

North American LS-DYNA User Forum 2023, November 16, 2023

Stress state dependent regularization

Dr. Filipe Andrade, Dr. Tobias Erhart
DYNAmore GmbH, an Ansys company

Dr. Markus Feucht
Mercedes-Benz Cars AG

Introduction

The issue of spurious mesh dependence, regularization

Mesh dependence

Different types

The expression “mesh dependence” is somewhat vague and can as such have different interpretations. Therefore, it is important to highlight the main differences between the typical interpretations of this term.

Geometrical mesh dependence

- A consequence of discretization using finite elements
- May affect solution under any loading (purely elastic, plastic, etc.)
- Generally converging when mesh is fine enough → **can be solved by refining or higher order elements**
- Shells and solids affected in a similar way

“Spurious” mesh dependence

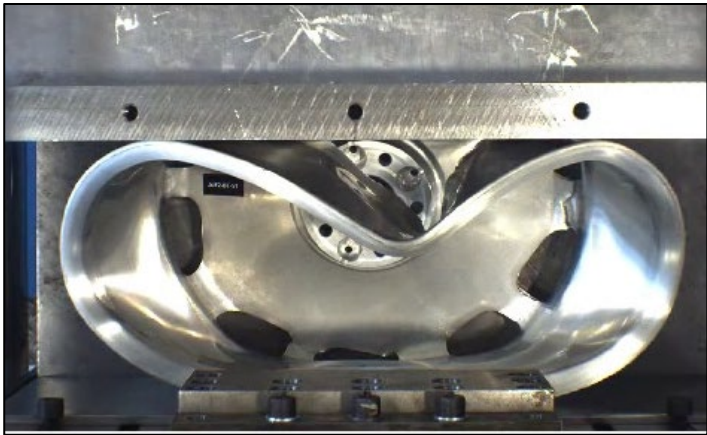
- A consequence of local continuum mechanics
- Only affects solution under certain conditions (e.g., after the necking point under a uniaxial stress state)
- Generally non-converging regardless how fine the mesh is → **cannot be solved by refining**
- Shells generally exhibit more spurious mesh dependence than solids

Regularization strategies are intended to tackle the **spurious kind of mesh dependence**

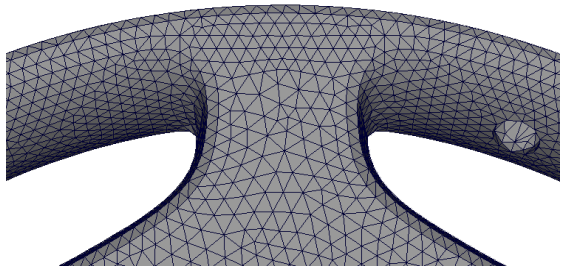
Ideally, only geometrically converged models should be regularized

Example: Geometrical mesh dependence

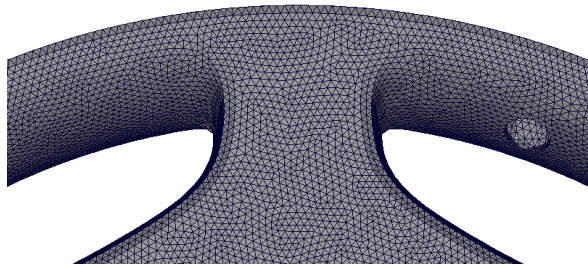
Wheel simulation



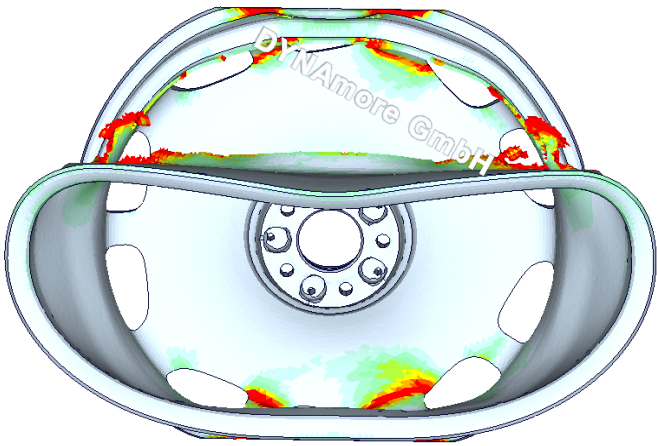
5.0mm



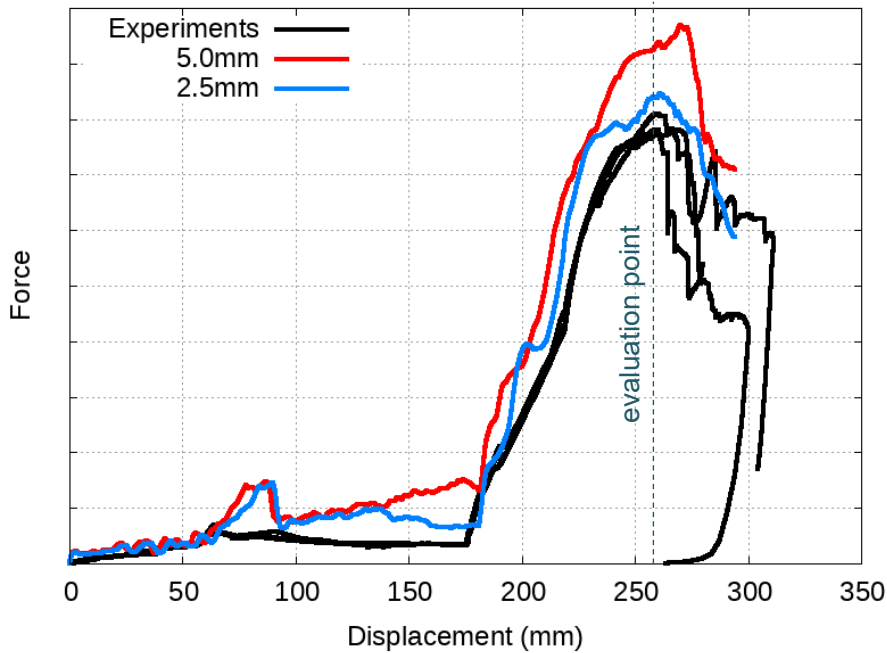
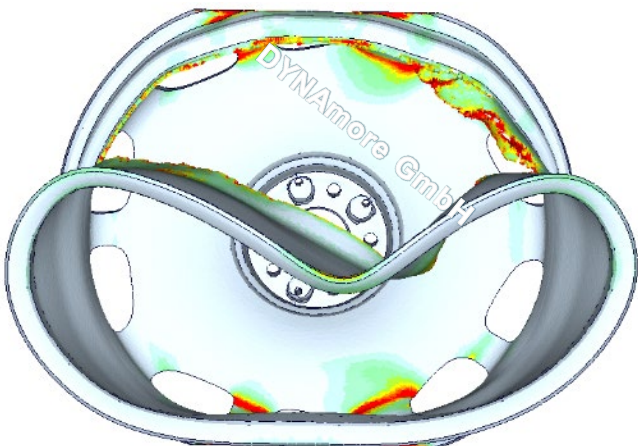
2.5mm



5.0mm

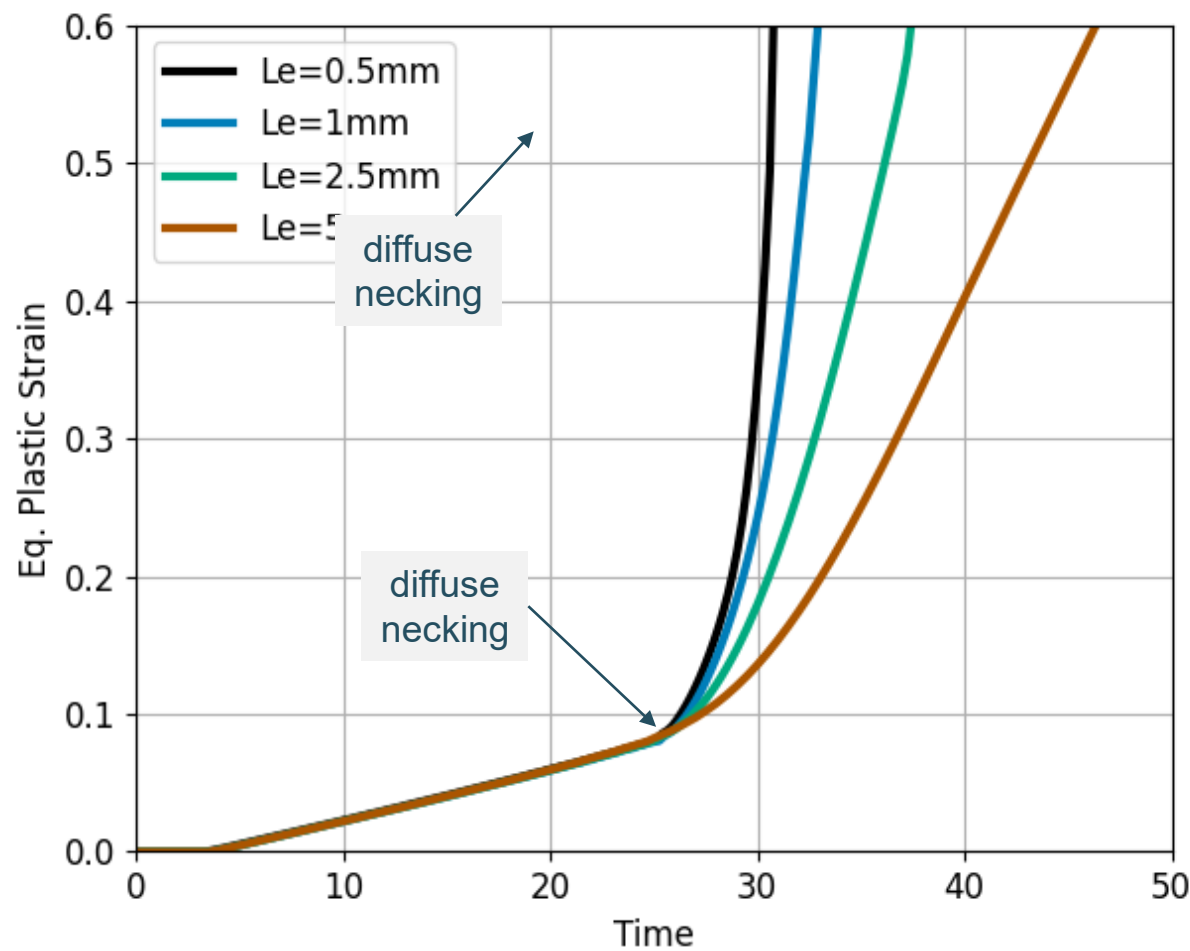
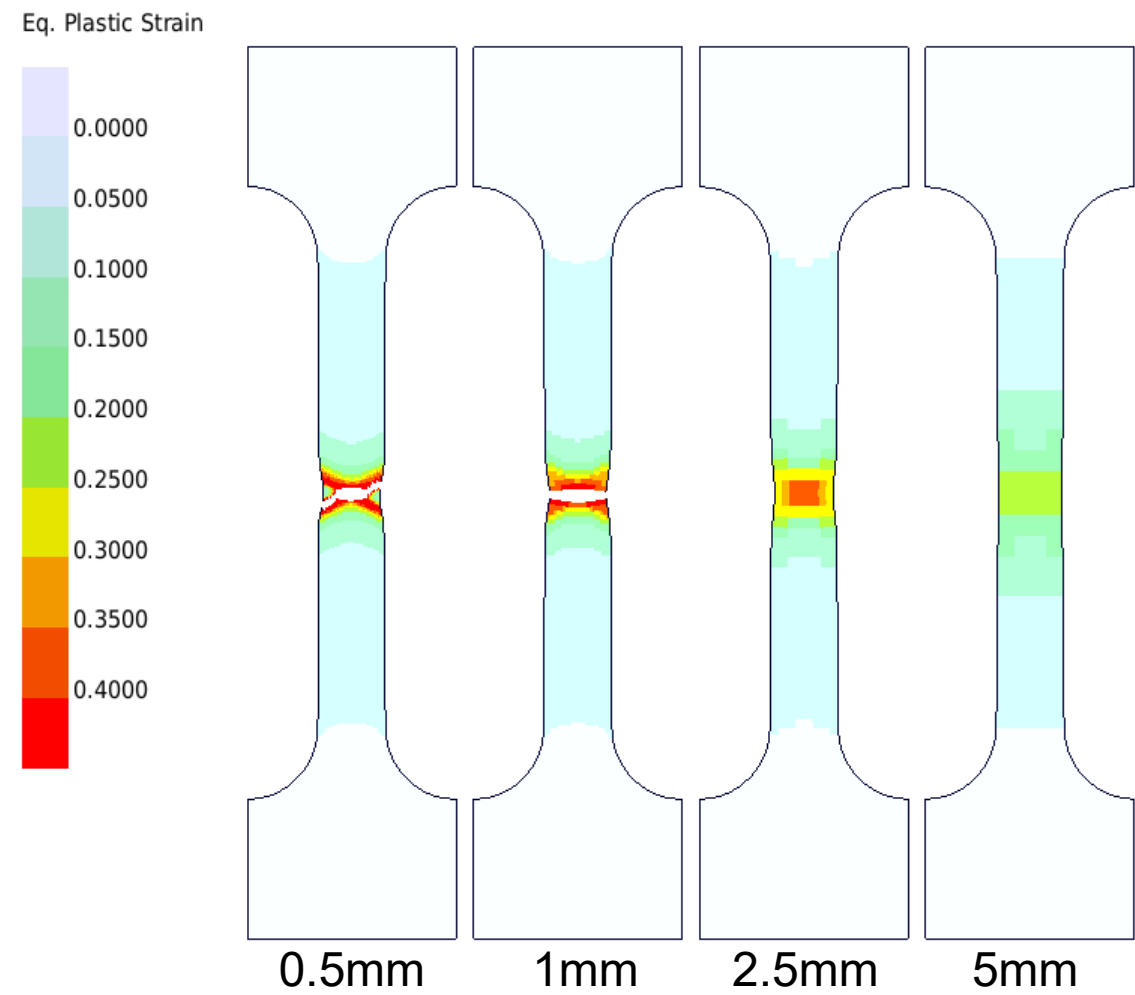


2.5mm



Onset of spurious mesh dependence

Reference: Quasi-static tensile test with different element sizes
Triaxiality 1/3 up to necking point

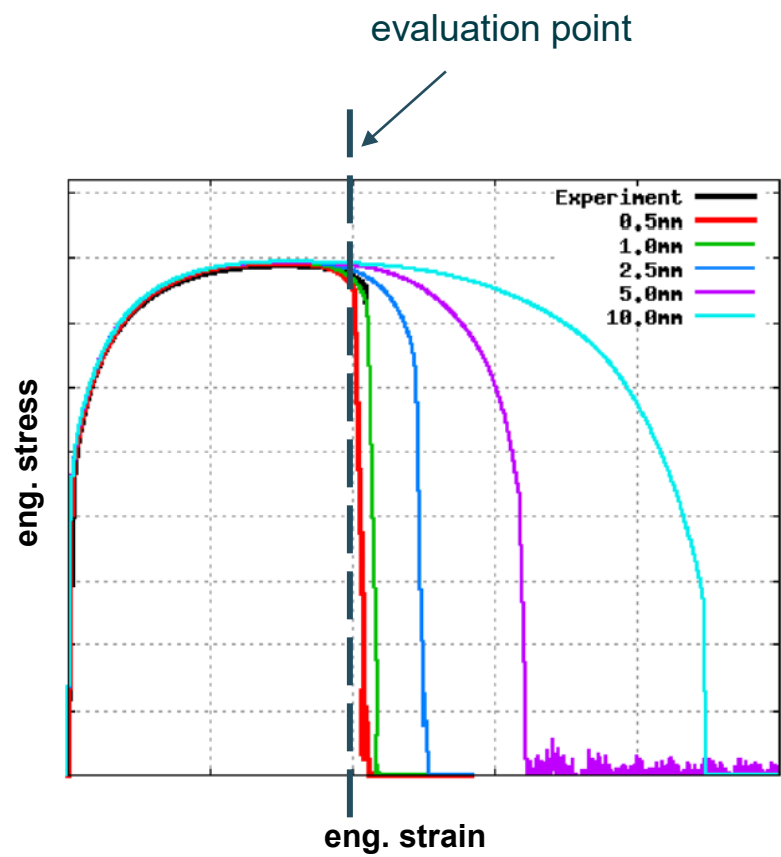


*MAT_024 + GISSMO (without regularization), monotonic hardening curve, no strain rate dependence

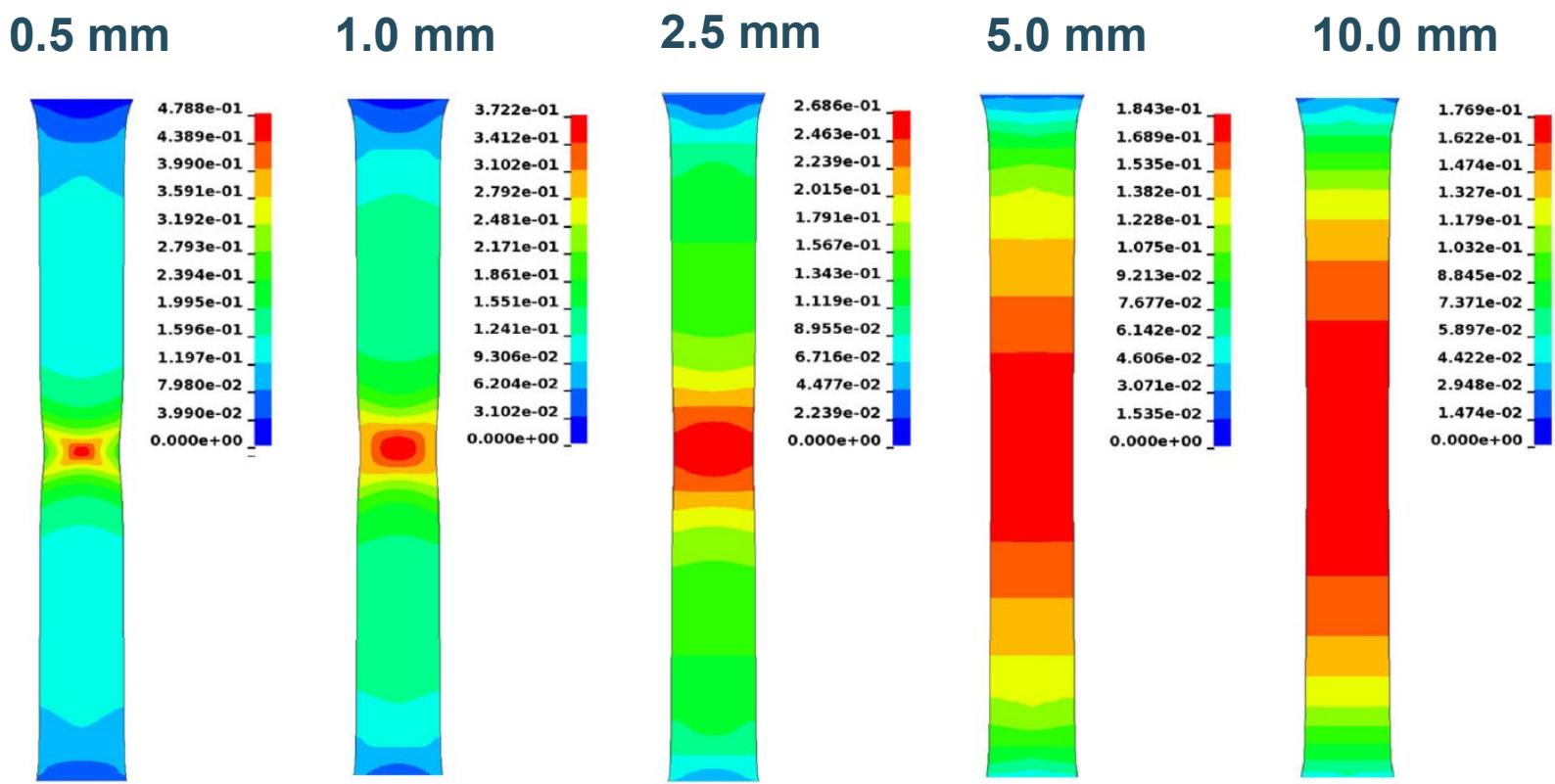
Example: Spurious mesh dependence

Simulation of a large tensile test, evaluation at the same deformation point

Different plastic strain distribution for each mesh size, failure takes place at different deformation points.

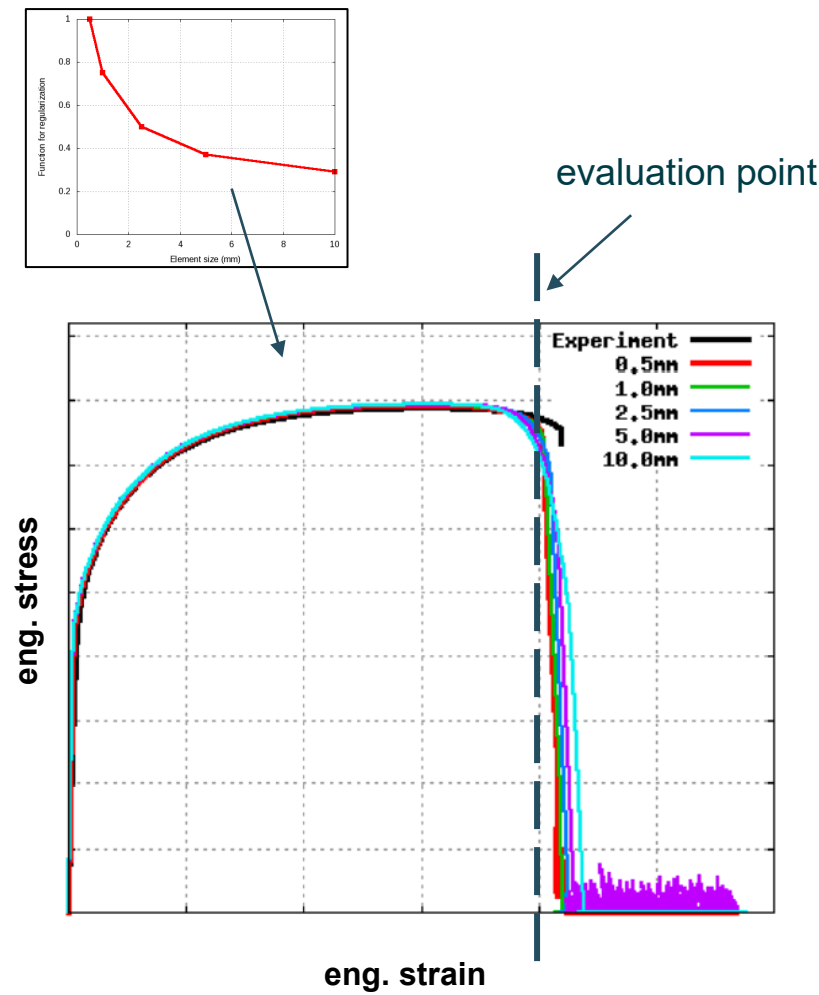


*MAT_024 + GISSMO (without regularization),
monotonic hardening curve, no strain rate dependence



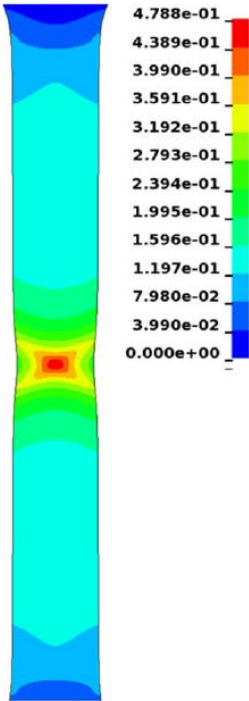
Example: Spurious mesh dependence

Simulation of a large tensile test after regularization (no stress-damage coupling)

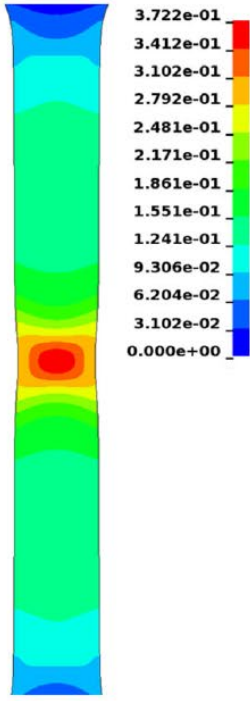


No change of plastic strain distribution, however, the deformation point at which failure occurs is now identical

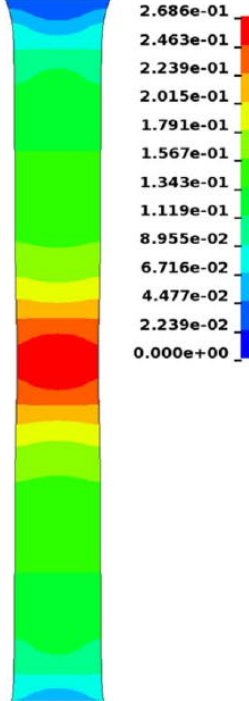
0.5 mm



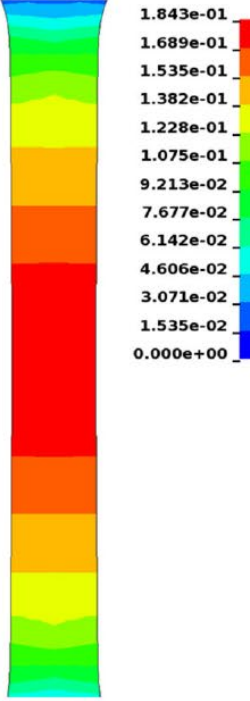
1.0 mm



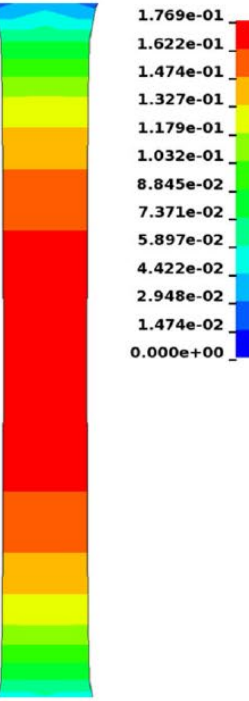
2.5 mm



5.0 mm



10.0 mm



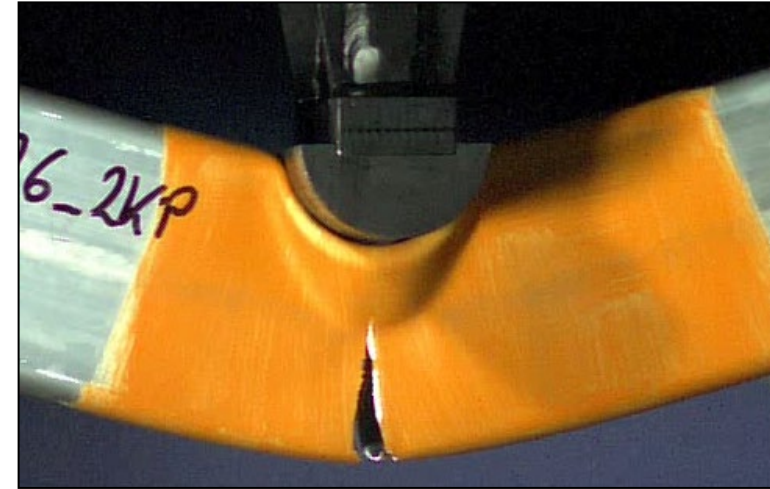
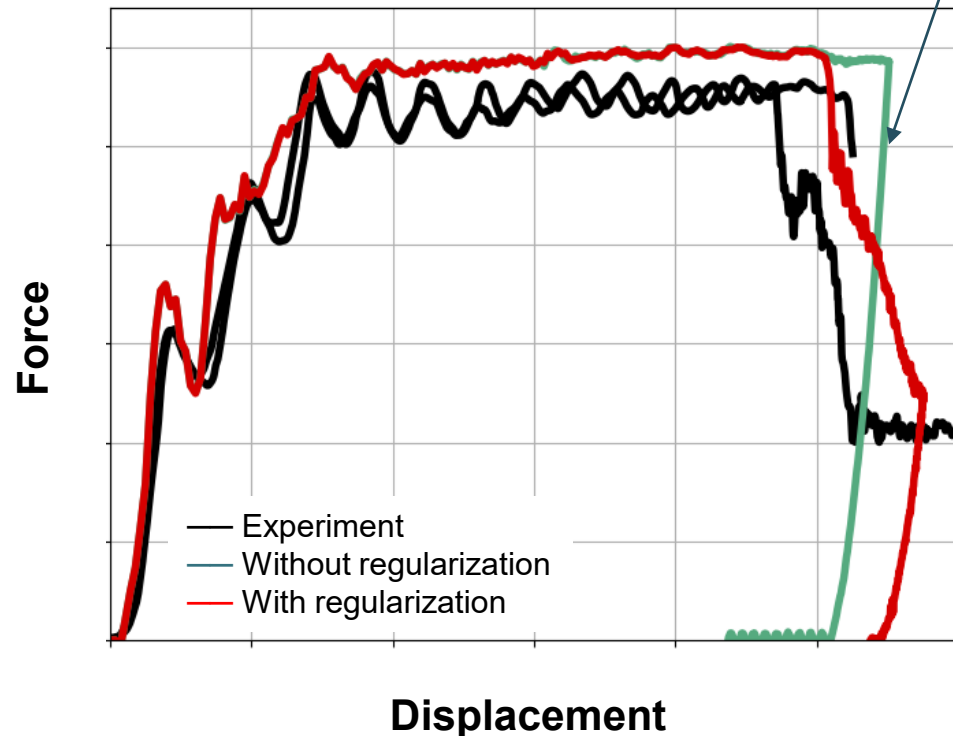
*MAT_024 + GISSMO (with regularization),
monotonic hardening curve, no strain rate dependence

Effect on component test simulation

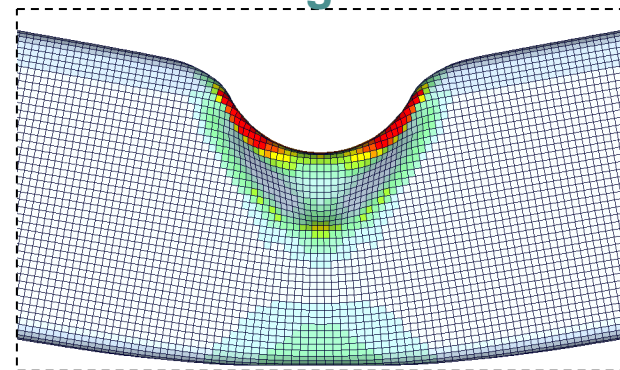
With and without regularization, shell elements, element size 3 mm

*MAT_024 + GISSMO, aluminum extrusion

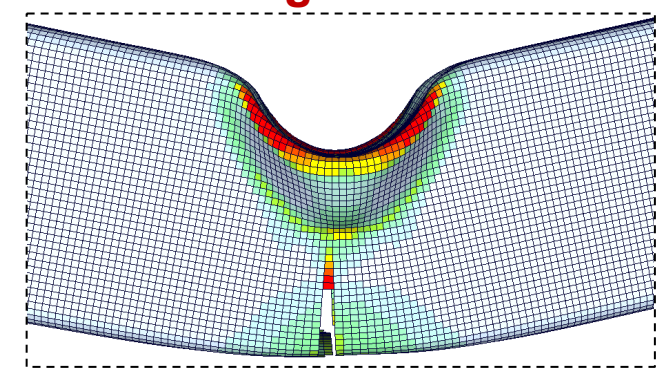
Elastic rebound is observed when running the simulation without regularization because **no element failure** has taken place



without regularization



with regularization



Remarks

Spurious mesh dependence and regularization

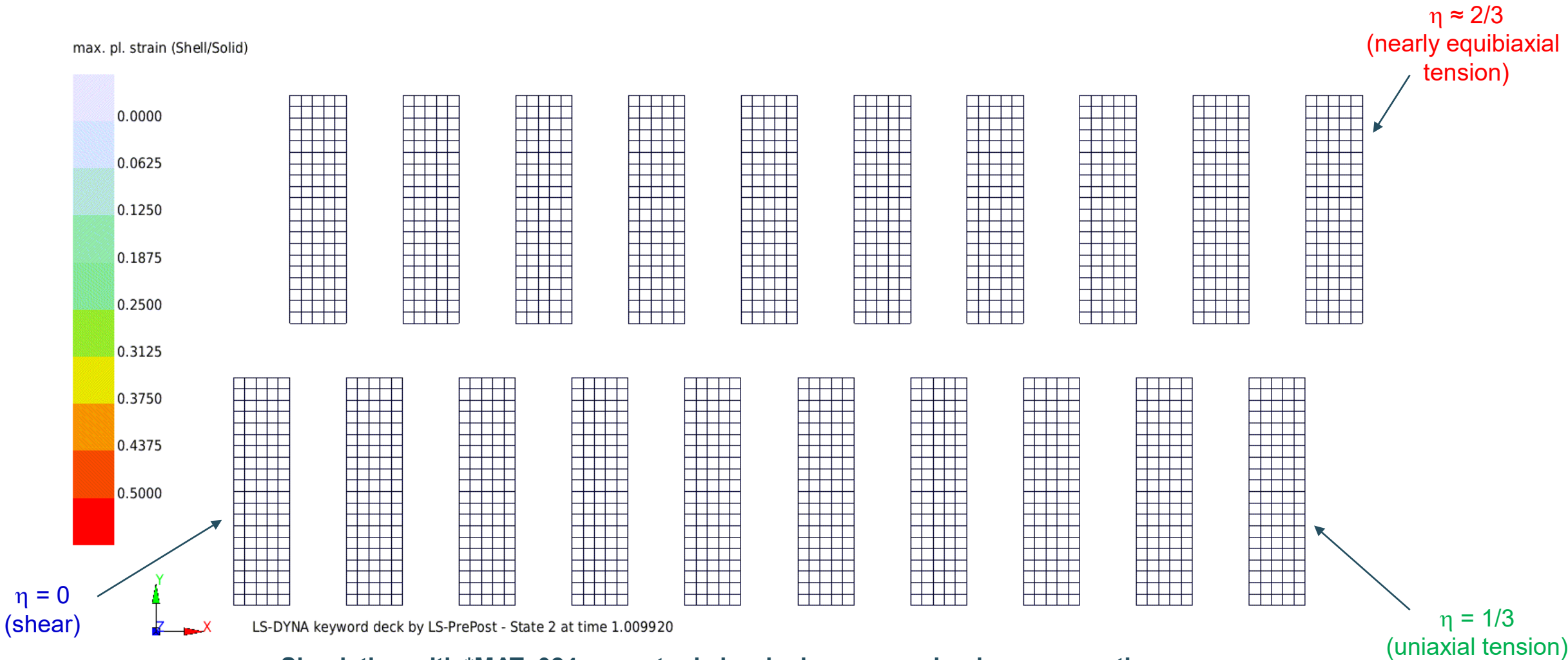
- Simple regularization strategies do not solve the underlying problem of spurious mesh dependence but compensate for its effects
- **Regularization will not help against geometrical mesh dependence!**
- Spurious mesh dependence is dependent on the stress state

Spurious mesh dependence

Onset depends on the stress state

Plane stress (shell elements)

Simulation of element blocks (width=5mm, height=20mm, element size=1.0mm)

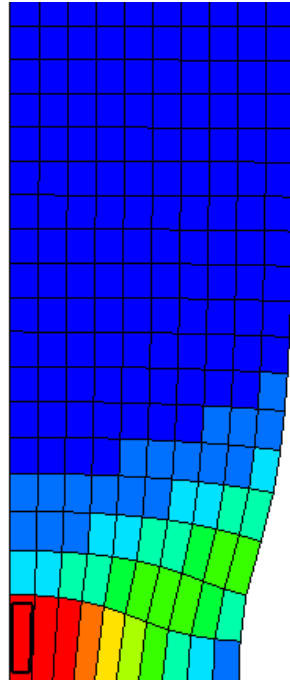
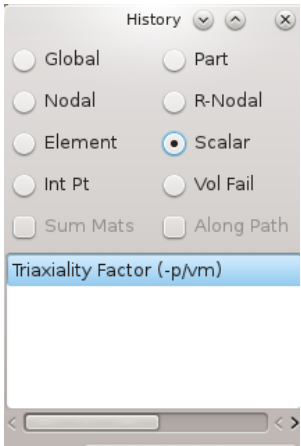
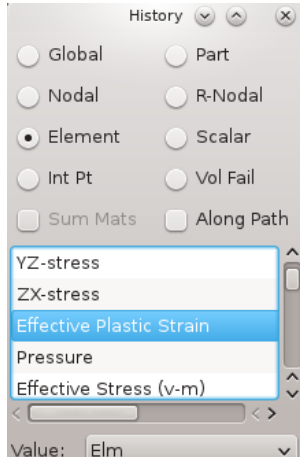


**Simulation with *MAT_024, monotonic hardening curve, aluminum properties,
failure strain = 2.0 for all triaxialities**

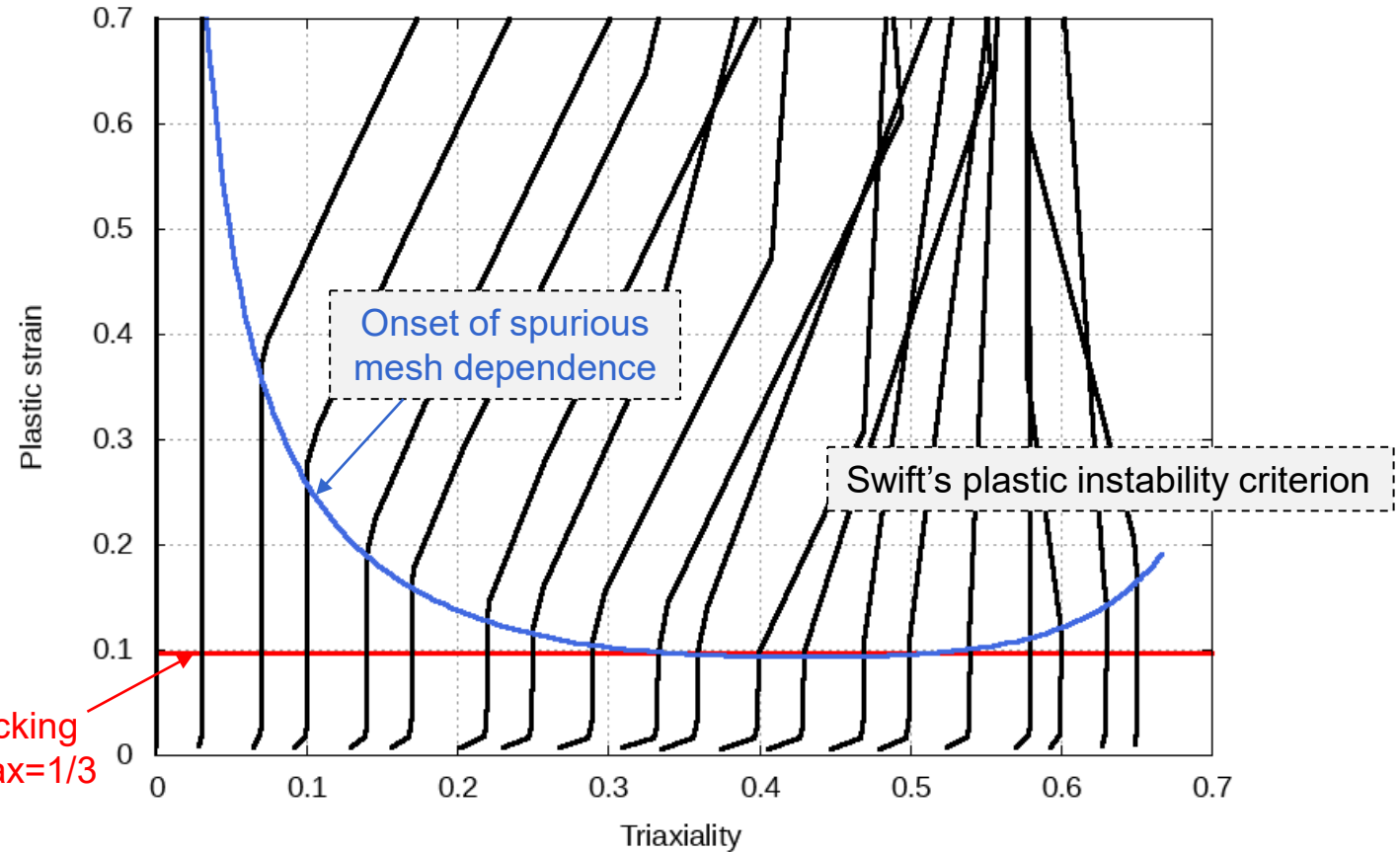
Plane stress (shell elements)

Strain-triaxiality paths (width=5mm, height=40mm, element size=0.5mm)

Aluminum extrusion



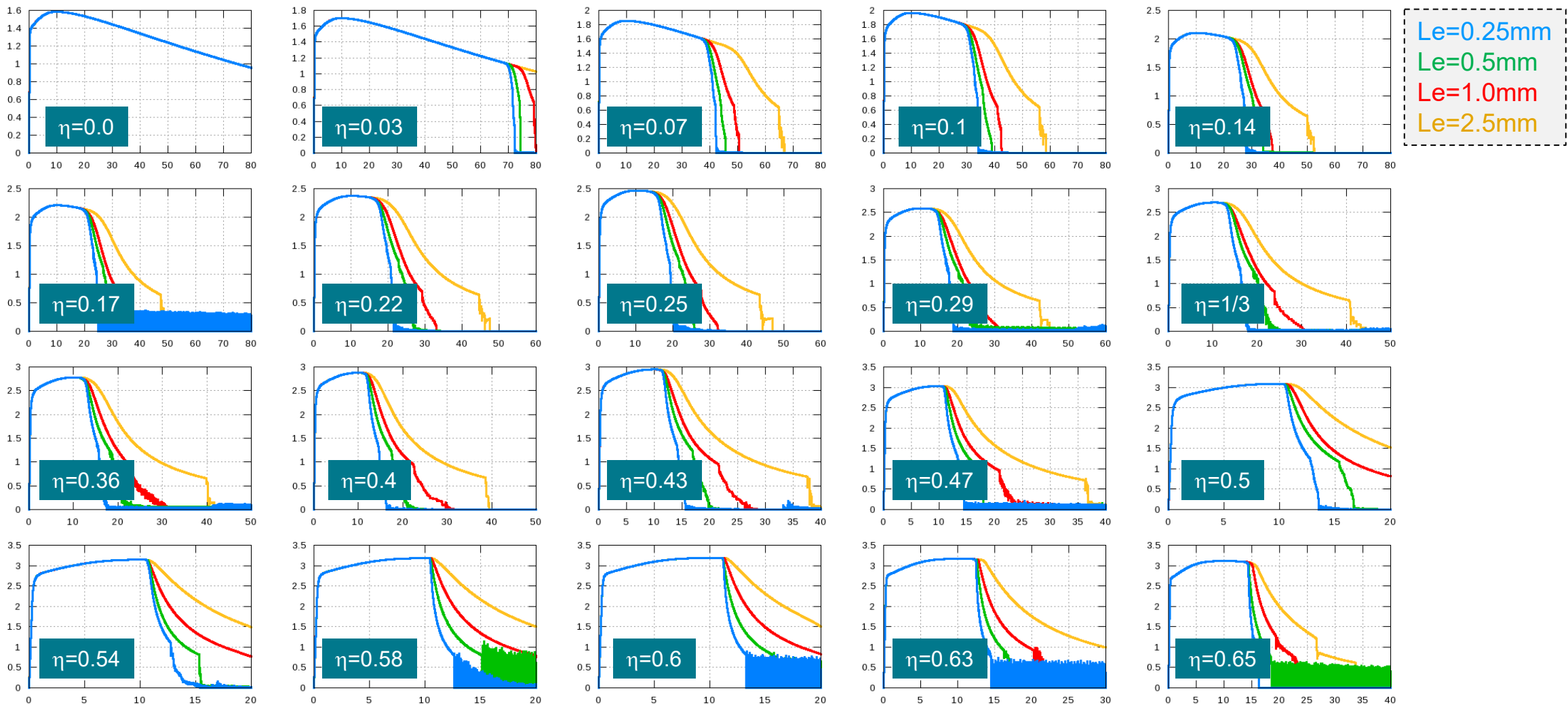
diffuse necking
strain at triax=1/3



DuBois, Feucht, Andrade, Graf, Conde, Haufe. Instability and Mesh Dependence – Part I.
Andrade, Feucht, DuBois, Graf, Conde, Haufe. Instability and Mesh Dependence – Part II.
16th LS-DYNA Forum, Bamberg, 2022.

Plane stress (shell elements)

Vertical reaction force vs. time (width=5mm, height=40mm)



Simulation with *MAT_024, monotonic hardening curve, aluminum properties, failure strain = 2.0 for all triaxialities

Regularization in LS-DYNA

*MAT_ADD_DAMAGE_GISSMO

*MAT_ADD_DAMAGE_GISSMO

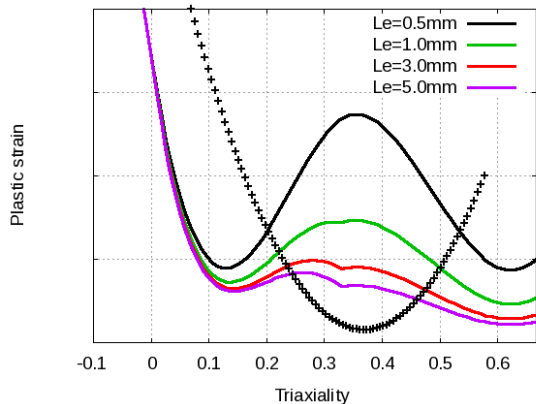
SHRF and BIAXF

*MAT_ADD_DAMAGE_GISSMO

| | | | | | | |
|----|-------|-------|--------|--------|---------|--------|
| \$ | MID | | DTYP | REFSZ | NUMFIP | |
| | 10 | | 1 | | -80 | |
| \$ | LCSDG | ECRIT | DMGEXP | DCRIT | FADEXP | LCREGD |
| | 100 | -200 | 2 | | 2.5 | 502 |
| \$ | LCSRS | SHRF | BIAXF | LCDLIM | MIDFAIL | HISVN |
| | | 1.0 | 0.0 | | | |

*DEFINE_CURVE

```
$ Regularisierung Girlande, triax = 0.3333333
$ lcid      sidr      sola      sclo      offa
$          502        0        1.0        1.0
$          x          y
$          0.5000      1.0000
$          1.0000      0.5350
$          2.5000      0.3500
$          5.0000      0.2500
$          10.0000     0.1800
```



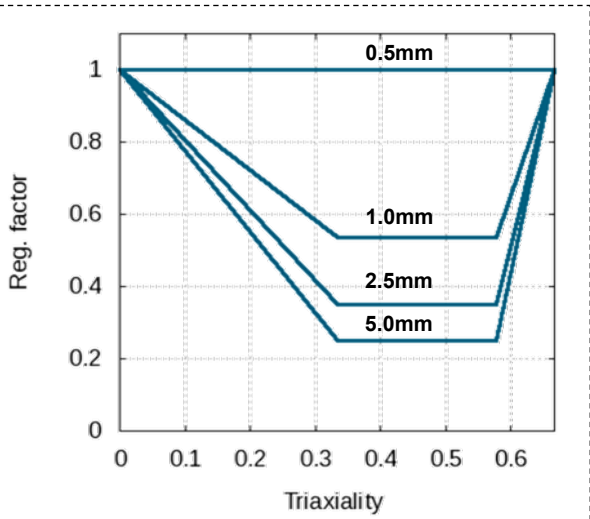
TABLE

*MAT_ADD_DAMAGE_GISSMO

| | | | | | | |
|----|-------|-------|--------|--------|---------|--------|
| \$ | MID | | DTYP | REFSZ | NUMFIP | |
| | 10 | | 1 | | -80 | |
| \$ | LCSDG | ECRIT | DMGEXP | DCRIT | FADEXP | LCREGD |
| | 100 | -200 | 2 | | 2.5 | -500 |
| \$ | LCSRS | SHRF | BIAXF | LCDLIM | MIDFAIL | HISVN |
| | | | | | | |

*DEFINE_TABLE

```
$ tbid
$ 500
$ triax
$ 0.0000000
$ 0.3333333
$ 0.5773500
$ 0.6666666
*DEFINE_CURVE
$ Regularisierung Girlande, triax = 0.0
$ lcid      sidr      sola      sclo      offa
$          501        0        1.0        1.0
$          x          y
$          0.5000      1.0000
$          1.0000      1.0000
$          2.5000      1.0000
$          5.0000      1.0000
$          10.0000     1.0000
*DEFINE_CURVE
$ Regularisierung Girlande, triax = 0.3333333
$ lcid      sidr      sola      sclo      offa
$          502        0        1.0        1.0
$          x          y
$          0.5000      1.0000
$          1.0000      0.5350
$          2.5000      0.3500
$          5.0000      0.2500
$          10.0000     0.1800
*DEFINE_CURVE
$ Regularisierung Girlande, triax = 0.57735
$ lcid      sidr      sola      sclo      offa
$          503        0        1.0        1.0
$          x          y
$          0.5000      1.0000
$          1.0000      0.5350
$          2.5000      0.3500
$          5.0000      0.2500
$          10.0000     0.1800
*DEFINE_CURVE
$ Regularisierung Girlande, triax = 0.6666666
$ lcid      sidr      sola      sclo      offa
$          504        0        1.0        1.0
$          x          y
$          0.5000      1.0000
$          1.0000      1.0000
$          2.5000      1.0000
$          5.0000      1.0000
$          10.0000     1.0000
```

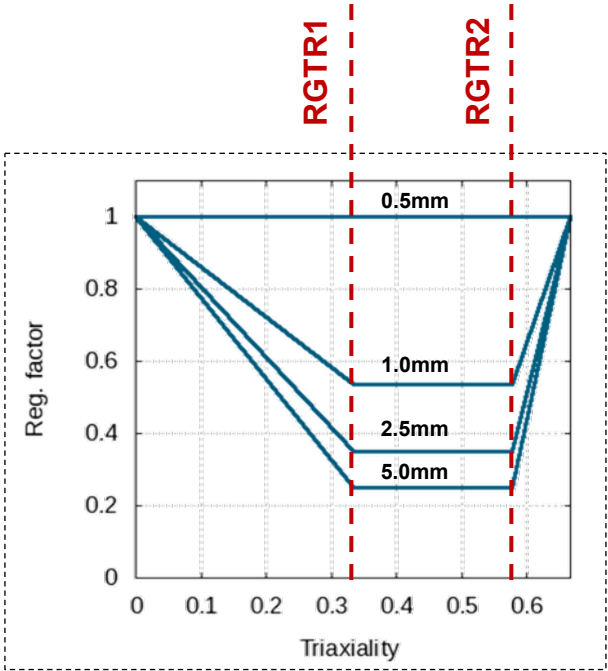


*MAT_ADD_DAMAGE_GISSMO

New feature! Available in current development versions

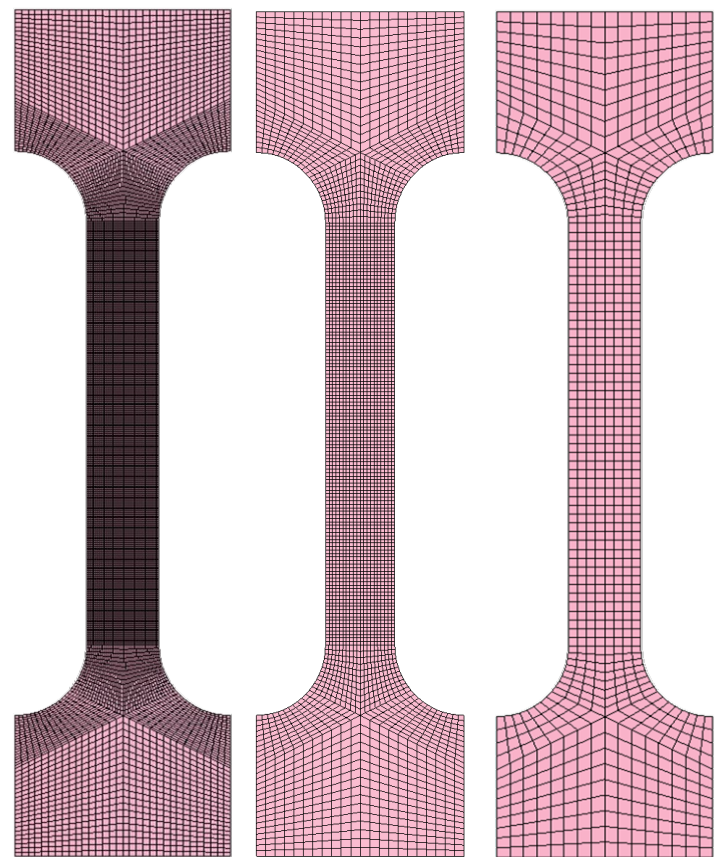
```
*MAT_ADD_DAMAGE_GISSMO
$ MID DTYP REFSZ NUMFIP
  10 1 -80
$ LCSDG ECRIT DMGEXP DCRIT FADEXP LCREGD
  100 -200 2 2.5 502
$ LCSRS SHRF BIAXF LCDLIM MIDFAIL HISVN
$ RGTR1 RGTR2
  0.33333 0.57735
```

```
*DEFINE_CURVE
$ Regularisierung Girlande, triax = 0,333333
$ lcid sidr scla sclo offa
  502 0 1,0 1,0
$      x y
      0,5000 1,0000
      1,0000 0,5350
      2,5000 0,3500
      5,0000 0,2500
     10,0000 0,1800
```

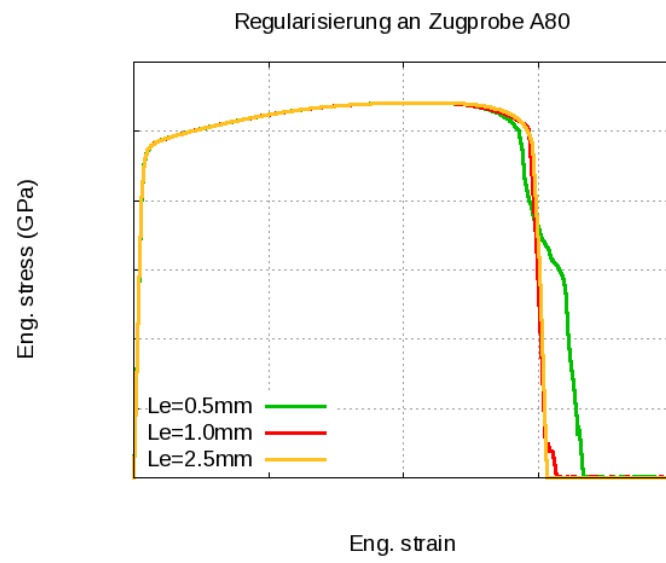
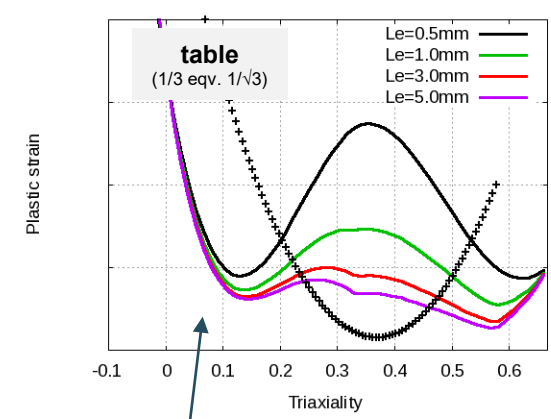
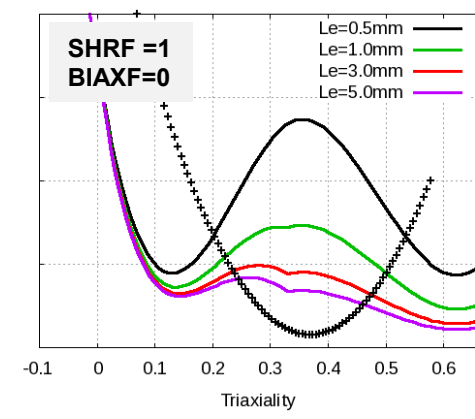
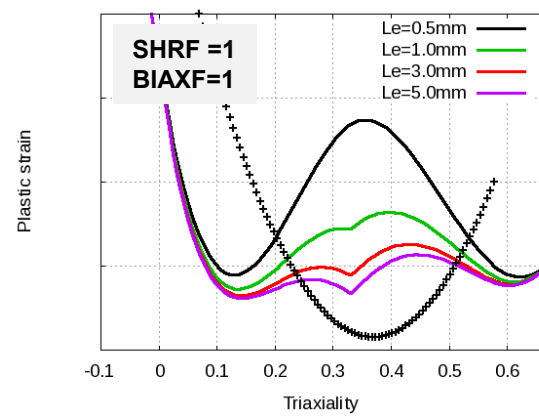


Example: Aluminum

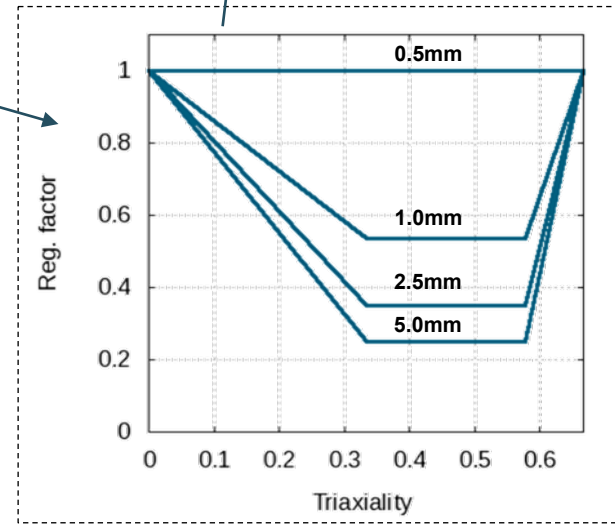
GISSMO material card, regularization factors from A80 tensile test



0.5mm (Reference) 1.0mm 2.5mm

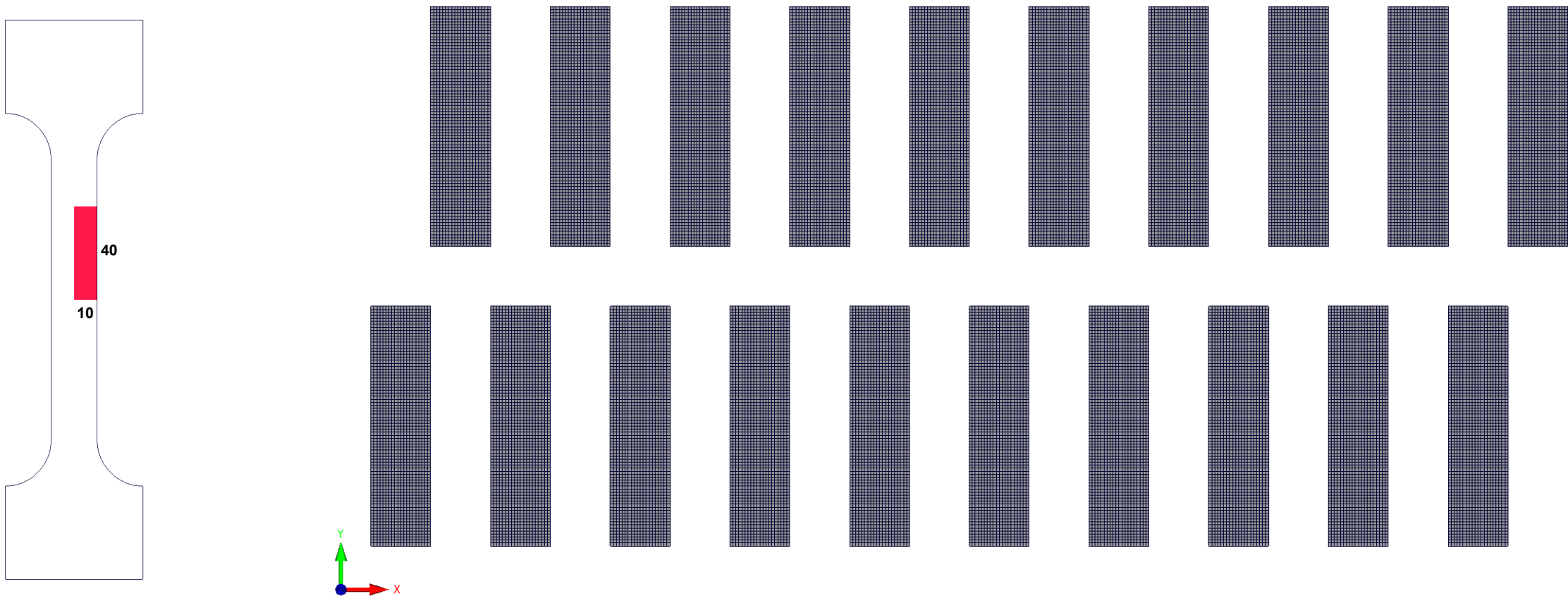


```
*DEFINE_TABLE
$
tbid
500
$
      triax
      0,0000000
      0,3333333
      0,5773500
      0,6666666
*DEFINE_CURVE
$ Regularisierung Girlande, triax = 0,0
$
lcid      sidr      scla      sclo      offa
501      0          1,0      1,0      0
$
      x          y
      0,5000    1,0000
      1,0000    1,0000
      2,5000    1,0000
      5,0000    1,0000
      10,0000   1,0000
*DEFINE_CURVE
$ Regularisierung Girlande, triax = 0,3333333
$
lcid      sidr      scla      sclo      offa
502      0          1,0      1,0      0
$
      x          y
      0,5000    1,0000
      1,0000    0,5350
      2,5000    0,3500
      5,0000    0,2500
      10,0000   0,1800
*DEFINE_CURVE
$ Regularisierung Girlande, triax = 0,57735
$
lcid      sidr      scla      sclo      offa
503      0          1,0      1,0      0
$
      x          y
      0,5000    1,0000
      1,0000    0,5350
      2,5000    0,3500
      5,0000    0,2500
      10,0000   0,1800
*DEFINE_CURVE
$ Regularisierung Girlande, triax = 0,6666666
$
lcid      sidr      scla      sclo      offa
504      0          1,0      1,0      0
$
      x          y
      0,5000    1,0000
      1,0000    1,0000
      2,5000    1,0000
      5,0000    1,0000
      10,0000   1,0000
```



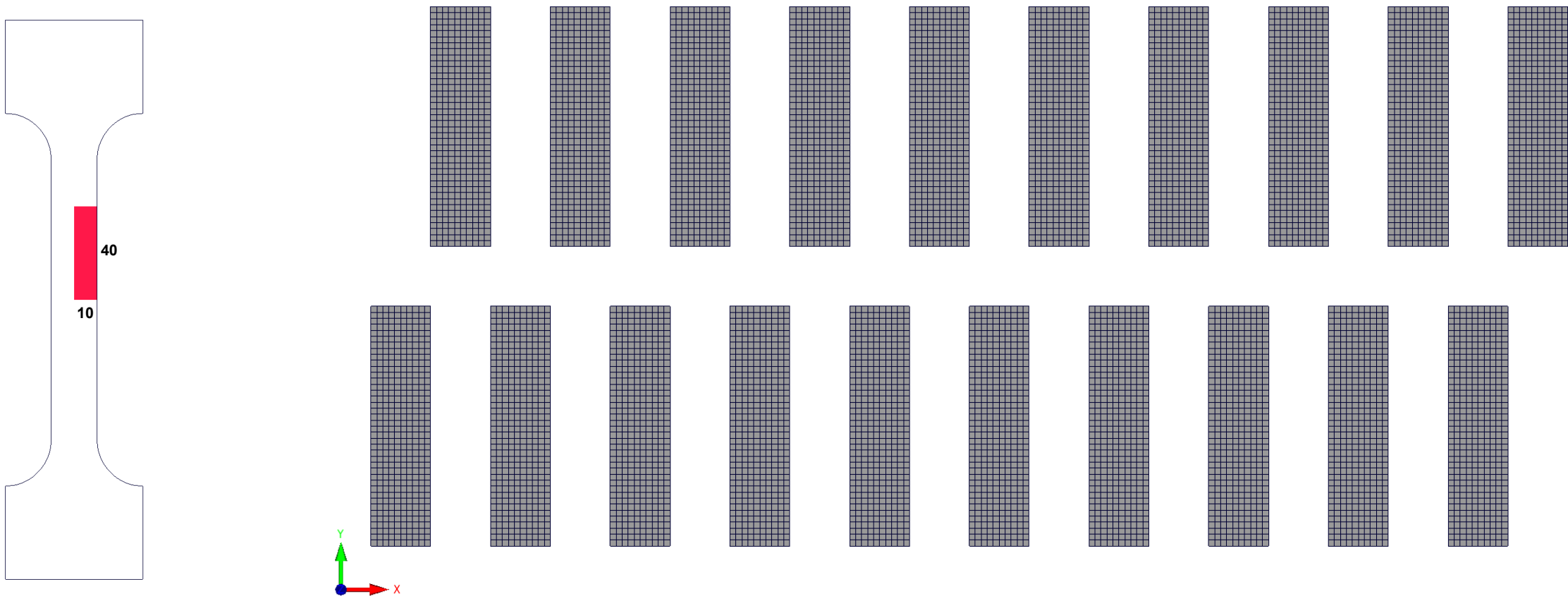
From A80 tensile test to blocks

Le=0.5mm



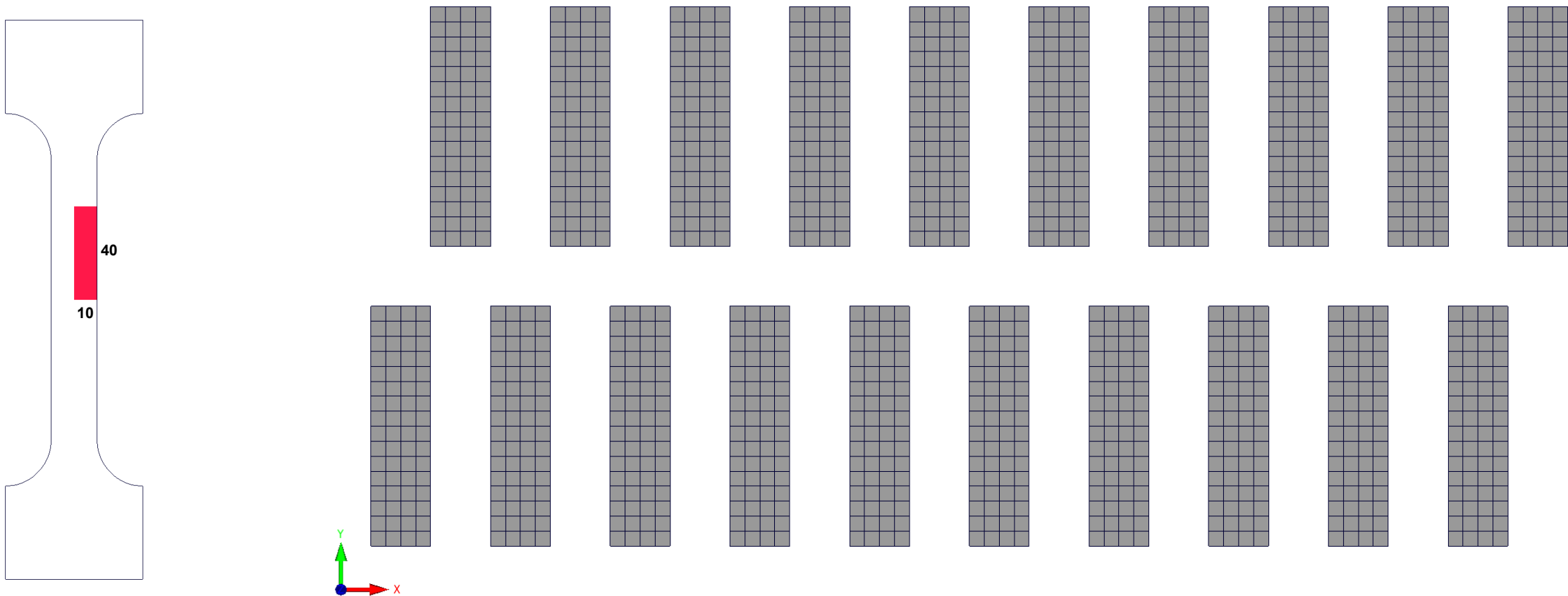
From A80 tensile test to blocks

Le=1.0mm



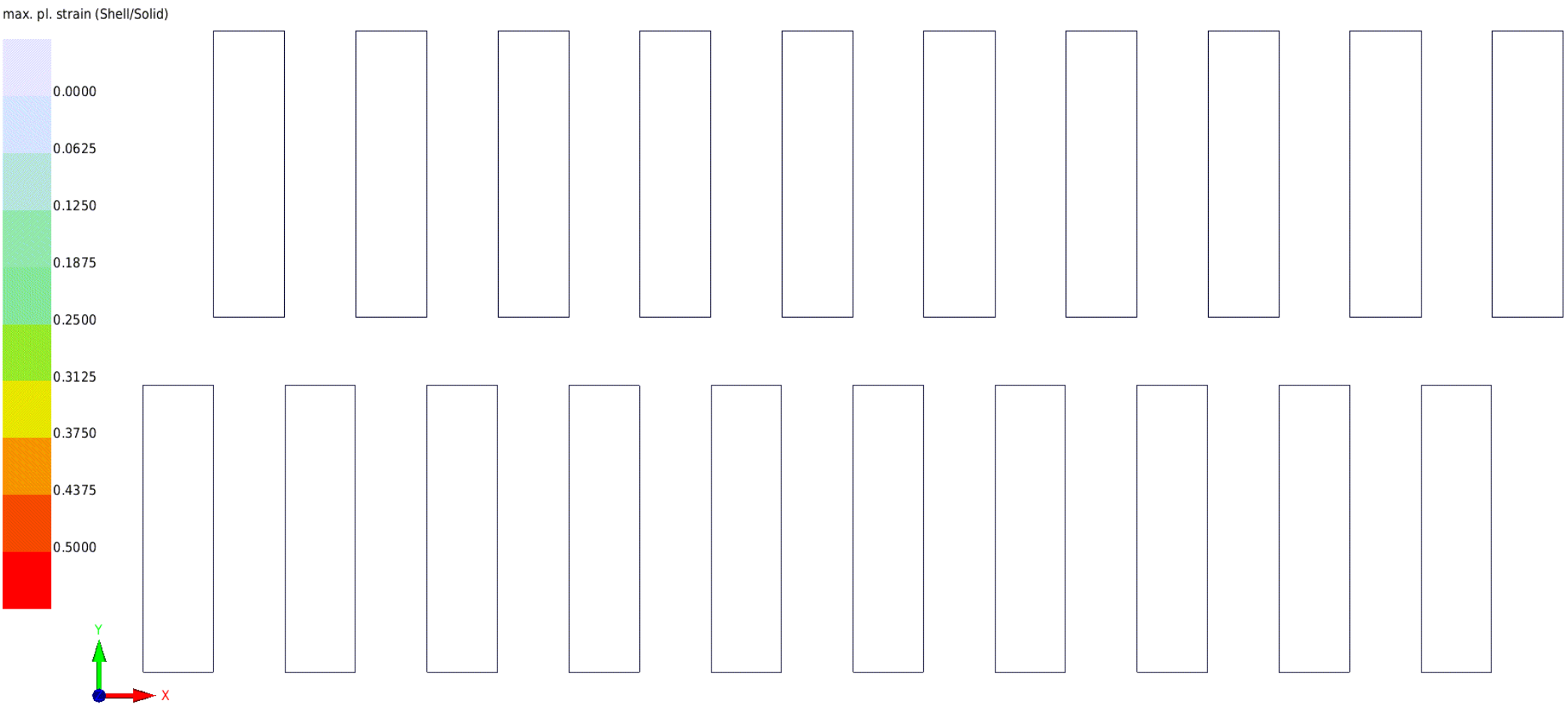
From A80 tensile test to blocks

Le=2.5mm



From A80 tensile test to blocks

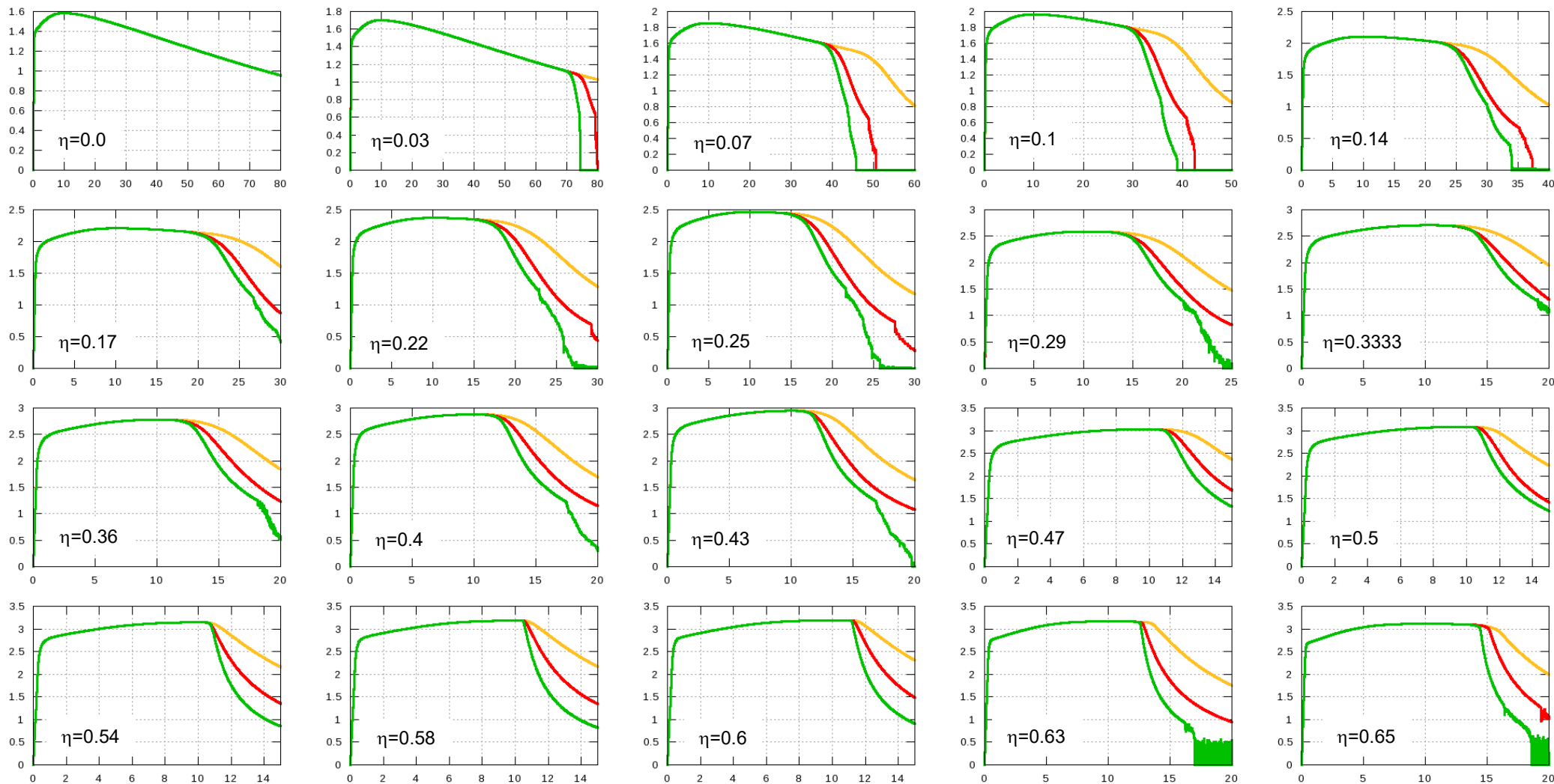
Deformation up to fracture



Y-force vs time

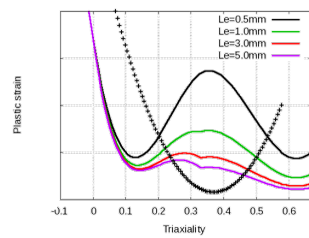
No regularization

Le=0.5mm
Le=1.0mm
Le=2.5mm

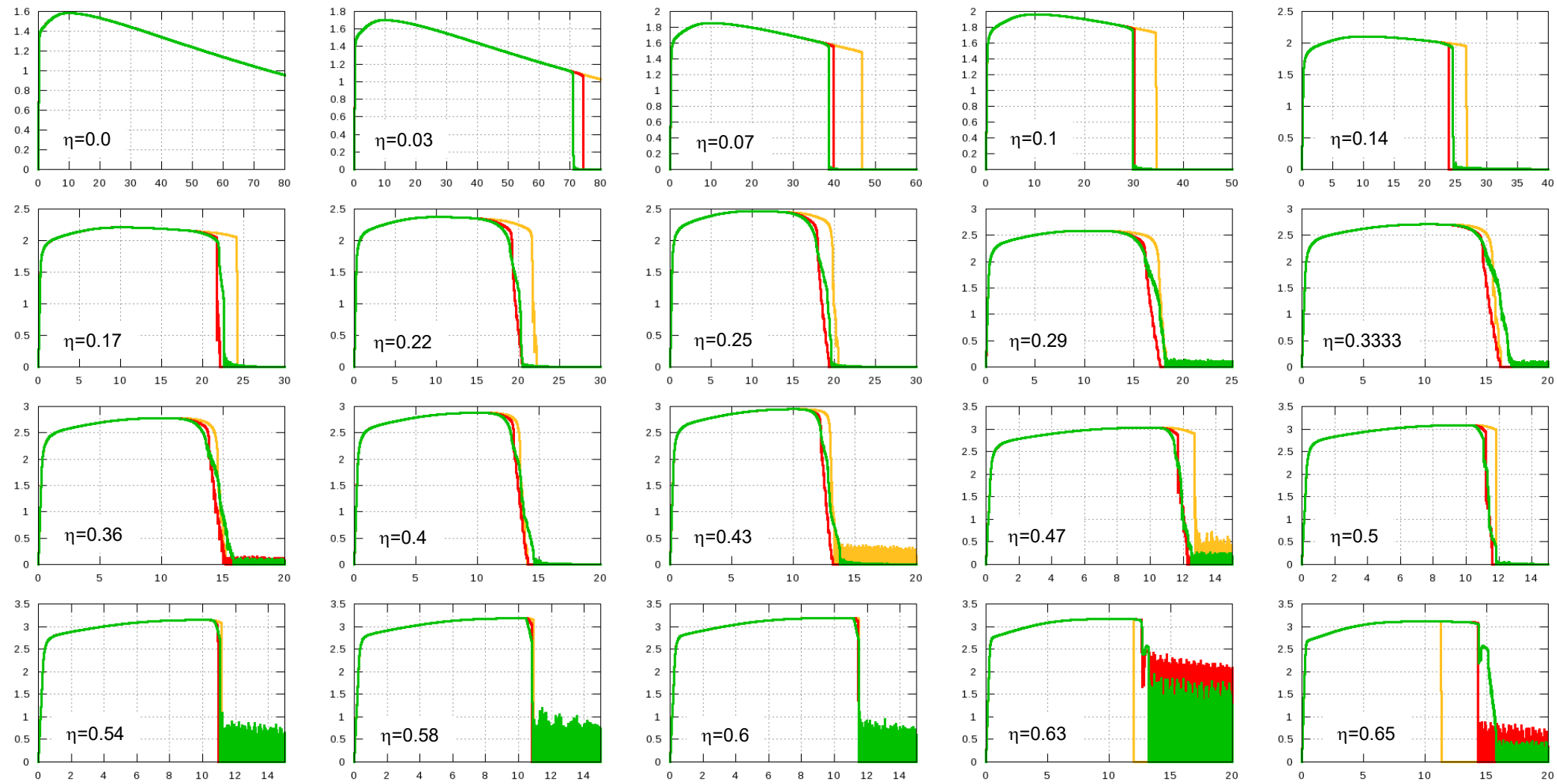


Y-force vs time

SHRF=1, BIAXF=0

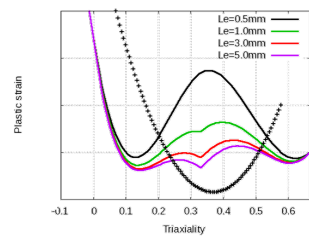


Le=0.5mm
Le=1.0mm
Le=2.5mm

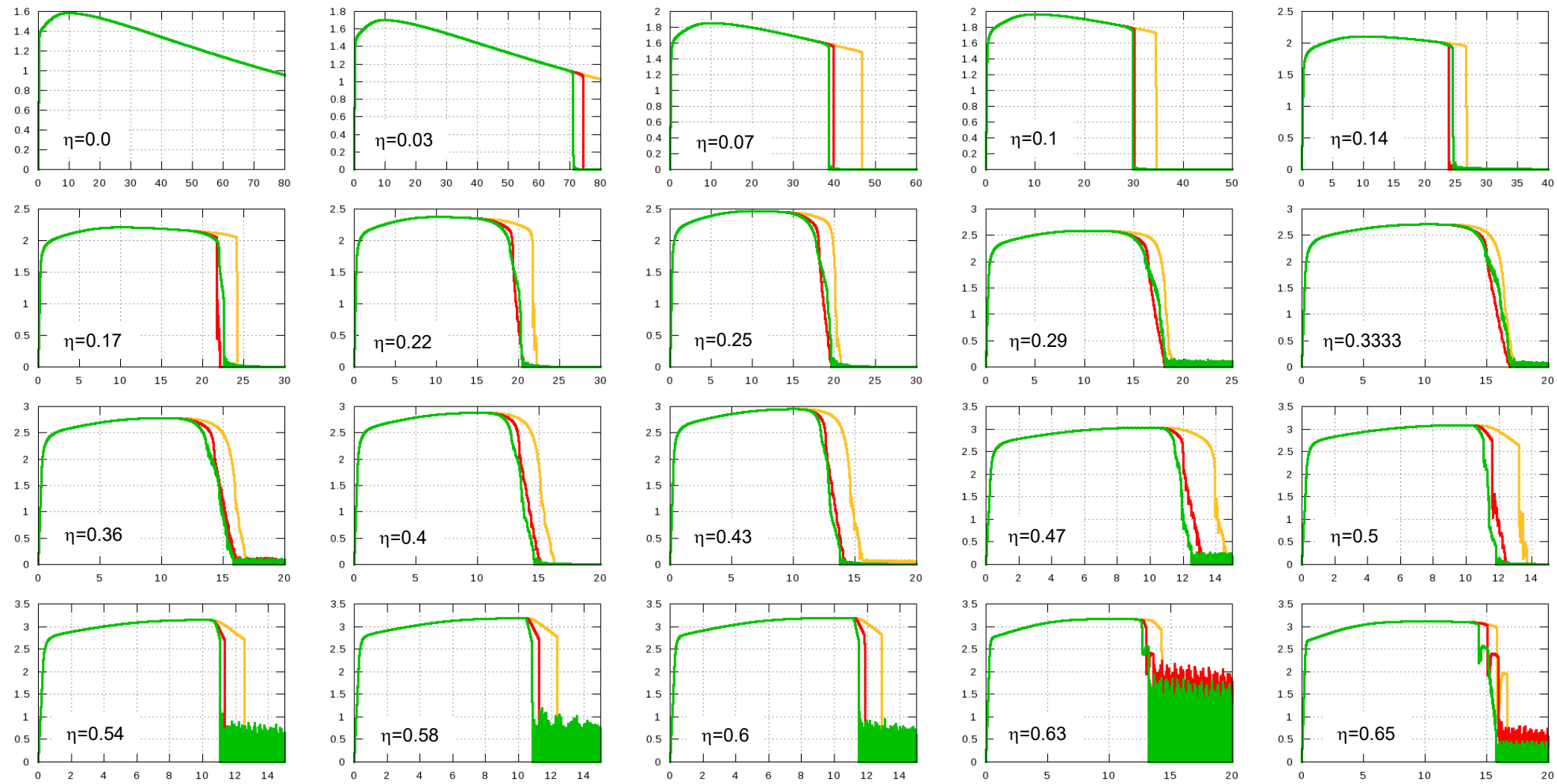


Y-force vs time

SHRF=1, BIAXF=1

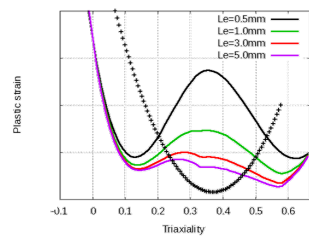


$Le=0.5\text{mm}$
 $Le=1.0\text{mm}$
 $Le=2.5\text{mm}$

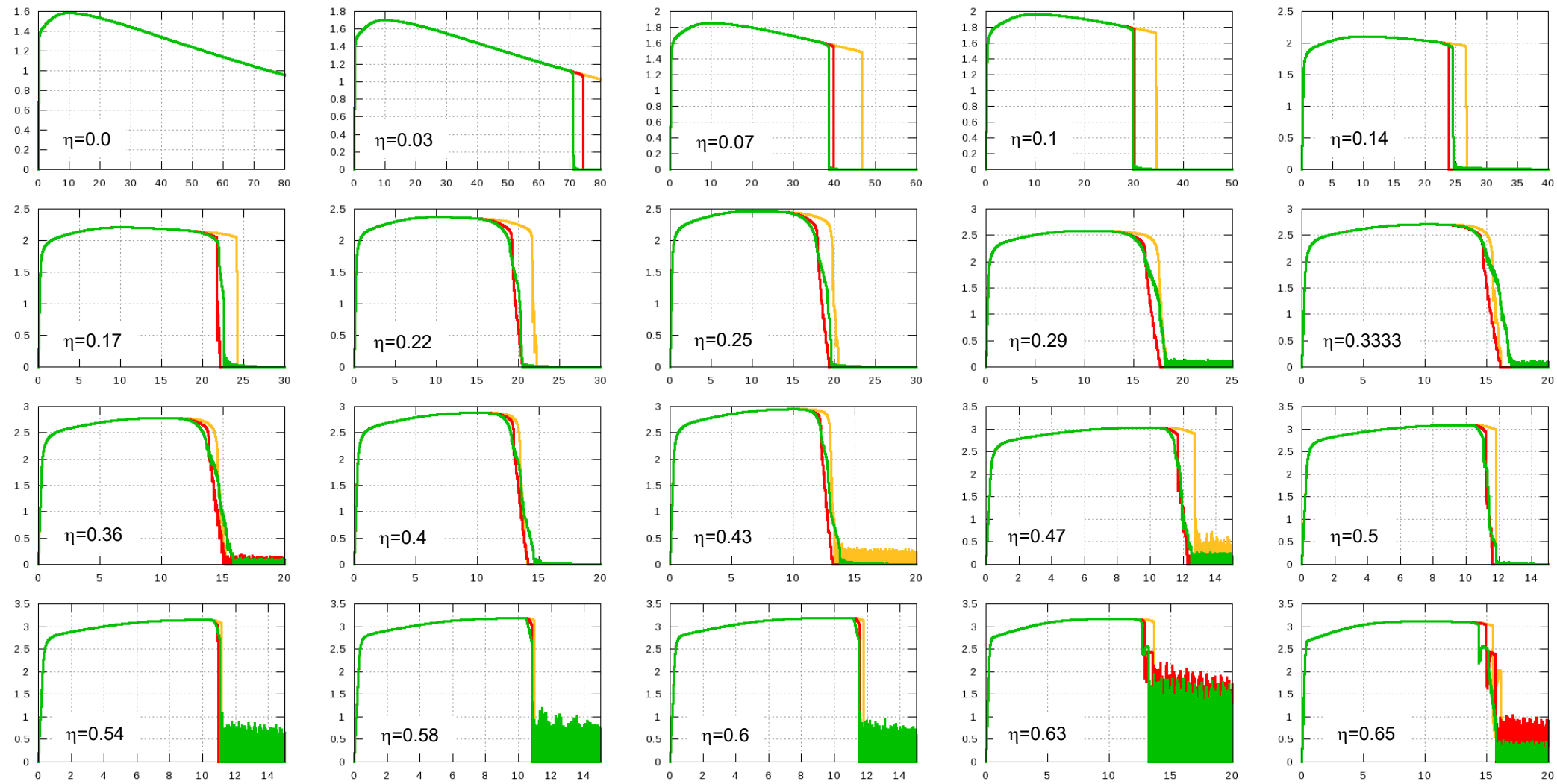


Y-force vs time

Table (1/3 eqv. 0.57735)



Le=0.5mm
Le=1.0mm
Le=2.5mm

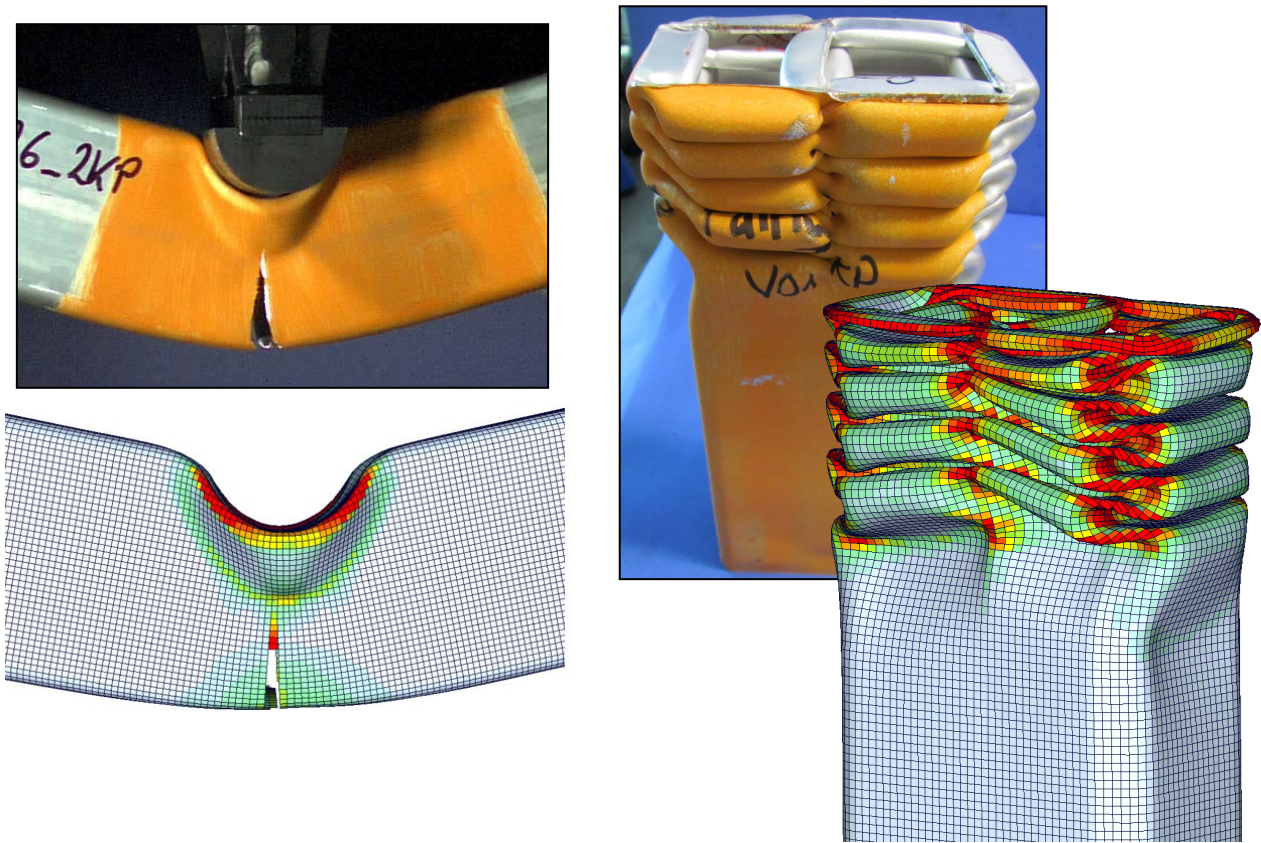


Old method: SHRF and BIAXF

SHRF/BIAXF brought significant improvement but often with contradictory results

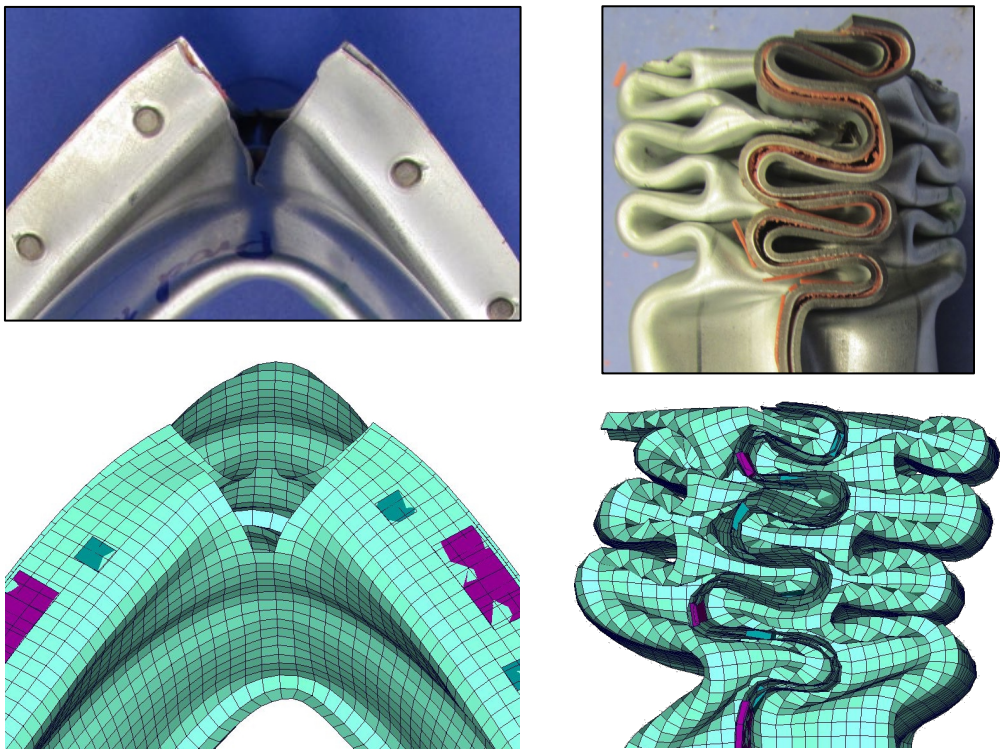
Aluminum extrusions

SHRF=1.0 BIAXF=0.0



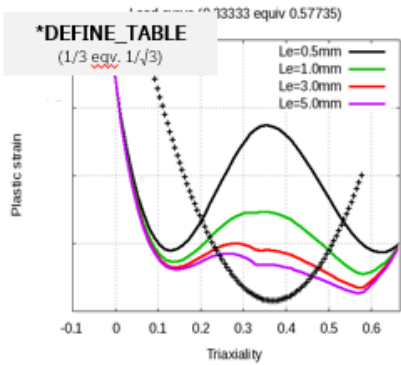
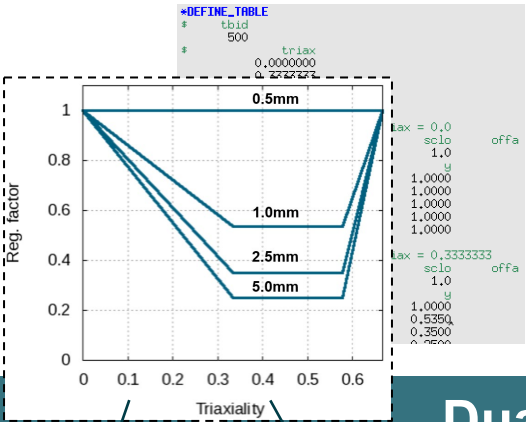
Dual- and complex-phase steel

SHRF=1.0 **BIAXF=1.0**



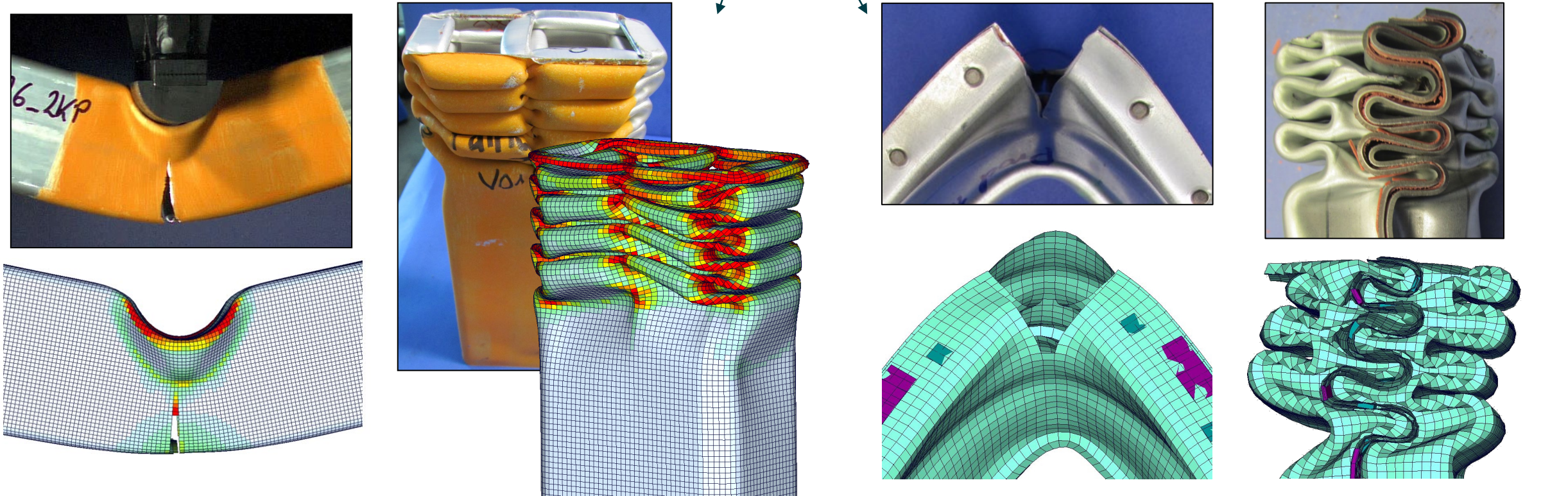
New method: Table

Implemented in R12,
RGTR1 and RGTR2 in dev versions



Aluminum extrusions

Dual- and complex-phase steel



Conclusions

- Spurious mesh dependence is stress state dependent
- Swift's plastic instability criterion predicts the onset of mesh dependence very accurately
- Regularization cannot help in geometrical mesh dependence, but can compensate for the effects of spurious mesh dependence
- Triaxiality-dependent regularization removes the ambiguity of using different values of SHRF and BIA XF for different materials
- It also brings significant benefits when mapping from forming to crash simulations
- A new feature is available in *MAT_ADD_DAMAGE_GISSMO through flags RGTR1 and RGTR2
- More investigation is necessary for stress states between triaxiality 0.0 and 1/3

Thank You

DYNAmore GmbH, an Ansys company
Industriestr. 2
70565 Stuttgart-Vaihingen
Germany

Tel.: +49 - (0)711 - 459 600 0
Fax: +49 - (0)711 - 459 600 29
info@dynamore.de

www.dynamore.de
www.dynaexamples.com
www.dynasupport.com
www.dynalook.com

© 2023 DYNAmore GmbH, an Ansys company. All rights reserved.
Reproduction, distribution, publication or display of the slides and content
without prior written permission of the DYNAmore GmbH is strictly prohibited.

Find us on



DYNAmore worldwide
Germany ■ France ■ Italy ■ Sweden ■ Switzerland ■ USA