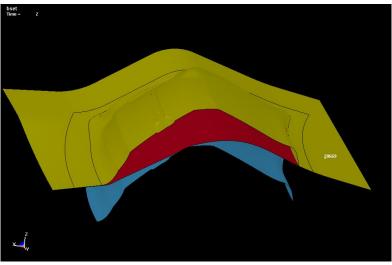


Reasonable time step should make sure that most of the rigid body motions are constrained



- A typical time step selection
  - 10 time steps are used
  - It took 12 minutes to finish with ONE CPU



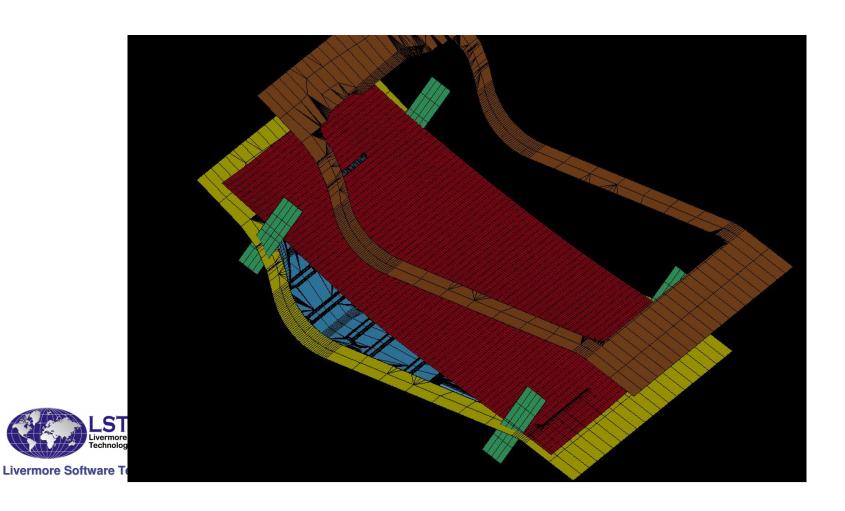


- Binder and Gravity Loading are combined into one simulation
  - In the old method, gravity loading has to be done separately
  - It allow mesh refinement during each step

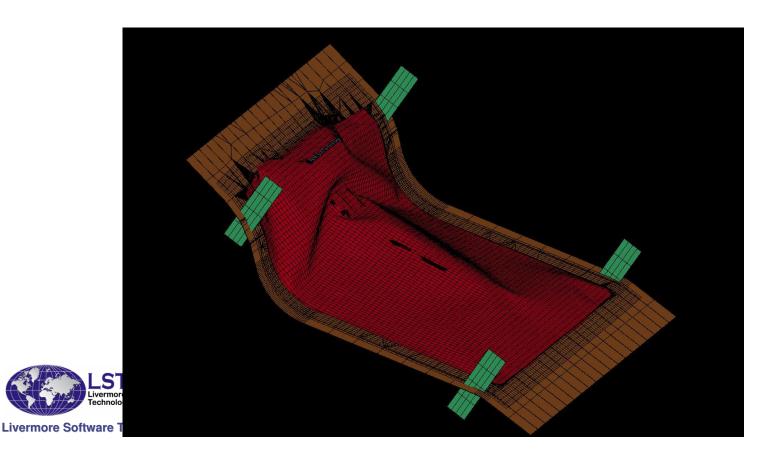
It allows quide pipe



After Gravity Loading, blank elements are refined



- Final geometry: all the buckling mode has been correctly simulated
  - It took 90 minutes to finish



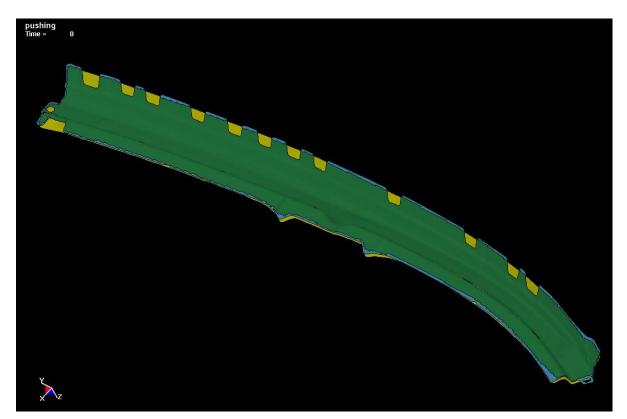
### Implicit Method: Other Applications

- Initially developed for metal stamping simulations that involve gravity loading of blanks on dies.
- Now extended as a general capability
  - Serial, SMP and MPP implementation
- Robust contact treatment
- Vehicle does not need to be supported to eliminate rigid body modes
- Elimination of loose parts not required
- Reduces model preparation time dramatically over traditional implicit method.

- Sometimes, springback predictions are not accurate
  - Scan data can be used to compensate the springback deviations
- Procedure in using scan data
  - Perform a forming simulation, and obtained the deformed part
  - Assume the scan data as rigid tools
  - Assume the deformed as a deformable
  - Apply internal pressure to the deformed part
  - After push, the blank geometry can be used as sprung shape
  - Use the same procedure as before and compensate the rigid
     tool

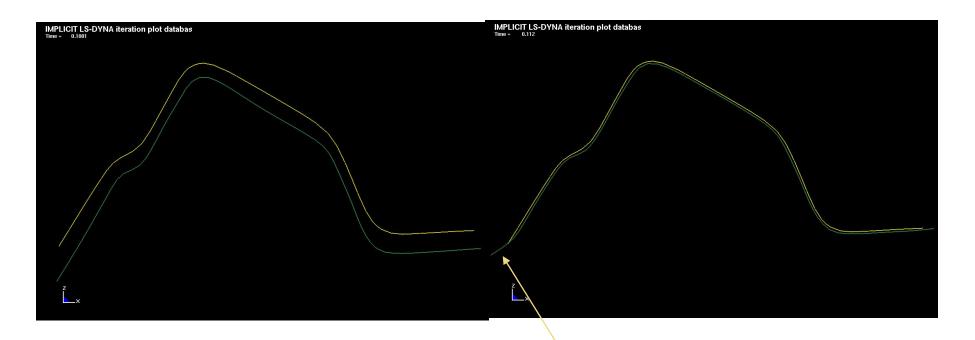
- How to push the deformed part to the scanned geometry?
  - The pressure is applied by using load mask
    - All the blank element will have a normal pressure
    - The pressure is applied to the opposite of element normal directions
      - Blank normal has to be checked
    - For most of the situation, the internal pressure can be in the range of 20 ~ 30 MPA
  - Use implicit method
    - Use the keyword: \*CONTROL\_IMPLICIT\_FORMING
    - One step pushing is used
    - The CPU time is small (usually can be done within a few minutes)

- Benchmark Study:
  - Number of element: 41,457
  - CPU cost: 7 minutes and 39 seconds





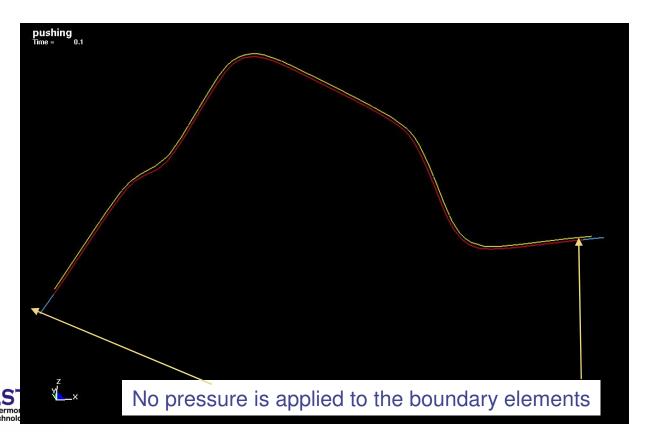
- Problems might happen during the pushing:
  - No support for the boundary elements





Over-bend in the boundary areas

- Change the boundary elements to a different part
  - Avoid applying pressure to the boundary element

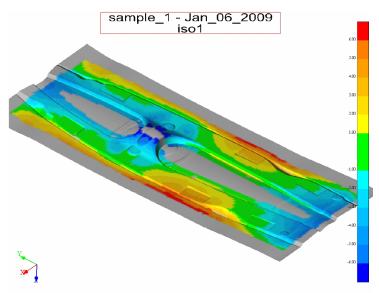


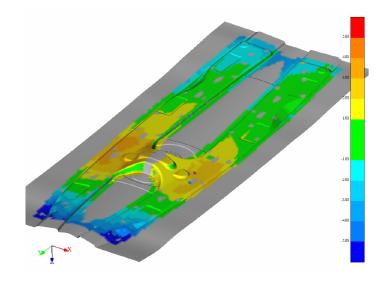
- After first tryout, the parts exhibited significant springback issues.
- Part could not even be placed on the checking fixture without extreme hand-working.
- Part was bowed throughout the length of the channel, twisted, there were bulges adjacent to the deeper areas of form, form depth in the deeper areas was incorrect, and the stepped flanges were crowning.
- Part showed that it was underbent just about everywhere.



Example:

- There is a dramatic change.
- A significant amount of springback has been removed
- With adjustments to magnification, an even more effective compensated shape could have been created.







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## Material kinematic hardening

Yoshida model Chaboche's model



### Springback Prediction for HSS

- Some parts are sensitive to stress noise
  - More accurate stress calculation is important



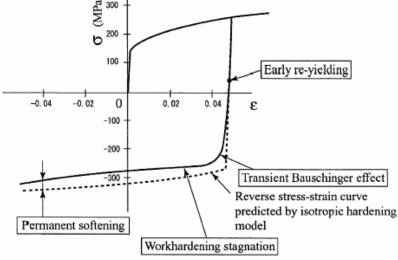


Twisting mode corresponding to the lowest frequency (17.97) And is far smaller than the next one: 54.7 Accordingly, small stress noise might excite the wrong twisting mode

- Recent researches found that Yoshida nonlinear kinematic hardening is the preferred one.
  - It can describe the softening effect from reverse loading

It can accurately represent the stress-strain curve

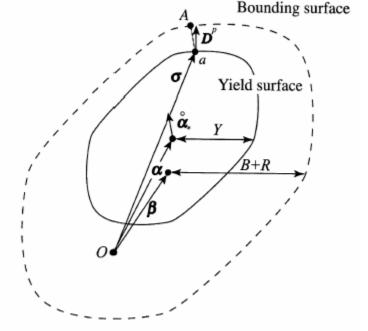
from the cyclic loa



- Key points of Yoshida's theory:
  - Yield surface does not change in size
  - Center of yield surface moves with deformation
  - Bounding surface change both in size and location

$$\alpha_* = \alpha - \beta$$
.

$$\overset{\circ}{\boldsymbol{\alpha}}_* = C \left[ \left( \frac{a}{Y} \right) (\boldsymbol{\sigma} - \boldsymbol{\alpha}) - \sqrt{\frac{a}{\bar{\alpha}_*}} \, \alpha_* \right] \dot{p}, 
\dot{p} = \sqrt{(2/3)} \boldsymbol{D}^p : \boldsymbol{D}^p, \quad \bar{\alpha}_* = \phi(\boldsymbol{\alpha}_*), \quad a = B + R - Y,$$





Bounding surface changes in both its size and location:

$$\dot{R} = k(R_{sat} - R)\dot{p},$$

$$\mathbf{\beta}' = k \left( \frac{2}{3} b \mathbf{D}^p - \mathbf{\beta}' \dot{p} \right)$$

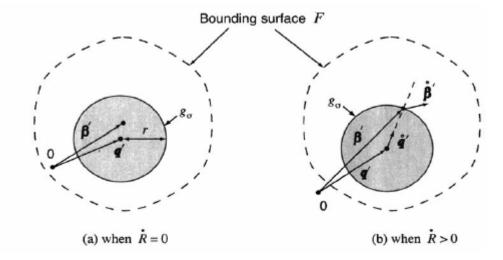
$$\sigma_{bound} = B + R + \beta = B + (R_{sat} + b)(1 - e^{-k\varepsilon^p}).$$

Work-hardening stagnation

$$g_{\sigma}(\boldsymbol{\sigma}', \boldsymbol{q}', r') = \frac{3}{2}(\boldsymbol{\sigma}' - \boldsymbol{q}') : (\boldsymbol{\sigma}' - \boldsymbol{q}') - r^2 = 0,$$

$$\mathbf{q}' = \mu(\mathbf{\beta}' - \mathbf{q}').$$

$$\dot{r} = h\Gamma, \quad \Gamma = \frac{3(\beta' - q') : \dot{\beta'}}{2r}$$



## Material Data Fitting Comparisons (DP600)



#### **Material Parameters**

Y = 0.253E+03

m = 0.405E+02

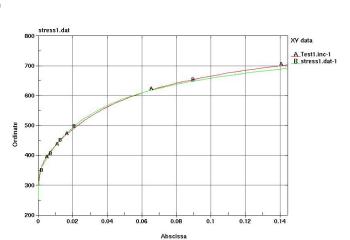
B = 0.342E+03

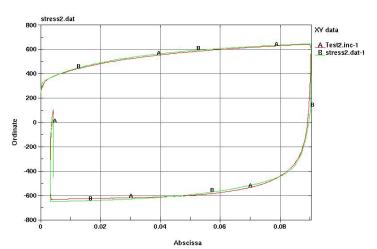
h = 0.100E+01

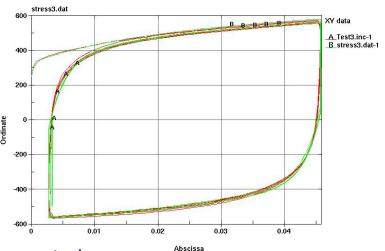
c = 0.430E + 03

b = 0.155E+03

rsat = 0.543E+03



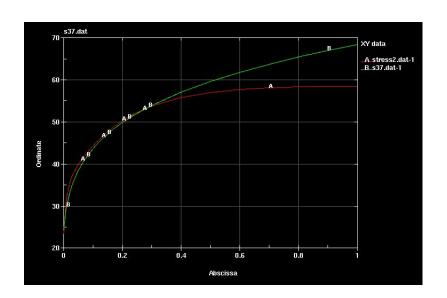




obsaiced parameters can nicely represent the stress-strain curves between loading paths.

## Yoshida's Model Characteristics

- Yoshida's Model can give good fit of the test
  - The effective strain usually small (<0.16)
- Yoshida Model shows saturation of stress

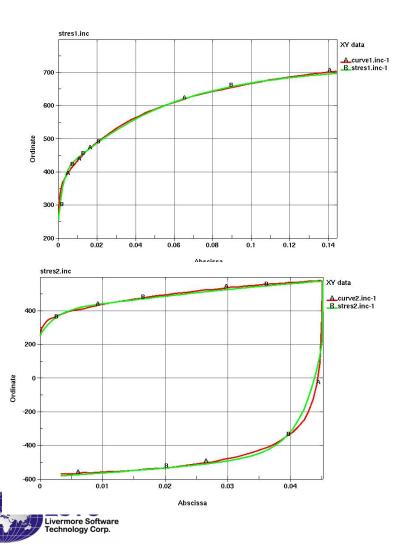


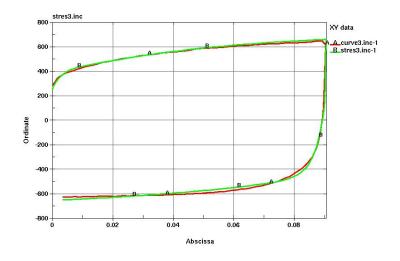
### Chaboche's Model (M103)

- Difficulty in using it
  - There are eight user-defined material variables, make it difficult for ordinary user to use it
  - The build-in curve fitting only works for one stress-strain curve
    - Uniaxial-tension curve has to be used
    - It is impossible to get an reasonable material parameters
- An optimization algorithm has been developed
  - It is a stand-along code
  - Up to eight stress-strain curves can be used as input
    - The file names should begin from curve1.inc, curve2.inc...
  - Many iterations will be needed

$$\overline{\sigma} = \sigma_0 + Q_{r1}(1. - \exp(-C_{r1}\overline{\varepsilon})) + Q_{r2}(1. - \exp(-C_{r2}\overline{\varepsilon})) + Q_{x1}(1. - \exp(-C_{x1}\overline{\varepsilon})) + Q_{x2}(1. - \exp(-C_{x2}\overline{\varepsilon}))$$

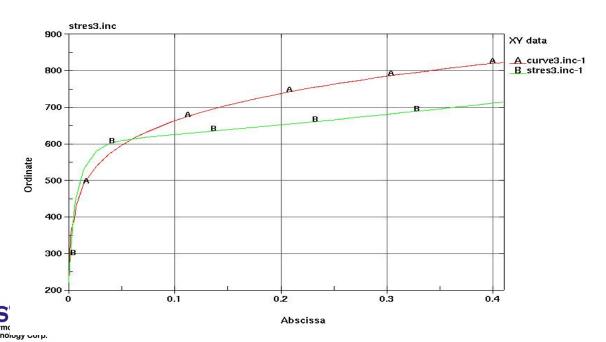
## Chaboche' Mixed-Hardening Model DP600





## Chaboche' Mixed-Hardening Model Saturation problem

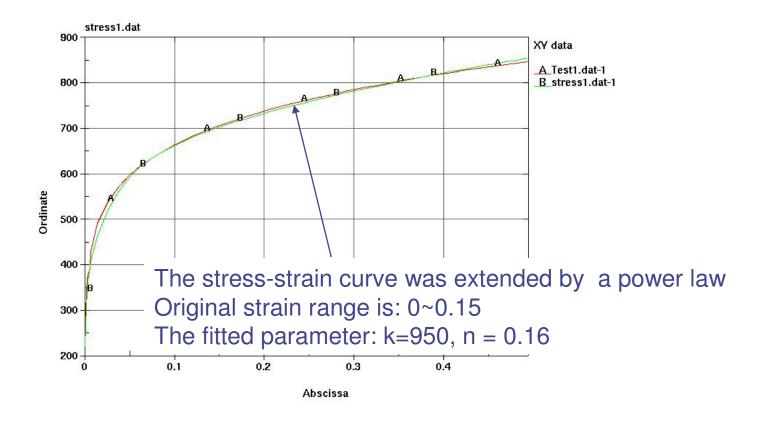
- Chaboche's Model has saturation problem
  - The stress strain curve is extended by power law
  - Then fit the curve and obtain the parameters
  - The fitted curve does not match the experimental data



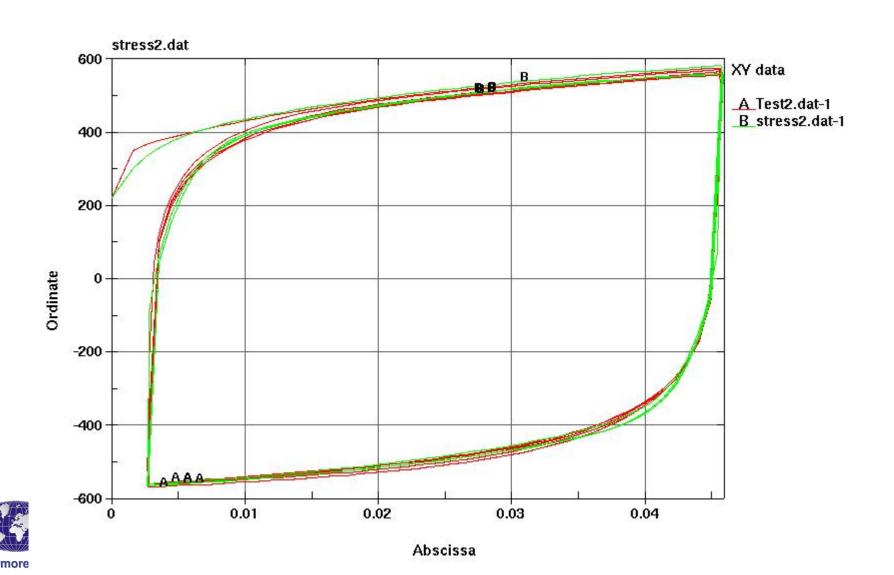
## Modifications of Yoshida's Model New Proposal

- A new algorithm has been proposed and tested
  - Use a power-law to replace the R calculation in Yoshida's model
  - The old function:  $\dot{R} = k(R_{sat} R)\dot{p}$ ,
  - The new function: R=Rsat(e+e0) <sup>n</sup>-Rsat(e0)<sup>n</sup>
  - This new function will not have saturation problem
  - There are two more parameters need to be fit: e0 and n

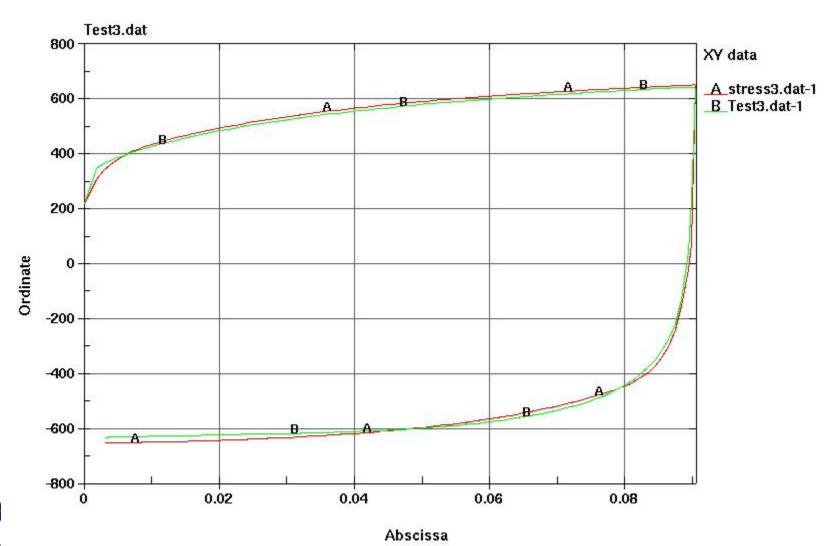
## Modifications of Yoshida's Model Example: DP600



## Modifications of Yoshida's Model Example: DP600



## Example: DP600





## Some New Keywords

important for line-die simulation

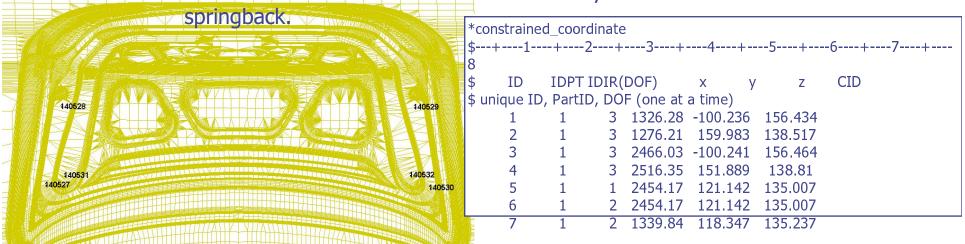
### Coordinate-based constraint

# Why we need to put constraints on coordinates

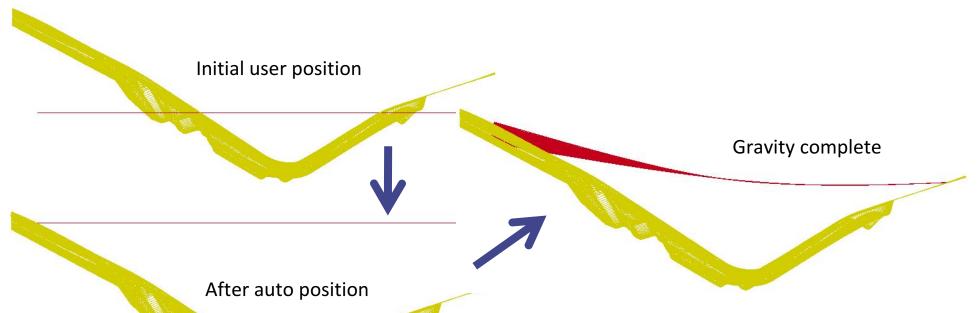
- It can be more accurate to constraint the model in a fixture
- It can automate line-die simulation

### The keyword is

- \*CONSTRAINED\_COORDINATE
- It can also be applied to local coordinate system
- Coordinates can be obtained from stationary tool in the CASE before



### \*CONTROL\_FORMING\_AUTOPOSITION\_PARAMETER



\*PARAMETER \$ PRMR1

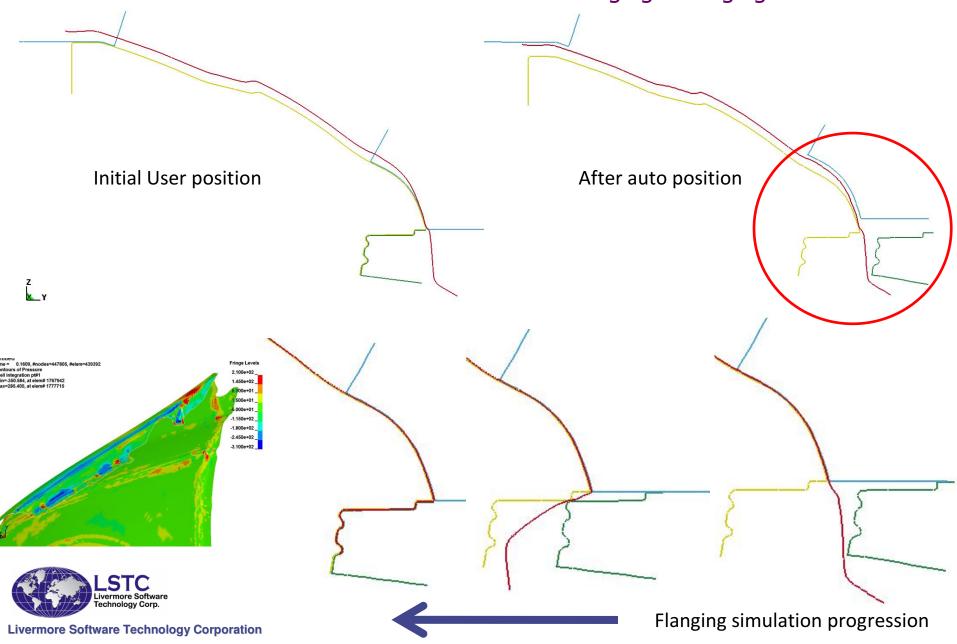
VAL1

```
R blankmv
          0.0
$---+---5---+---6---+---8
*control_forming_autoposion_parameter
               DIR
                    MPID position premove THICK PARORDER
   PID
$ blankmv
                                               blankmv
$---+---5---+---6---+---7----8
*PART MOVE
$ PID
          VOMX
                    YMOV
                              ZMOV
                                    CID
            0.0
                      0.0
                           &blankmv
$$$$$$$$$$$$$$$$$$$$$$$$$$$$$$$$$$$$
*CONTROL TERMINATION
                DTMIN ENDNEG ENDMAS
$ ENDTIM ENDCYC
 1.0000
*CONTROL IMPLICIT forming
```



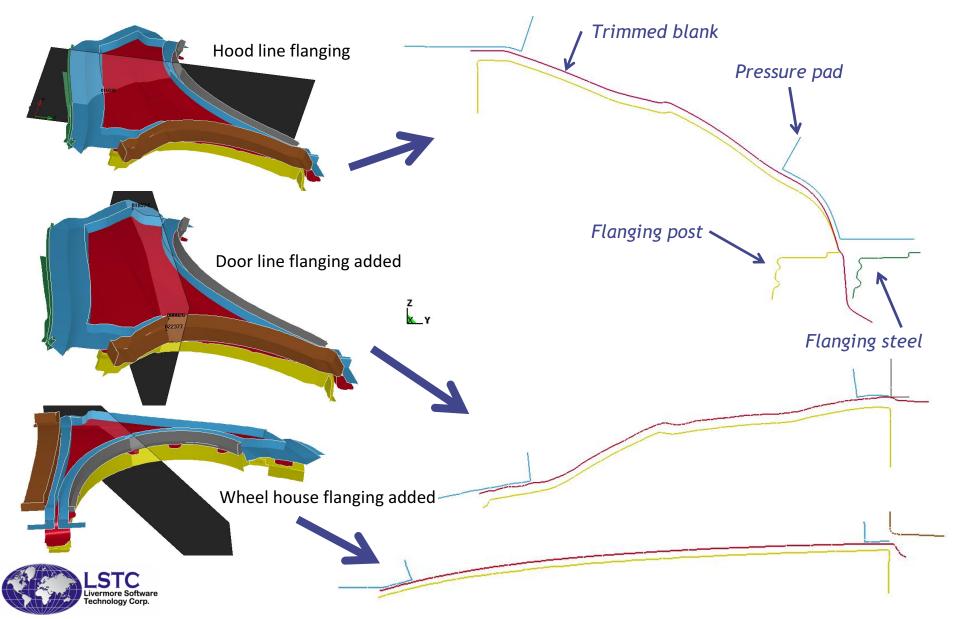
### Line Die Simulation

NUMISHEET Fender on Air with Flanging - Flanging



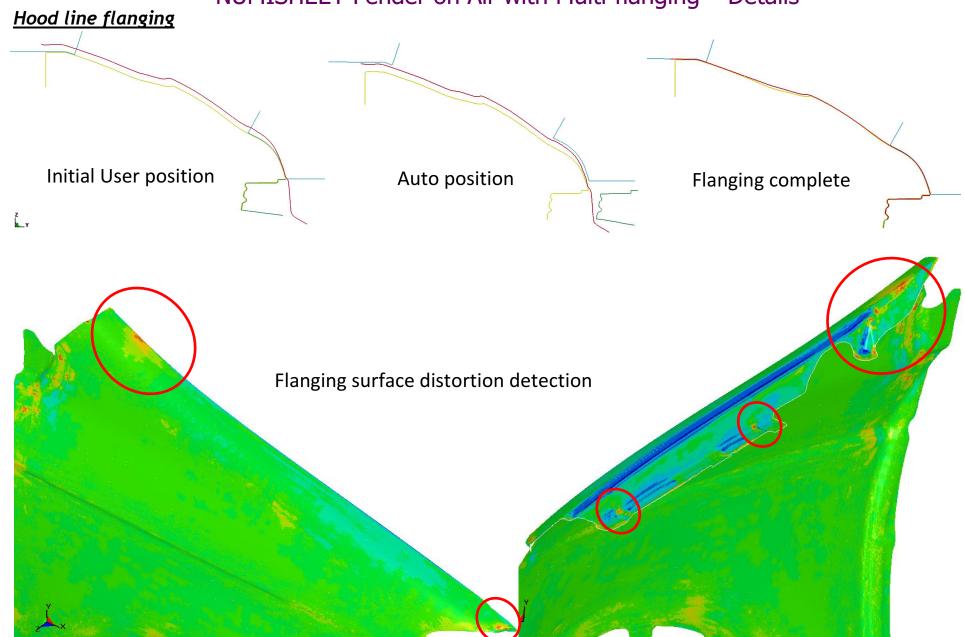
### Line Die Simulation

NUMISHEET Fender on Air with Multi-flanging – Flanging in Three Areas

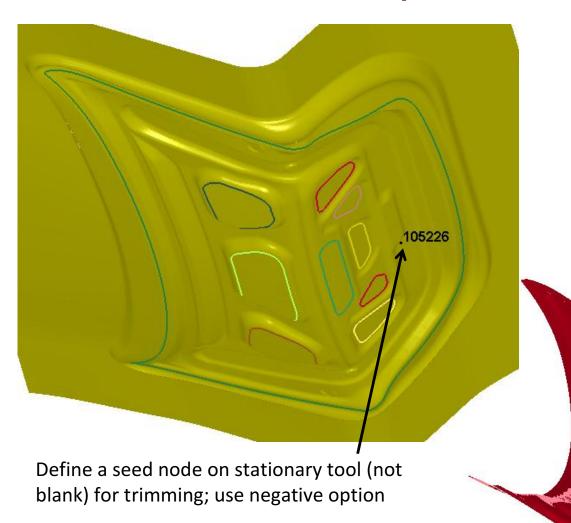


### Line Die Simulation

NUMISHEET Fender on Air with Multi-flanging – Details



## **New Option for Trimming**



\*node 105226,2026.19,292.148,-134.788 \*DEFINE\_CURVE\_TRIM\_NEW \$# tcid tctype tflg tdir tctol toln nseed 1 2 0 0 0.250 -105226 deck\_trimline.iges

#### Or,

\*DEFINE\_CURVE\_TRIM\_NEW

\$# tcid tctype tflg tdir tctol toln nseed
 1 2 0 0.250

deck\_trimline.iges

\*define\_trim\_seed\_point\_coordinates

\$ NSEED X1 Y1 Z1 X2 Y2

Z2

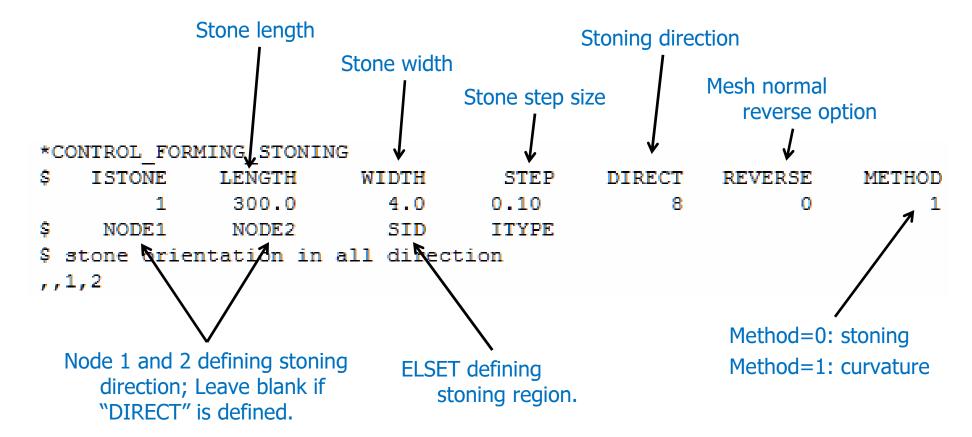
**Trimming** 

1 2026.19 292.148 -134.788









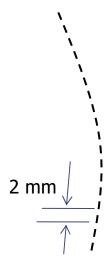
- Smoothed surface for stoning and curvature calculation
- Stoning direction can be manually defined by two nodes, or input # of directions without Node1 and Node2 definition



♦ Stoning method – no mesh refinement 2.0mm element size around

door handle untrimmed - one way curvature.

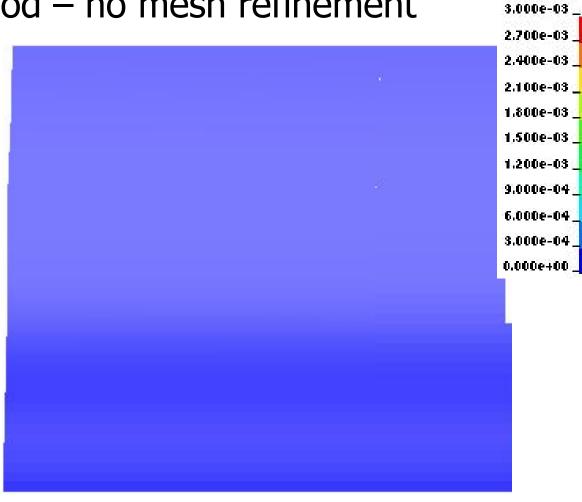
- Mesh built with surface (not splitting from a coarser mesh)
- Results are in the order of 1.0E-04 to 1.0E-05.



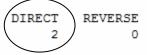
#### **Summary:**

Basically no differentiation in surface lows.



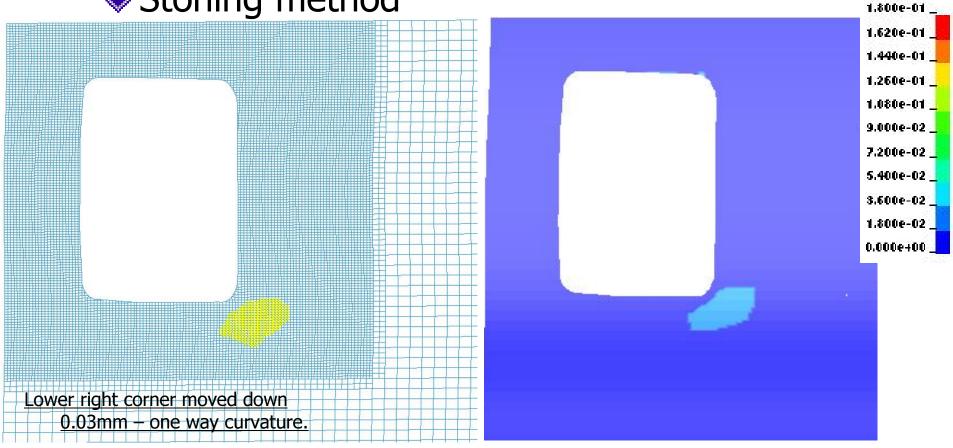


*CONTROL_FORMING_STONING						
\$	ISTONE	LENGTH	WIDTH	STEP		
	1	150.0	4.0	1.00		
Ş	NODE1	NODE2	SID	ITYPE		





Stoning method



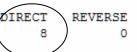
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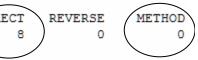
#### **Summary:**

Expected.

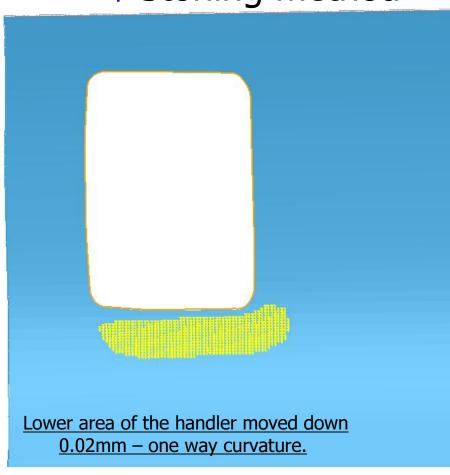


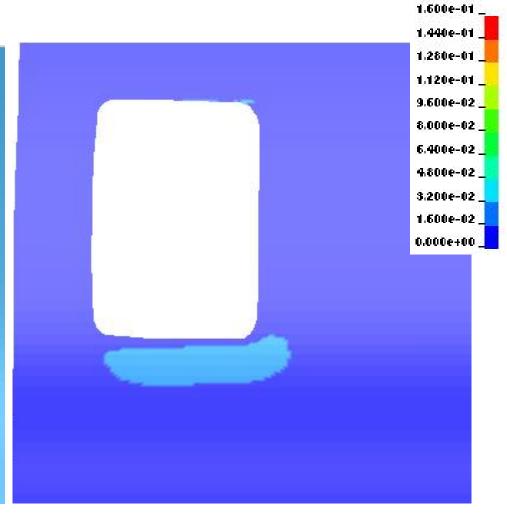
*CONTROL_FORMING_STONING						
\$	ISTONE	LENGTH	WIDTH	STEP		
	1	300.0	4.0	0.10		
Ş	NODE1	NODE2	SID	ITYPE		
ş	stone ori	entation in	all direct	ion		





Stoning method





#### **Summary:**

Expected.



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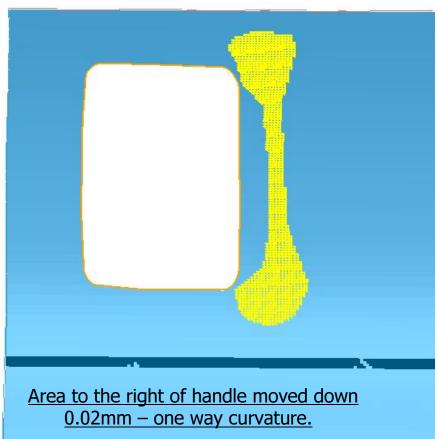
*CONTROL FORMING STONING						
\$	ISTONE	LENGTH	WIDTH	STEP		
	1	300.0	4.0	0.10		
\$	NODE1	NODE2	SID	ITYPE		

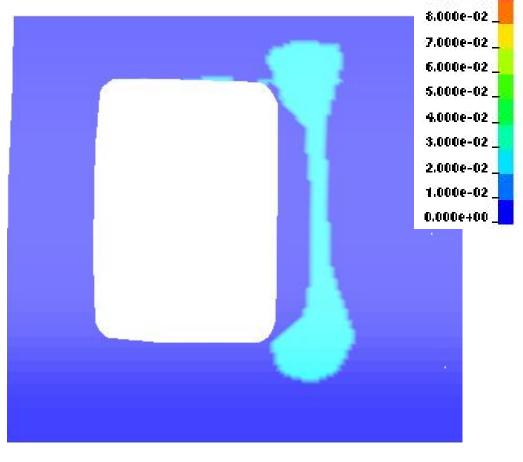
\$ stone orientation in all direction

,,1,2



Stoning method





#### **Summary:**

Expected.



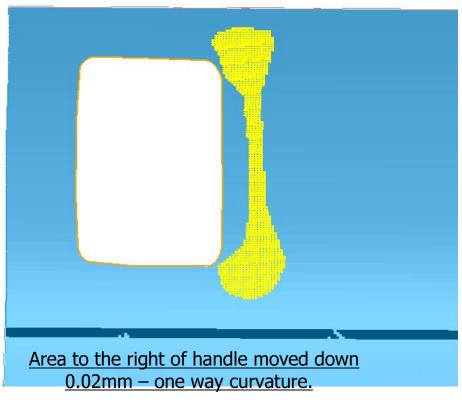
\*CONTROL\_FORMING\_STONING
\$ ISTONE LENGTH WIDTH STEP
1 150.0 4.0 1.00
\$ NODE1 NODE2 SID ITYPE
,,1,2

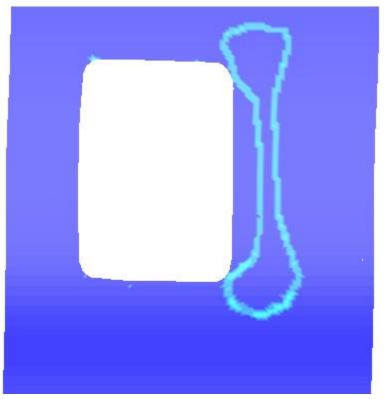


1.000e-01

9.000e-02

Curvature method





### 1.150e-02 \_ 9.400e-03 \_ 7.300e-03 \_ 5.200e-03 \_ 3.100e-03 \_ 1.000e-03 \_

2.200e-02 \_ 1.990e-02 \_ 1.780e-02 \_ 1.570e-02 \_ 1.360e-02

#### **Summary:**

Captures the boundary of the surface low area. Inside of the boundary no curvature change. Expected.



*0	CONTROL_FORM	ING_STONIN	1G	
Ş	ISTONE	LENGTH	WIDTH	STEP
	1	150.0	4.0	1.00
Ş	NODE1	NODE2	SID	ITYPE
Ş	stone orien	tation in	all direct	ion

,,1,2





### Conclusions

- LSDYNA's Implicit capability becomes even more robust and efficient
- LSDYNA can continue maintains technical leader in sheet stamping simulation