# Recent Advances in Isogeometric Analysis

David Benson



## Isogeometric Analysis Development Teams

- LS-DYNA Solver
  - David Benson
  - Attila Nagy
  - Stefan Hartmann
  - Liping Li
  - Marco Pigazzini
  - Lam Nguyen
  - Gunther Blankenhorn

Most don't work on IGA 100% of the time, but the people listed have made significant contributions to IGA in LS-DYNA.

- Prime Mesh Generator
  - Harsh Vardhan
  - Jan Frykestig
  - Wenyan Wang
  - Mukul Kanitkar
  - Aditya Mukane
- ACE Team
  - Sourabh Chadha
  - Amit Nair
  - Lukas Leidinger



# Modeling Large Castings

Ansys

#### **Aluminum Castings for Large Structural Components**

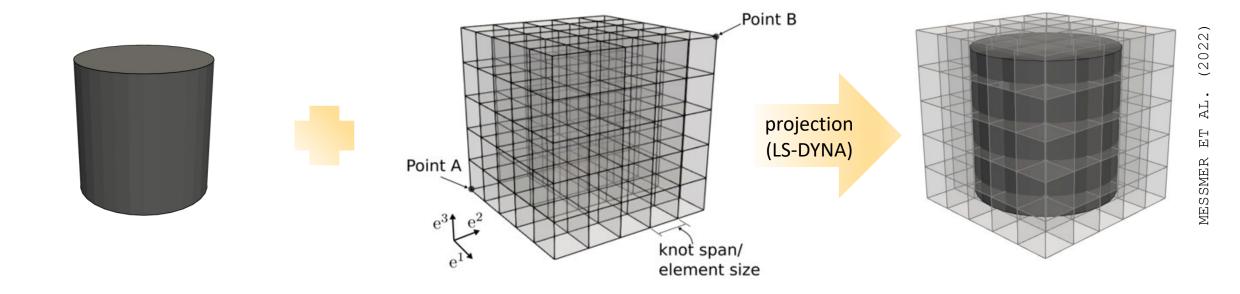


<u>GW Celebrates... the Tesla Giga Casting (gwcast.com)</u>

- Large castings eliminate hundreds of parts.
- Challenges include control of porosity and other defects that can reduce structural integrity.
- Solids are required to model complicated three-dimensional stress states accurately.
- FEA tet elements: small time step versus solution accuracy.
- FEA hex elements: difficult to mesh.



#### Trimmed Solids: The Basic Concept



Boundary representation (CAD) cleaned up during preprocessing

\*IGA\_3D\_NURBS\_XYZ
(quadratics are sufficient)

#### Isogeometric model

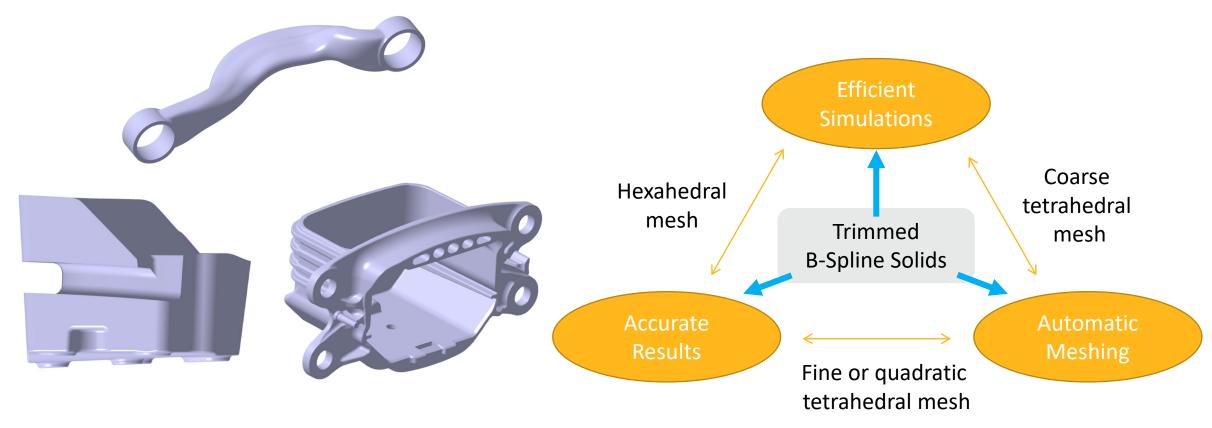
\*IGA\_2D\_BREP
\*IGA\_VOLUME\_XYZ
\*IGA\_SOLID
\*SECTION IGA SOLID



## Trimmed IGA Solids for Crash

#### Conflicting goals:

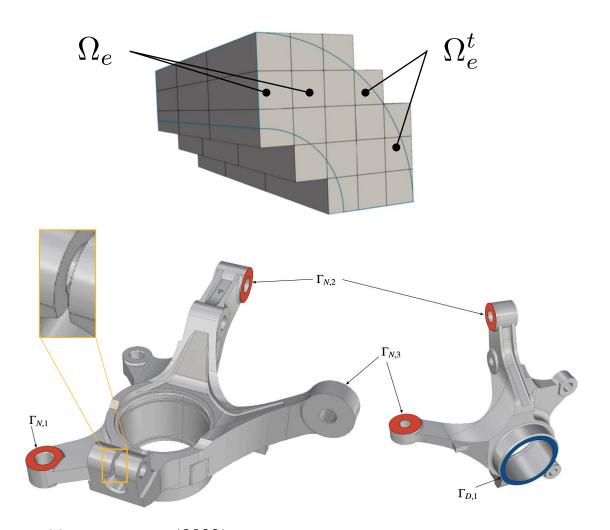
Each FEA option satisfies 2 out of 3.



Courtesy of BMW Group



## Highlights of Trimmed Solid-Based IGA



MESSMER ET AL. (2022)

- Simplified discretization (PDE)
- Element-wise specific quadrature rules for trimmed elements
  - Meshing (element quality is less important)
- Not boundary-fitted
- Feature preserving (BC)
- Stability (control points associated with trimmed basis)
- Local initial and geometry driven refinement may be necessary
- Accurate enough and very efficient



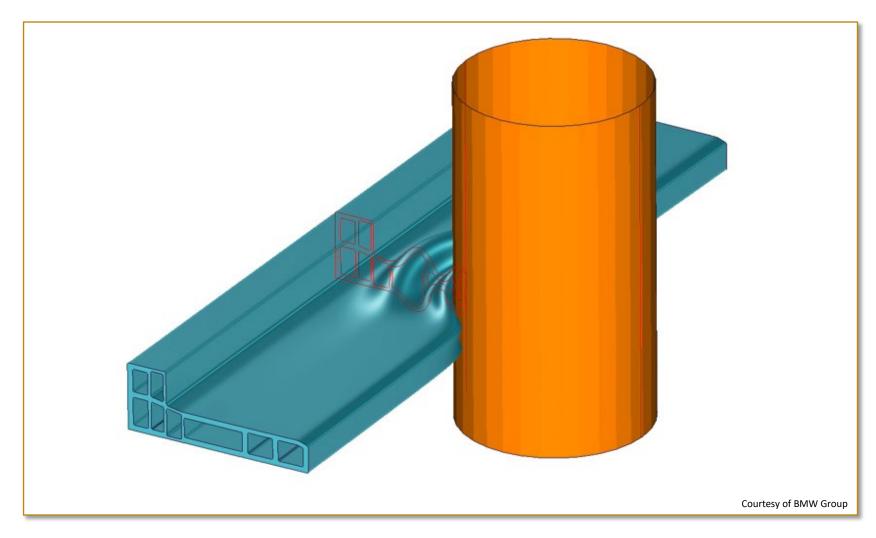
Extruded profile

length: 800mm

cross-section: 200x72mm

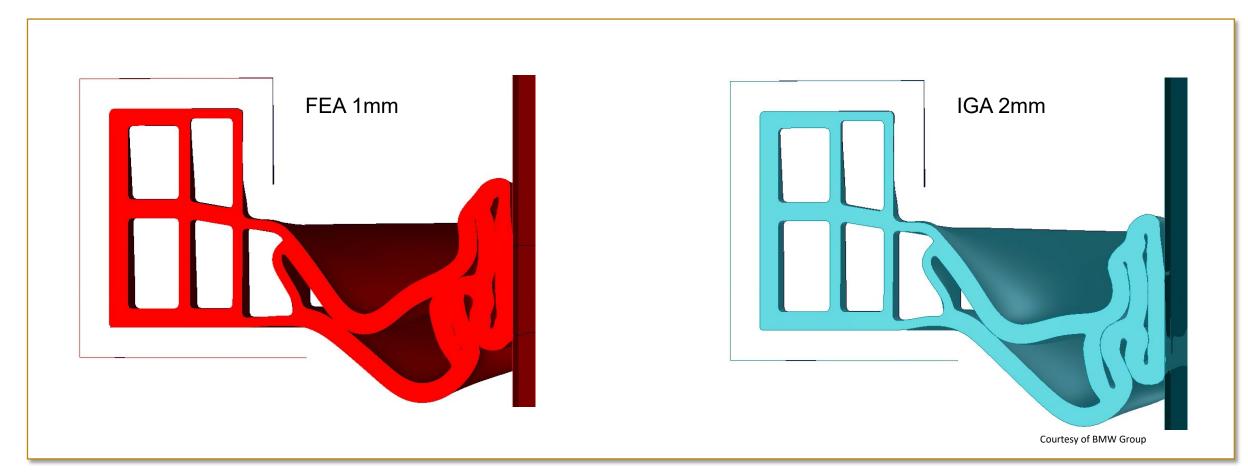
elasto-plastic material (without failure)

impactor: rigid cylinder





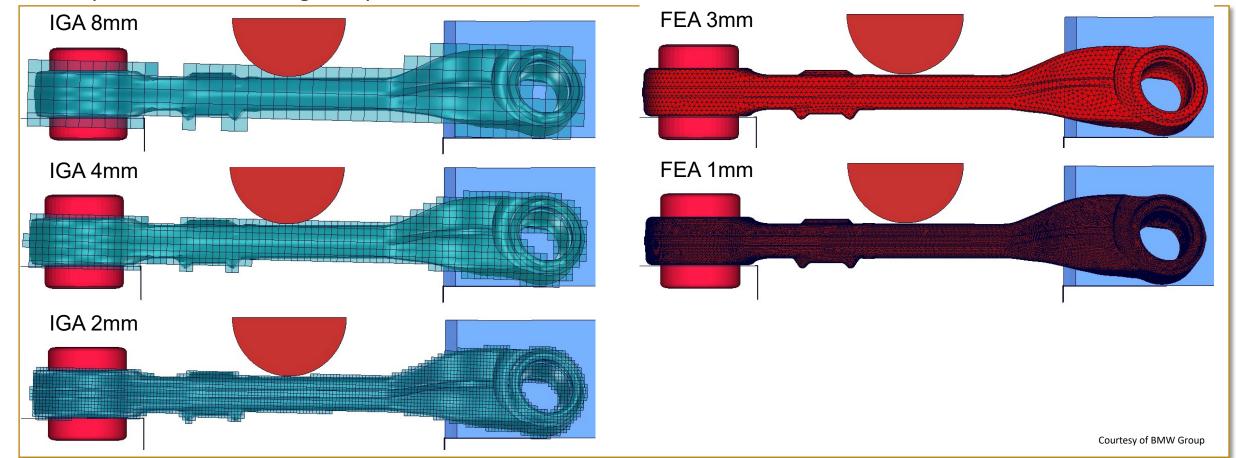
Local buckling pattern of the cross-section in the center of the profile





Wishbone (3-point-bending test)

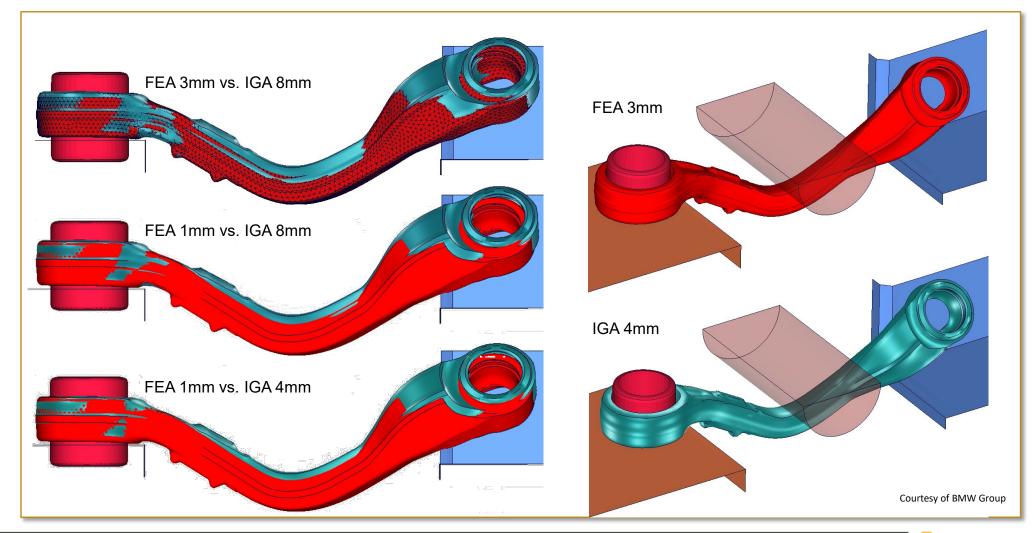
elasto-plastic material – rigid impactor – 5 discretizations





Wishbone (3-point-bending test)

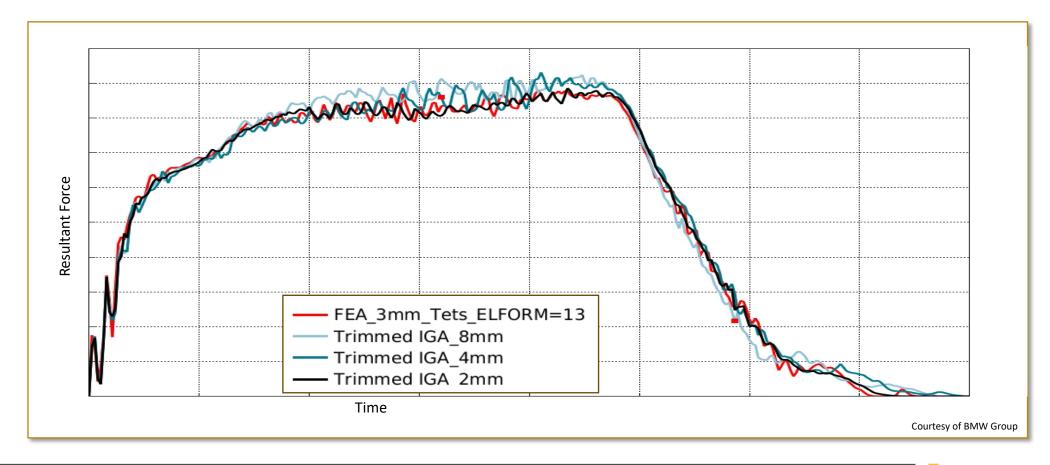
good agreement





Wishbone (3-point-bending test)

Resultant force vs. time – all very similar





Wishbone (3-point-bending test)

- some statistics:
  - larger time step / no mass scaling / faster computation

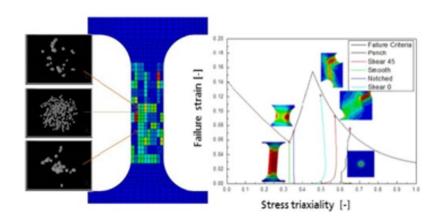
Model	time step (dt0, used)	added mass	added mass ratio	solved on	CPU time
FEA 3mm (ELFORM=13)	1.80E-07	1.0433E-05	2.05%	12 CPUs, local	54 min 19 seconds
FEA 1mm (ELFORM=13)	1.80E-07	2.3102E-03	455%	48 CPUs, cluster	2 hours 46 minutes 57 seconds
IGA 8mm	1.29E-06	0	0	12 CPUs, local	0 hour 6 minutes 12 seconds
IGA 4mm	6.47E-07	0	0	12 CPUs, local	0 hour 16 minutes 51 seconds
IGA 2mm	3.18E-07	0	0	48 CPUs, cluster	1 hour 15 minutes 40 seconds

Courtesy of BMW Group

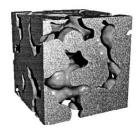


#### Material Model for Castings in Crash

- Spatial variation of properties with \*MAT\_TAILORED\_PROPERTIES (251)
  - J2 plasticity with history-dependent yield stress using TABLE\_3D/4D
  - New in R15: history dependent stiffness (Young's modulus), too
  - can be defined via \*INITIAL\_STRESS\_SHELL/SOLID
  - that "history" could be porosity from a preceding flow process



Courtesy of Tobias Erhart, Ansys Germany



Wodning 200.

| Section | Participation | Part

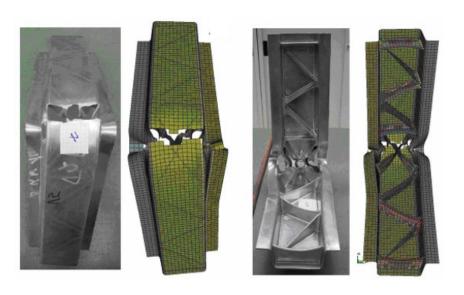
e.g. Hardin & Beckermann (2012): "Integrated design of castings: effect of porosity on mechanical performance"



### Damage/Failure for Castings in Crash

- Spatial variation in \*MAT\_ADD\_DAMAGE\_DIEM/GISSMO
  - Locally altered failure properties, e.g., porosity from manufacturing process
  - initial conditions as history variables on \*INTIAL\_STRESS\_SOLID/SHELL
  - alternatively, via node sets: new keyword \*INITIAL\_HISTORY\_NODE\_SET
  - or as stochastic distribution: \*MAT\_ADD\_DAMAGE\_GISSMO\_STOCHASTIC
  - Failure strain as function of stress state and history variable(s)
  - multi-dimensional functions needed: keywords\*DEFINE\_TABLE\_XD or \*DEFINE\_TABLE\_COMPACT

Courtesy of Tobias Erhart, Ansys Germany





#### Mapping Data to Meshes: Two Methods

• Both methods only work for initializing existing history variables. They do not, for example, allow spatially varying elastic modulus for arbitrary materials.

#### ENVYO

- Product for mapping originally from Dynamore, now an Ansys product.
- Generates a full deck restart file with mapped data.

#### \*PART\_FIELDS

- Define a point cloud in LS-DYNA input that is mapped to the mesh.
- Material data and the mesh are independent, allowing
  - Mesh refinement without remapping material data externally.
  - Changing the material processing (the point cloud data) without the extra step of generating a restart file.



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## IGA & FEA Time Step Size

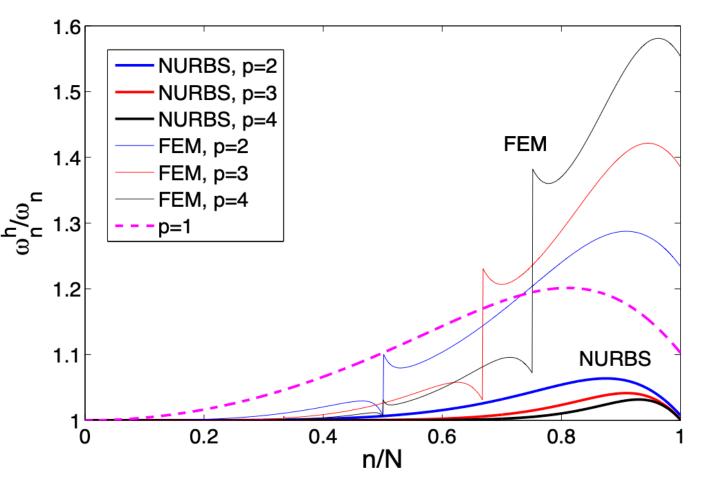
Ansys

#### Time Step Size and Stability: Basic Relationships

$$\triangle t \le 1/\Lambda_{max} = \ell/c$$

"ISOGEOMETRIC DISCRETIZATIONS IN STRUCTURAL DYNAMICS AND WAVE PROPAGATION"

J. Austin Cottrell, Thomas J.R. Hughes, Alessandro Reali and Giancarlo Sangalli ECCOMAS Crete, Greece, 13-16 June 2007



Why higher order FEA time step sizes decrease: error in the maximum eigenvalue.

Why higher order NURBS time step sizes don't decrease with degree.



#### Time Step Size and Stability: Useful Heuristics

- For logically regular trimmed and untrimmed IGA meshes, the time step is approximately the same as for linear FEA.
- For IGA meshes containing extraordinary points, the stable time step size depends on the method used to treat the extraordinary points.
  - No treatment (C0 continuity): no impact on the time step.
  - Special treatment to improve continuity:
    - Depends on the method, which depends on the mesh generator.
    - The time step size may be reduced.
    - Set ISDO=3 on \*CONTROL\_TIMESTEP and make a short test run to obtain the maximum possible time step size. ISDO=3 calculates the maximum eigenvalue by power iteration.



# Frequency Response Function Analysis with IGA

Slides courtesy of Yun Huang.



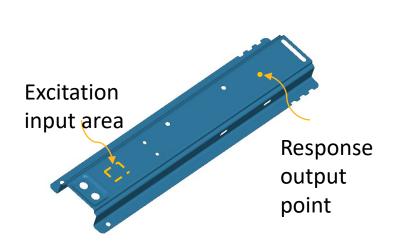
#### Towards One IGA Model for Crashworthiness and NVH

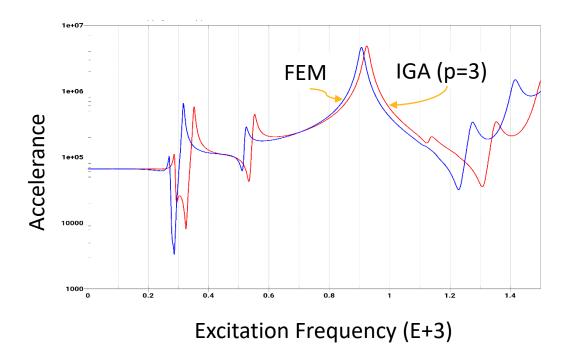
- Modal analysis is performed on the IGA not on the FE discretization.
- Compatible with concepts and features developed for the \*IGA keywords.
- Steady state dynamics and random vibration fatigue currently use the IGA interpolation finite elements but will be migrated to direct IGA analysis in the future like the modal analysis.

Isogeometric Eigenvalue and Modal Analysis



Frequency Domain Analysis Output

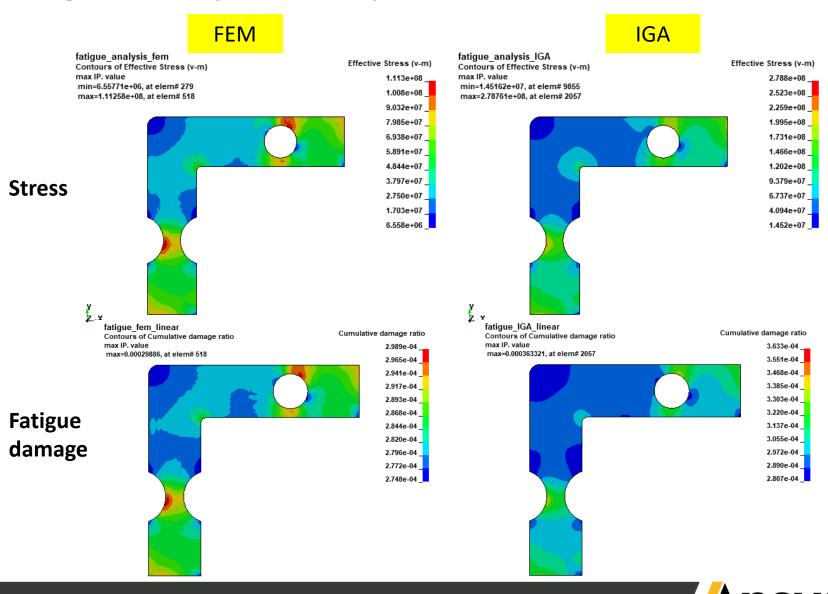






#### Random Vibration Fatigue Analysis (compare with FEM)

- Good match for both stresses and fatigue damage ratios.
- IGA provides more
   accurate stress values
   (stress concentration), thus
   more accurate fatigue
   damage.
- IGA takes less CPU time.



## **Commitment to New Technology**

- We have ongoing contact and collaborations with the leaders in IGA technology including:
  - Thomas J. R. Hughes, inventor of IGA.
  - Yuri Bazilevs, co-author on the first IGA paper.
  - Jessica Zhang, first post-doc of T. J. R. Hughes devoted to meshing.
  - Hugo Casquero, U. of Mich. Assistant Professor, meshing and element technology. Post-doc of Jessica Zhang.
- Our element implementations permit immediate use of any basis functions that can be expressed through Bezier extraction operators and NURBS.